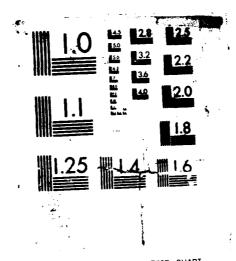
LIGHTWEIGHT TOWED HOWITZER DHMONSTRATOR PHASE 1 AND PARTIAL PHASE 2 VOLUM (U) FMC CORP MINNEAPOLIS MINN NORTHERN ORDNANCE DIV R RATHE ET AL APR 87 FMC-E-3841-VOL-A DARA21-86-C-8047 F/G 19/6 MD-A183 982 1/4 UNCLASSIFIED NL



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Lightweight Towed Howitzer Demonstrator

Final Report

Volume A

Overview



April 1987

Contract Number DAAA21-86-C-0047

FMC CORPORATION
Northern Ordnance Division
4800 East River Road
Minneapolis, Minnesota 55421

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20. ABSTRACT (Continue on reverse side if necessary and identify by block number)	
The LTHD (Lightweight Towed Howitzer Demonstrator) was to be a 9,000 lb equivalent to the M198, transportable via Blackhawk helicopter, with reduced emplacement time using fewer personnel. The FMC design achieved weight reduction via a mortar-like configuration, composites structure, and hydraulic actuators. Recovery of power from the recoil system, in turn, facilitated crew reduction via hydraulic emplacement, four-way joystick tube lay, and power ramming. FMC completed Concept Development (Ph I) and two-thirds of Detailed Design (Ph II) prior to funds running out.	

* CONTRACTOR SOCIOES

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	Actuator - Lanyard
	Cradle

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Abbreviations

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ARDEC CAD	Armament, Research, Development and Engineering Center computer-aided design
CEL	FMC Central Engineering Laboratories
83	center of gravity
DOF	degree-of-freedom
DTIC	Defense Technical Information Center
FEA	Finite Element Analysis
GFE	Government-Furnished Equipment
H!F	human factors
HMMWV	High-Mobility Multipurpose Wheeled Vehicle
LAPES	Low Altitude Parachute Extraction System
LTHD	Lightweight Towed Howitzer Demonstrator
NBC	nuclear-biological-chemical
0E	quadrant elevation
TDP	Technical Data Package

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The author apologizes if any names were inadvertently left out.

Don Sweitzer

FMC LTHD

Summary

Introduction	

A Blackhawk-mobile 155mm howitzer has been identified as a potential need within the US Army. The Lightweight Towed Howitzer Demonstrator (LTHD), providing M198 performance in a 9,000 lbm package, is a step toward determining the feasibility (and perhaps tradeoffs necessary) to meet this need.

This report documents the FMC effort toward this project. It is organized into nine volumes by the basic subjects covered.

Vol	Description
A	Overview
В	Technical Presentations
C	Dynamic Analysis
D1	Structural Analysis less Cradle and System
D2	Structural Analysis of Cradle
D3	Structural Analysis of System
E	Hydraulic Component Design by York
F	Systems Engrng Analysis, QA, Test Plans
G	Technical Data Package

Program Performance Status Summary -

Contract of the Contract of

The following summarizes the status of the design activities at the time the program funding was discontinued.

- 1. It was estimated that we were 65% complete with the Phase 2 Engineering effort. This estimate was based on our latest estimate to complete data which projected that the Phase 2 activities would not finish until week ending May 17.
- 2. Approximately 55% of the parts have been detailed [G, Tech Data Package].
- 3. The major area in which we were behind schedule and risks remained was the composite cradle. As is evident by the detail which is included in the analysis section, this part is the most complex and difficult and the design was still in a state of flux. We had identified a qualified subcontractor and had defined the terms and conditions for the contract.
- 4. The [unofficial] engineering estimate for Ph 3 cost is \$4.5 million.

Technical Overview -

The "power-to-weight ratio" necessary creates, besides the weight reduction, very significant firing and towing stability challenges. These challenges are further enhanced by the rapid emplacement/speedshift/displacement requirements.

A trade-study of 144 basic configurations [B/100pg3-13] resulted in the selection of a long recoil stroke, low trunnion, mortar-like configuration (with forward-pointing trails) for firing stability, simple (minimum weight) recoil system and load path, and its affinity to produce component shapes highly compatible with high strength-to-weight composites.

A serious problem with the mortar-like configuration came to be breech access and projectile ramming.

Due to the already present hydraulic system, it was decided to build upon this system by adding a hydraulic rammer and a recoil energy recovery system to compliment it.

Hydraulics were already employed

To minimize the weight of the equilibration-elevation-traverse subsystem (through the use of kevlar-wrapped hydraulic actuator technology developed for aircraft [by York Industries]), and

To extract the spade to achieve the displacement time requirement.

Extensive use of hydraulics, while providing design flexibility, threatened to complicate the controls and degrade reliability to an unacceptable point.

Fortunately, our teaming relationship with Marotta Scientific produced a clean system solution to the numerous potential trouble points. The controls are "user friendly", and reliability analysis suggests the LTHD will meet M198 reliability targets [F/130].

The anticipated weight savings of the mortar approach using high-low bearings (elimination of the 250 lbm turntable bearing plus heavily loaded understructure) was largely consumed by the energy recovery system addition.

The FMC LTHD, as a result of the breech access problem, broke even on what was originally felt to be a weight advantage over a more traditional configuration, and traded simplicity for energy recovery.

A second serious problem came to be our underestimation of the magnitude of complexity, interdependence (design-materials-analysis-fabrication), and resultant design and fabrication costs of composite components.

Two major structures (the gimbal and platform) were converted to metal (titanium) to shed some of this risk, leaving the cradle and trails.

Slow resolution of this problem (on the cradle and trails) not only added to the design interdependence problem, already being strained by the hydraulic subsystem, but also put the demonstration in jeopardy and delayed recognition of pending cost overruns.

These delivery and cost problems produced a situation that made it best from both the Army's and FMC's viewpoint that the effort be terminated.

Although we were unable to progress to a point of justified confidence, the mortar configuration is still felt to provide components more compatible with composites than a conventional howitzer.

A more detailed view of the project is provided by section 110 of this volume [A/110], the Design Descriptions and Considerations (in a structured format). This section, in turn, makes extensive reference to the balance of this report for more detailed information.

The following paragraphs in A/110 are of general interest:

Achievement vs Goal Summary, Further Weight Reduction, and Lessons Learned

Once again, this report is divided into the volumes listed below. These volumes, in general, include only the most current relevant data.

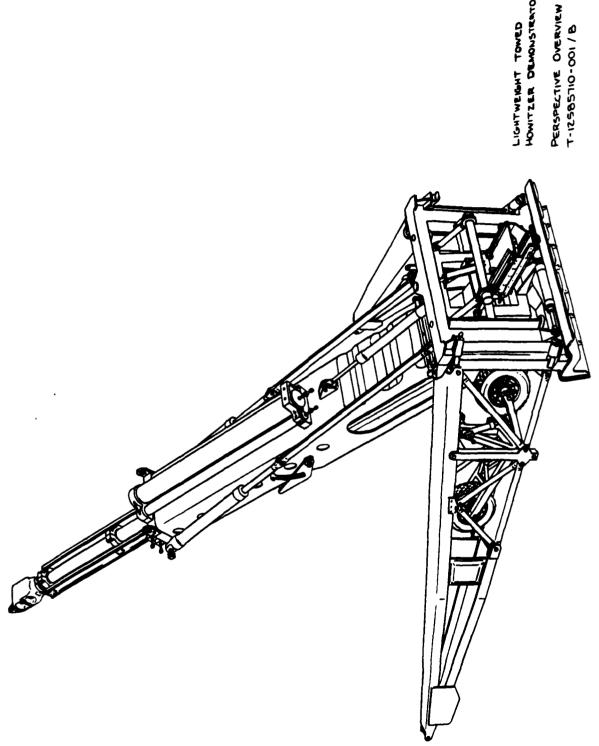
The design evolution records have not been included. The only exception to this is the cradle [D2] and the fixed orifice recoil [C/210].

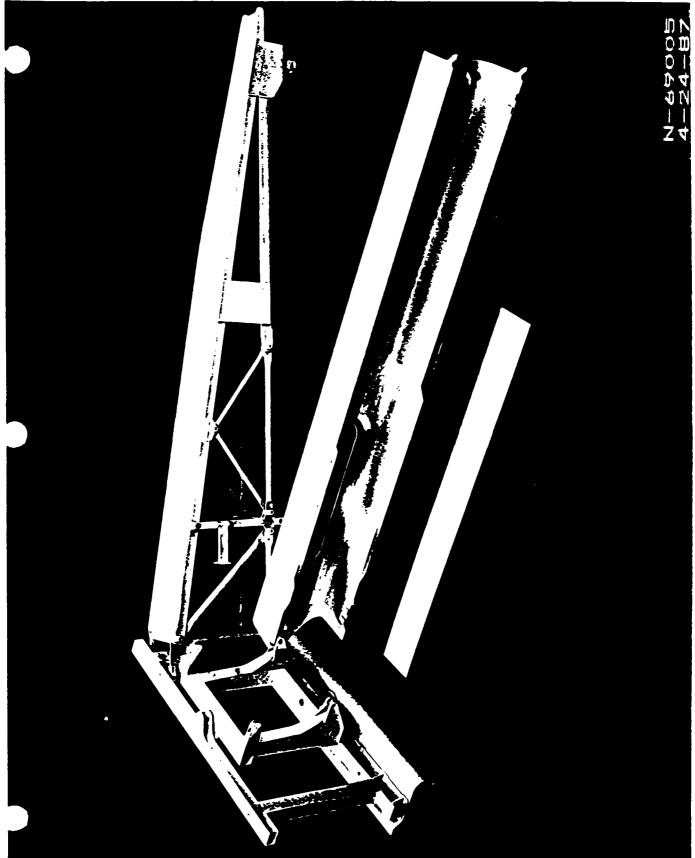
When design has evolved beyond the analysis (frequently due to results of the analysis), this is noted.

Vol	Description
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G	Technical Data Package

Technical questions should be directed to Bart Anderson, 612/337-3325 or 571-9201.

LIGHTWENSHT TOWED HOWITZER DEMONSTRATOR





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FIG. A/ 104-10 FMC LTHIN CARRIAGE (1/2 STALE MODEL)



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This alphabetical list of subjects facilitates a description of the FMC LTHD design as well as the considerations involved in the decision process used to arrive at each design position.

Accumulators Latch Position Achievement vs Goal Summary Lessons Learned Actuators Loading System Breech Mortar Configuration Breech Band Muzzle Brake Cannon Outer Breech Band Cannoneer 1 Manifold Platform Compound Actuator Assembly Primer Auto Loader Cradle Rails (see Cannon) Elevation/Equilibration/Traverse System Recoil System Energy Recovery System Spade Fire Control Speedshift Assembly Firing Plate Thermal Growth Control and Anchors Firing Stability Towing Stability Further Weight Reduction Trails Gimbal Valves Heat Removal and Sustained Firing Rate Walking Beams Hydraulic leaks

The following "paragraphs", one for each subject above, list the major considerations and features of the design chosen, within a hierarchial structure. Each subparagraph is preceded by a symbol to denote the relative advantage or disadvantage to the system provided by that item.

Symbol []	Meaning
+	Advantage to system
_	Disadvantage to system
*	Neutral to system
/	Considered but not used
>	Potential weight savings

Only "considered but not used" items that are considered currently pertinent are included.

Within each paragraph are also references to other sections within this report.

References enclosed by () refer to other paragraphs within this section; for example, (see Accumulators).

References enclosed by [] refer to other sections (possibly in other volumes); for example, [B/100pgl4] refers to Volume B, Section 100, page 14.

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Accumulators

- + Kevlar-wrapping (except reservoir accumulator); Kevlar is a TM of DuPont
 - + Reduces weight
 - + Increases resistance to explosion from small arms fire
 - + Basic design is already certified for flight and in use

- + Short stroke indicators
 - + Provide adequate volume indication in minimum space for minimum weight
 - + Basic design is already certified for flight and in use

Achievement vs Goal Summary [B/600pgl2thru35, for goals]

- (* indicates acceptable performance anticipated)
- Weight reduction
 - Probable weight = 9,200 lbm (see Further Weight Reduction)
 - * System configuration (see Mortar Configuration)
 - * Structural loading [D1/160, Load Conditions]
 - * Component configuration [G/050, Index to TDP]
 - * Material selection
 - * Structural Analysis [D1,D2,D3; Structural Analysis]
 - LTHD design is weak in documented composites producibility
 - * Firing stability [C/140]
 - * Towing stability [C/230]
 - + Probably better than M198
 - * Downrange accuracy
- * M198 envelope
 - * Cl30, LAPES, helicopter, 5 ton truck
- * Operational procedures [A/140]
 - * Emplacement/Speedshift/Displacement in 3 minutes with crew of 4
 - * Firing
- * M198 Performance
 - * Rate of fire
 - * Maximum
 - M198 is 4 rounds per minute, LTHD is 3.2
 - * Sustained
 - * Heat capacity of lightweight tube could be a problem
 - + Unique tube mounting provides additional heat conduction, storage, and convection (see cannon)
 - * Analysis of heat removal with free air needed, but problem probably resolvable
 - * Range
 - * AZ/QE limit
- * Cost if fielded
 - * Production
 - Hydraulic rammer and energy recovery system increase cost
 - * Quality
 - Honeycomb in cradle and trails may present QA problem
 - + Human factors
 - + Lunette load, spade & firing plate installation/removal, breech operation, ramming, and elevation/traverse reduce load on cannoneers below that of M198.
 - * RAM-D [F/130]
 - * Soft degradation
 - LTHD degradation may not be as soft as M198
 - + Safety [B/400pg7-2] [F/120]
 - + Probability of hand-in-breech, getting-hit-by-recoil, blastoverpressure-damage, load-hitting-helicopter, and falling-under-wheels reduced relative to M198.

Actuators

+ Kevlar-wrapping (when actuator is large enough to justify)

- + Reduces weight
- + Basic design is already certified for flight

- + Locked position is unaffected by hydraulic failure
 - + BearLoc (TM of York) provides normally on lock at any piston position
 - * Used on elevation and traverse system
 - * End cap/sleeve is interference-fit to rod, must pressurize to release
 - + Pinlocks provide lock with hydraulic failure (used in landing gear)
 - * Used on walking beam actuators
- + Self-lubricating bearings employed extensively

Breech

- * M185 style breech
- * Standard spring pack is retained (loading is at 600 mils max)
- * Hydraulically opened and closed [C/100]
 - + Eliminates one of M198's safety hazards [B/400pg7-2]
 - * Cannoneer cannot reach breech while operating control valve
 - + Facilitates compliance with human factors requirements
 - + Lighter than titanium breech cam
 - * Pilot-operated check with manual override facilities
 - * manual opening
 - * protection from closing due to a hydraulic hose failure
 - * manual closing (by pushing manual override button)
- * Primer Auto Loader mounted on door to facilitate mechanized primer handling
 - * Cannoneer insertion of primer from side is probably not reliable method
 - + Keeps Cannoneer from having to stand directly behind breech
 - * Hydraulic interlock (see Cannon) only allows primer insertion at battery *Normal load position is 3 feet out of battery

Breech Band [D1/150] [D1/190]

- * A two-piece design is used to facilitate breech removal without turning the tube.
- * The inner breech band is steel.
- * The outer breech band is aluminum. See Outer Breech Band.
- Whether or not it's necessary to keep the inner and outer breech band centerlines coincident over a broad temperature range and if so, how, has not been worked out.

Cannon

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- * Cannon recoil path is guided by rails attached to sides of tube
- * AlSiC rails are attached to tube via five collars
- + Rails, by being part of recoiling mass instead of stationary mass;
 - + increase the recoiling mass
 - + increase the heat convection area (from the tube)
 - complicate tube replacement
- * Collars are spaced to take spin-up torque from battery or load position
- * Titanium collar (nearest breech) serves as thermal anchor
 - * Provides strength required to accelerate rail assembly during recoil
- * Other collars are AlSiC
 - * lower strength requirement facilitates lighter material
 - * AlSiC CTE match to tube is adequate
 - * maximum heat transfer to rails
 - * collar-key geometry maintains tube to rail centerline despite CTE delta

* Rail and collar assembly is interchangeable from tube to tube

Cannoneer 1 Manifold

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- * Mounted on inside of cradle with control handles protruding thru cradle wall
- * Provides all loading controls and energy level gauge (see Loading System)
- * Control valve handles must be depressed before position can be changed

Compound Actuator Assembly

- * Most of major hydraulic components are mounted between the two "end caps", the front-cradle manifold and the midcradle manifold
 - * two recoil and two counterrecoil actuators set length
 - * Cylinders made of AlSiCp
 - * thermal growth closer to steel (than aluminum)
 - * good bearing properties

/ kevlar-wrapped steel considered by dropped due to thermal insulation

- * other end-to-end components are fixed on one end and floated on the other
 - * two counterrecoil accumulators
 - * load position actuator (sets and holds tube at "latch" position, see Latch Position)
 - * ramming actuator (pulls load tray forward, see Loading System)
 - * tube bundle to transfer fluid control signals between manifolds
- * Most "automatic" control valves are mounted within two "end caps"
 - * ram control (see Loading System)
 - * rail position valves "feel" flats in rail, identifying load and battery positions
 - * checks, pressure control, flow control, etc.
- * All components can be removed without pulling compound actuator assembly
- * Way bearings for rails mount within "end caps"
 - + easily replaced without pulling tube
 - + provides maximum wheelbase for recoiling mass (about 102")
 - * explosively bonded bronze to aluminum provides minimum weight bearing + Self-lubricating fabric bearing from Torrington also looks feasible
- * Alignment of tube centerline with AZ-QE centerlines over full temperature range facilitated via compound actuator mounting to cradle (see Thermal
- Anchors and Thermal Growth Control).

 * Projectile spin-up torque taken from rails and reacted into cradle via mid-cradle manifold [D1/170]
- * Recoil force taken from recoil rods and reacted into cradle end via front-cradle manifold
- * Heat rejection from recoil
 - * into the oil will be transferred into large-convection-area aluminumwalled reservoir accumulator mounted on top of cradle
 - * into recoil cylinder walls will have to be removed from air space within cradle via vent holes in cradle rood (see Cradle)
 - * is somewhat insulated from counterrecoil nitrogen precharge by insulated accumulator walls
- Both the front and midcradle manifolds are expected to have trouble meeting their weight targets
 - * FEA analysis on them has just started
 - * the forged aluminum is a long lead item
 - * if strength is a major problem, titanium plate would solve that problem, but the weight problem would probably intensify

Cradle [D2]

* Carbon fiber epoxy with nomex honeycomb core

- + should provide better dimensional stability than traditional steel cradle
- * Front edge has to be thickened to about 0.5" to handle recoil loads
- * Discontinuity at midbottom where spin-up torque is taken out is too complex * should be reconfigured (although Morton Thiokol feels its ok as is)
 - * minization of discontinuity would simplify multiple sourcing
- * Should add holes in roof forward of midcradle manifold to vent trapped hot air (primarily from barrel)
 - * strains forward of midcradle manifold are very low
- * Torrington fabric bearing rings may be best method to provide galvanic insulation and transfer load into trunnion
- * Morton Thiokol is interested in making a subscale cradle under IRAD
- * Design is probably conservative
 - * Design torque is too high
- * Titanium shims, as a means to increase joint carrying capacity, failed

Elevation/Equilibration/Traverse System [C/110]

- * Elevation actuator located to maximize resolution at low QE's [B/700pg19]
 - + results in a single actuator, which minimizes weight
 - * one actuator is less likely to fail than two, but redundancy is gone
 - produced reduced resolution at very high QE's
- * Equilibration actuators are as forward as possible
 - + Improves firing stability
 - + Improves equilibration match
 - + Simplifies load path into cradle
 - + Facilitates same load path as that used to tie in trails
 - Necessitates "equilibration links" [D/120]
 - * composite strut with self-aligning bearing in each end
 - * slip joint in equilibration actuator to accommodate any "lift"
 - / original kevlar cables (tm of dupont) were abandoned because
 - pulley size and weight exceeded space and weight available
 - cable twist could rotate equilibration rod end, possibly causing interference with cradle wall
 - minor concern over cleaning chemicals from bundle of cable strands
 - / Slip ring tie-in to front-cradle manifold, proposed for durability reasons, was dropped for cost and lead time reasons
- * Equilibration system adjusted for temp with accumulator volume adjustment
 - * Uses central accumulator detached from equilibration actuators
 - * one accumulator is less likely to fail than two, but redundancy is gone
 - Accumulator location may allow gas to pick up heat from reservoir
 - + Elimination of oil reservoirs and their maintenance to avoid dry seals (as on M198)
 - + Facilitates bellows accumulator and its advantages [B/400pg4-24]
 - + Would eliminate need for precharge maintenance
 - + Would provide faster gas temp stabilization after elev or depression [B/400pg5-27]
 - / Bellows accumulator not used for LTHD due to long lead time, tooling costs, and poor last-minute design-change flexibility
 - + Pressure adjusted from stored energy via pressure intensifier
 - + Amount of equilibration required can be read on pressure gage
 - + Elimination of need to mechanically move pivot point
 - + Reducing time to adjust to a minimum (probably in 3 min empl time)
- + Equilibration pivot point optimized [C/110]
- + QE firing lock is BearLoc on equilibration actuator (see Actuators)
- * Equilibration is turned on after emplacement, off during displacement

- * Distance of tube from AZ/QE axis necessitated concern over aiming accuracy (see Fire Control)
- / Slip rings from Torrington and/or Shamban were considered but not used
 - + Increased fluid system stiffness
 - Increased weight
 - * Analysis and experience suggested they weren't needed
 - * They could be added later

Energy Recovery System

- * Tapped from counterrecoil portion of recoil system (see Recoil System)
- * Is controlled to constant pressure (3,000 psi) by a pressure regulator
- + Can be supplemented by an external hydraulic power supply
- + Energy available is displayed at cannoneer 1 manifold (see Loading System)
- + May facilitate reduction in crew
- If insufficient energy is available, human effort required is slightly more than totally mechanical system
- Energy recovered from firing zone 4 and below may not be enough to cover energy requirements Equilibration Link
- * Firing a charge with no projectile may facilitate recharging energy level

Fire Control [D/130]

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- * M198 fire control is mounted on secondary trunnion
 - Special direct fire scope is required to see over roll bar (see Platform) [B/700pg69-70] [F/140]
 - + damping can be added to attenuate firing shock seen by fire control
- * Secondary trunnion follows primary trunnion via a parallel linkage
 - Additional sources of error are introduced [B/600pg106] [D/130pg4]
 - * Parallel linkage is carbon fiber epoxy
 - + maximum dimensional stability over temperature range
 - * Linkage is adjustable
- * Fire control is high relative to cradle
 - + Panoramic telescope has 6400 mil field of view

Firing Plate (see Spade)

Firing Stability [C/140]

- * Firing stability (hop and skid) was achieved by employing the concepts in order of increasing risk and weight (lowest risk and lightest concepts applied first)
 - * Lower risk lighter weight concepts employed
 - * Low trunnion
 - * Long recoil
 - * Shifting weight forward and to the right (for projectile spin-up)
 - * Hydraulic system forward
 - * Equilibration system forward
 - * Brake system in right trail
 - * Increased spade area
 - * Rod pull profile that overloads system early (and underloads later) relative to maintaining constant overturning moment [B/700pg15]
 - * Rigid trails to minimize storage of strain energy (with subsequent release during counter recoil)

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* Concepts applicable to mortar configuration not employed / High beta/blast-overpressure muzzle brake (see Muzzle Brake) * Viewed as beyond logical scope of LTHD / Mild version of soft recoil (see Recoil System) * Increase in risk not warranted / Secondary recoil * Added weight without facilitating any weight reduction Further Weight Reduction (a possible strategy) * Correct overdesign in projectile spin-up torque * Design projectile spin-up torque was 42,000 ft-1b (PIMP) * Should be closer to 27,000 ft-1b with PIMP as a minimum life req'mt * Build "prototype" with conservative safety factors and test to determine realistic design loads (will probably weigh about 9,200 lbm) * Update loading conditions from test data * Evaluate alternative materials and cut margins * Composites * Fatigue test composite coupons representative of structure * Trim composite structure per coupon fatigue test results > Save about 150 1bm * Titanium * Evaluate titanium/carbon-fiber-epoxy hydrid * Rough optimize using FEA * Build and strain test weldment (maybe not titanium) * Optimize wall and weld sections for life requirements > Save about 150 1bm * Hydraulic components Evaluate use of titanium as rod and liner material * Determine availability of aluminum-lithium * In 6 inch plate for front and midcradle manifolds * In smaller sizes for actuator end caps, cannoneer 1 manifold, etc * Evaluate practicality of replacing or supplementing Energy Recovery System with APU > Save about 100 1bm * Fire control * Evaluate possibility of electronic fire control * Basic Issue Items * Determine practical minimum weight collection * Determine practical design weight with M198 range > Probably about 8,800 1bm * Determine weight-range tradeoff * Reduce range with light barrel option to 15km * Reduce equilibration by changing accumulator * Leave balance of structure alone to retain 30km version * Build prototype and test in two versions * 9,000 1bm with 30km range (RAP) > Start with 8,800 1bm LTHD > Add 200 lbm for shields and thicker skins as required to increase durability * 7,700 1bm with 15km range (nonRAP, zone 7 max) * Lighter tube > Save 1,000 1bm * Different equilibration accumulator

* Reduced assortment of basic issue items
> Save 100 lbm

Gimbal [D/140]

- * Use of titanium (low CTE) minimizes thermal growth delta versus cradle
- * Large rectangular shape with tapered box cross-section provides
 - + Upper traverse bearing, permitting elimination of heavy turntable bearing
 - + Good strength-to-weight structure to handle projectile spin-up torque
 - + Potential candidate for weight reduction through use of carbon-fiber-epoxy
- * Provides mounting for load tray track and fire control secondary trunnion

Heat Removal and Sustained Firing Rate

- * Thermal analysis should be performed to see if heat buildup in cradle is ok
 - * Cradle probably should have holes in roof (see Cradle)
 - * Most of hydraulic heat will probably materialize outside of cradle (see Compound Actuator Assembly)

Hydraulic leaks

- * Design efforts to minimize internally-induced leaks
 - * Closed system has to be charged to be filled, eliminating dirt ingestion during oil replenishment
 - * Control valves are poppet-style (zero-leakage, contaminant-resistant)
- * Design efforts to minimize externally-induced leaks
 - * Manifolds used to greatest extent possible
 - * Valves and accumulators are subplate or cartridge mounted
 - * No pipe thread (except for two pressure gauges)
 - * Lead time problem, better sealing thread would be used in production
 - * Lines are protected
 - * Tube bundles with (o'ring) push connections used when possible
 - * Front-cradle manifold to midcradle manifold
 - * Midcradle to Cannoneer 1 manifold
 - * Single lines are routed in out-of-the-way areas
 - Hoses must be routed from Cannoneer 1 manifold to gimbal (across trunnion)
 - This joint is not yet configured and thus could cause problems
- * Design efforts to minimize damage due to leaks
 - * Power on valve is just downstream of energy storage
 - * Shutting off valve seals oil in storage
 - * Ports to pressure gages are normally closed (no leak with broken gage)
 - * Lines to trails are fuzed (flow will stop if a line breaks)
 - * Equilibration system will not allow tube to fall even due to total loss of hydraulic fluid (see Actuators [BearLocs]).
 - * Breech can be opened and closed manually (see Breech)
 - * Projectile can be loaded manually (see Loading System)
 - * Primer Auto Loader can be operated manually (see Primer Auto Loader)
 - * Walking beam actuators, if fully extended, will not release load even with a total loss of hydraulic fluid (see Actuators [pin locks]).

Latch Position

- * Load Position Actuator (LPA) pushes against outer band with reservoir pressure at battery
 - * Larger counterrecoil force holds cannon at battery

- * When cannon recoils, LPA follows outer band to LPA's stroke limit (3 feet)

 * Large flow passage from res'r accumulator allows high tracking velocity
- * With high zones, LPA brings c'recoiling cannon to rest with cushion
- * LPA has four cushions for recoil/c'recoil and load/battery conditions * Cushions are easily modified
- * With low zones, Cannoneer 1 controls allow LPA to drive cannon out to load pos'n (36" from battery)

Lessons Learned

- * Good informal communications with Army personnel is crucial to fully understanding the potential technical and management problems associated with the design and end-use of Army materiel.
 - * Surfaces problems (via proper channels) quickly.
- * Composite components necessitate early involvement and good informal communication between experienced design, manufacturing, materials, and analytical personnel.
- * A mortar-like howitzer provides a lightweight and stable structure, but some of the weight savings will be lost to solving the breech access problem.
- * Energy recovery can be used to solve the mortar-like howitzer breech access problem
 - + Reduce load on crew and crew requirements
 - + Reduce some hazards
 - Increase hardware cost
 - Probable decrease in soft degradation.

Loading System

- * Loading is performed at these conditions to allow 95th percentile Cannoneer 1 to insert propellant into breech from side.
 - * 0 to 600 mils
 - * with cannon 36" out of battery
- * Elevation system is designed to facilitate rapid tube depression and elevation to maintain firing rate [C/110]
 - Maximum firing rate is roughly one round every 18.8 seconds (M198 spec is one round every 15 seconds)
- * All loading controls are located at Cannoneer 1's manifold
 - + Cannoneer 1's manifold cannot be reached from breech.
 - Cannoneer 1 cannot hold bag in Swiss Notch with his/her hand.
 - * Not important, since all loading is done at 600 mils or below.
 - + Cannoneer 1 cannot close breech on his/her hand.
 - * The breech area is a safety hazard on the M198.
- * Loading Controls are arranged in natural sequence and orientation.
 - * Check energy level valve
 - + Pressure gage with settable external "energy dial"
 - * allows Cannoneer 1 to set dial according to volume indicators on counterrecoil accumulators, thus providing a ready reference as to whether or not there is enough energy to complete ramming cycle.
 - * Open/close breech valve
 - * Can only be operated at load position (36" out of battery)
 - * Can be manually opened and closed (see Breech)
 - * Ram with automatic return, slow ram, and manual retract valve
 - + Ram with auto return keeps Cannoneer 1 from accidentally releasing ram lever too soon (resulting in an inadequate ram)

- * Slow ram and manual retract used to position load tray behind breech
 - * for advancing Copperheads to breech (for manual ram)
 - * to catch stuck projectile during extraction
- * Move cannon to battery pos'n valve (from load pos'n, 36" out of battery)
 - * Releases actuator holding cannon out of battery in load position
 - * Can also be used to move cannon back to latch position when firing zone 4 and below because cannon does not recoil far enough to reach latch position.
- * Primer extraction and insertion valve (operates primer auto loader)
 - * Can only be operated at battery position
- * Lanyard valve (fires primer)
 - * Can only be operated at battery position
 - * Can be fitted with long lanyard and operated remotely (if off "safety")
- * Ramming actuator pulls tray forward, returns load tray to position behind recoil path
 - * Shock absorbers on load tray decelerate tray when they impact breech face
 - * Projectile momentum carries it into forcing cone
 - * Variant of flick ramming
 - * Longer acceleration distance should reduce structural deflection at point of release, possibly reducing variance of extraction force
 - * Load Tray is U-shaped, hangs from overhead track
 - * Replaceable wear bars
 - * Ogive restraint limits forward travel of projectile when projectile is thrown into tray
 - * Ogive restraint lifts automatically when load tray hits breech face
 - * U-shaped leaf spring holds base during ram while allowing ease of loading.
 - * Load tray hangs from three-section track
 - / A flexure was considered but dropped due to:
 - * conflicting need for flexibility (to minimize force required to bend), and
 - * rigidity (load tray stability), as well as
 - * strain necessary to make operational beyond material limits
 - * Forward section is suspended from cradle roof
 - * Rear section slides in a mount to the cradle
 - * Midsection connects forward section to rear section by hinge joints
 - * Removal of this section also provides hoist access to pull breech
 - * Vertical turns in track due to elevation (at hinge joints) necessitate

 * Load tray hanging from mini-walking beam suspension (minimizing bump
 - from vertical turns).
 * Track thickness is varied to take up clearance during "final
 approach" to ram.
 - * Load tray velocity is controlled by rammer control
 - * Expected to maintain +-10% velocity over temperature extremes
 - * Adjustable low speed section to get load tray into forward section
 - * Low speed section "time out" is based on oil volume used
 - * Adjustable high speed section to reach ram velocity before breech
 - / Mechanism to correlate ram velocity with QE was configured (connected to equilibration actuator) but not used when decision was made to limit loading height to 600 mils
 - * Adjustable retract velocity
- * Manual loading
 - * Disconnect load tray from ramming actuator (allowing tray to roll freely)
 - * Depress tube to 150 mils

* Set projectile on tray, roll up to breech, and ram with long staff used to extract stickers

Mortar Configuration

- + Facilitates long recoil
- + Facilitates constant stroke recoil, simplifying recoil system
 - + Moves muzzle brake forward of crew positions relative to M198
 - + Reduces crew exposure to blast overpressure (6' or 2db)
 - Extends barrel farther from AZ/QE axis, possibly resulting in pointing accuracy problems
 - * Analysis suggests this problem is manageable
- + Provides ability to retract barrel for tow, providing compact package
 - + Improves road maneuverability in tow configuration
 - + Cradle protects crew from long recoil, a safety hazard [B/400pg7-2]
 - + Improves resistance to helicopter damage during helicopter lift, a current hazard with the M198 [B/400pg7-2]
- + Facilitates low trunnion height
 - + Minimizing jog in structure reacting horizontal recoil force into ground
 - Neg 5 deg QE is probably impractical
 - Requires secondary trunnion
 - * If electronic fire control is an option, the secondary trunnion may not be needed because the indirect fire control could be taken from transducers, while the direct fire scope could be mounted to the cradle.
- + Load path for high and low QE uses same structure
 - + Minimum structure, minimum weight
 - Wide platform/spade requires ground to conform to howitzer rather than traditional 3-point mount conforming to ground
- + Configuration can be resolved into large section tubular beams and/or trusses (platform, gimbal, cradle, trails)
 - + Maximum strength and stiffness to weight
 - + Compatible with composite structures
- + Forward trails
 - + Facilitates balanced weight distribution
 - + Minimum chance of rebound instability during counterrecoil
 - + Significantly shortens walking path between ammunition and loading points
 - + Pallets can be set up a few feet from load tray entrance without getting in way for speedshift (see Speedshift Assembly)
 - + Width near lunette reduces probability of rollover
 - Potential jackknifing problem when backing up
 - + May "catch" cannoneer that falls out truck, preventing him/her from going under wheels, a current M198 hazard [B/400pg7-2]
- Poor breech access
 - Necessitates providing a latch position for loading [B/500pg3]
 - Necessitates hydraulically powered rammer
 - Necessitates energy recovery system to power rammer (see Energy Recovery System)
 - Significantly complicates hydraulic system
 - + Recovered energy can be used to power other subsystems (see Energy Recovery System)
 - + Potential crew reduction

Muzzle Brake

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* M198 muzzle brake with integral lunette

- + Towing load path goes directly into barrel
- * Titanium casting
 - * New tooling required for lunette
 - * Weight reduction
 - * Improved corrosion resistance
- * Structural analysis
 - * Extensive structural analysis not warranted
 - * M198 brake limitation is wear, not strength
 - * Accuracy of structural analysis not sufficient to resolve difference as small as that between strength of titanium planned and steel used by M198
 - * If titanium brake fails, replace with M198 brake
 - * Make special "brake" just for towing
 - * CTE mismatch with tube could cause basic concept problem
 - * Analysis suggests CTE mismatch is not a problem [D/180]
- / High beta/blast-overpressure muzzle brake was considered but not used / Risk and magnitude of scope not worth gain (see Firing Stability)

Outer Breech Band

- * Transfers recoil force from recoil and counterrecoil rods to inner breech band.
- * Receives hydraulic signals from centers of recoil and counterrecoil rods and transfers them to the actuator that opens and closes the breech, the actuator that indexes the primer autoloader, and the actuator that fires the primer.
- * The outer breech band is made of aluminum for three reasons besides minimum weight and cost
 - * good CTE match with the midcradle manifold (avoid rod binding due to CTE mismatch).
 - * provides heat sink with respectable convection area to remove heat from the breech.
 - * good hydraulic manifold
- * The recoil and counterrecoil rods are connected to the outer breech band in such a manner to allow some float to eliminate the potential for binding as the rods are retracted to the battery position.

Platform

- * Use of titanium (low CTE) minimizes thermal growth delta versus gimbal
- * Large rectangular shape with boxed cross-section provides
 - + Upper traverse bearing, permitting elimination of heavy turntable bearing
 - + Good strength-to-weight structure to handle projectile spin-up torque
 - + For deep trail section (see Trails)
 - + Natural roll bar to protect against roll-over damage
 - + Potential candidate for weight reduction through use of carbon-fiberepoxy

Primer Auto Loader

- * Twenty round drum
- * Primer actuator
 - * extracts spent primer
 - * indexes drum
 - * inserts new primer

- * Lanyard actuator
 - * fires primer using standard firing mechanism
- * Manual operation
 - * Primer actuator motion can be overridden by a hand lever
 - * Lanyard can be attached to firing mechanism

Rails (see Cannon)

Recoil System (see also Compound Actuator Assembly)

- * Long stroke minimizes recoil force
 - * Stroke limit is set by barrel length
 - * 105" total stroke
 - * 6" free recoil delays major loading until after shot ejection
 - * 96" effective recoil
 - * 3" overtravel cushion
- + Traditional orifice rod configuration
 - + Orifice rod profiled to provide constant load if fired from latch also provides fairly optimum profile for minimum hop under normal (fire from battery) condition.
 - + Free recoil delays major structural loads until after shot ejection
 - / Single orifice system was dropped due to complex interaction with Energy Recovery System [C/210]
- * Physically part of Compound Actuator Ass'y (see also Compound Actuator Assembly)
 - * Facilitates integration of recoil system with other hydraulic functions
 - * Recoil and counterrecoil actuator rods are hollow to minimize weight and to provide fluid signals to outer breech band (see Outer Breech Band)
 - * Counterrecoil provides input to energy recovery system (see Energy Recovery System)
 - * One counterrecoil actuator and counterrecoil accumulator are traditional.
 - * Second counterrecoil actuator acts as a single stage hydraulic pump.
 - * Second counterrecoil accumulator acts as a receiver for the second actuator.
 - * Recoil actuators used to extend barrel for firing, retract barrel for towing, from control on front-cradle manifold [see Mortar Configuration]
 - + Same function can also be used to stroke barrel into staff (held by rope to front-cradle manifold) to hydraulically remove stickers.
- / Recoil force reduction through a mild form of soft recoil might be feasible / Fire from the latch position would facilitate

some reduction in muzzle brake beta (and blast overpressure) and/or some increase in charge

- / Latch position with the long stroke of the FMC LTHD provides
 - * 69" cook off buffer
 - * 6" hangfire buffer

forward trails minimize chance of nosing over

* 30" soft recoil stroke

Spade

- * Effective spade area must be increased over M198 to maintain same skidresistance due to weight reduction
- * Main spade is mounted beneath platform
 - + Simple load path
 - + Also serves as firing plate
 - + Easily replaced plastic inserts protect platform from rock damage
 - + Integration with platform eliminates loose piece
 - * Towing stability with walking beams facilitates raising CG enough to maintain adequate ground clearance without taking off spade
 - Three-point engagement with soil requires some compliance by soil
- * Main spade is titanium
 - + More ductile and higher strength than AlSiC
 - + Almost half the weight of steel
 - + Minimal corrosion problem
- * Perforations will probably be added
 - + Weight reduction
 - + Limit crack propagation
 - + Minimal impact on holding capability
- / Spades were considered [B/400pg4-25] for end of trails, but abandoned
 - + would aid in controlling skid
 - + ground engagement would help maintain lay during unavoidable rebound hop
 - potential lateral moment on trail is excessive

Speedshift Assembly

- * Mounts beneath cradle
- * Detach link, swivel down, attach link, speedshift
- * Reverse to stow speedshift
- LTHD does not speedshift about AZ, thus howitzer lay will be lost
- + Panoramic telescope has 6400 mil range, so howitzer can be relayed from neighboring howitzer (see Fire Control)
- + Ammunition, even though close to loading points, does not have to be moved to speedshift (see Mortar Configuration)

Thermal Growth Control and Anchors

- * Due to the mix of materials with different CTE's
 - * Thermal growth control is important to insure that excessive strain and stress doesn't damage structure during temperature excursions.
 - * Thermal anchors are critical where maintenance of critical dimensional features during temperature excursions is necessary.
- * Tube to collars and rails

PASSESSE SECTION

- * Collars are made of titanium where high strength is required and of AlSiC where strength requirement is less.
- * Hoop stresses induced by tube temperature changes and internal pressurization is controlled by
 - reliefs on the inside diameter of the collars.
- * Stresses along the length of the rails is minimized by using AlSiC for the rails (vs aluminum) and controlled by allowing the collars to slip along the tube.
- * Four keys in each collar anchor the collar to the tube to carry the projectile spin-up torque and maintain angular alignment.
- * The (titanium) collar nearest the breech serves as locates the rails along the length of the tube (a thermal anchor).

- * Rails to compound actuator assembly
 - * X-axis control and anchor (using an xyz coordinate system as viewed from the breech).
 - * x-axis control is achieved by making the right flat and thus allowing it to float.
 - * x-axis is anchored by the "V" on the lower surface of the left rail thus traverse error within the rail mounting system due to temperature excursions is negligible
 - * Y-axis control and anchor
 - * sufficient vertical differential growth is provided by 0.010" bearing clearance.
 - * gravity sets the anchor at the bottom surface of the rails until projectile spin-up occurs, at which time the right rail anchor shifts 0.010" upward as the bearing load is shifted from the bottom to the top. The tube centerline moves about 0.007" in a parallel fashion about the "V" of the lower left anchor.
 - * Long span between way bearings (102") further minimizes already negligible tube lay errors.
- * Compound actuator assembly to cradle (see Compound Actuator Assembly for thermal growth considerations within the assembly) [B/700pg3lthru33]
 - * Recoil force is taken out at front-cradle manifold
 - * Recoil actuator rods transfer load into manifold
 - * Manifold end loads cradle
 - * Vertical posts stick through cradle
 - * on centerline of cradle end edge to eliminate CTE z-axis mismatch
 - * Bottom of front-cradle manifold serves as x-y-z-axis anchor
 - * Bottom post is "headed" to tie compound actuator assy front to cradle
 - * Top post serves as x-axis anchor
 - * Projectile spin-up torque is taken out at midcradle manifold.
 - * Vertical posts stick through cradle to transfer torque.
 - * Bottom of midcradle manifold serves as x-y-axis anchor.
 - * Bottom post is "headed" to tie compound actuator assy rear to cradle.
 - * Top post serves as x-axis anchor.
 - * Post holes in cradle are slotted to facilitate z-axis growth
 - * Fit-as-assembly sequence facilitates alignment with AZ-QE axis
 - * Post removal facilitates compound actuator assembly removal without refitting
- * Trail assembly to cradle and platform (see Trail for thermal growth considerations within the assembly).
 - * Trail attachment to front-cradle manifold assembly accommodates z-axis growth.
 - * Trail attachment to platform serves as x-y-z-axis anchor.
 - * Torrington fabric thrust bearings locate y-axis
 - * Clearance in thrust bearings provides for thermal growth
- * Gimbal and platform
 - * both are titanium which, with its low CTE, are relatively stable dimensionally
 - * Torrington fabric thrust bearings locate y-axis
 - * Clearance in thrust bearings provides for thermal growth
- * Tube bundle to Cannoneer 1 manifold
 - * Push tubes (o'ringed slip joints) are used to accommodate the CTE mismatch between the cradle and the tube bundle (stainless steel)

Towing Stability

- * Preliminary analysis suggests stability of M198 will be exceeded (video tape provided to ARDEC 04Jun86) [C/230]
 - * Walking beam multifunction capability (see Walking Beams) provides
 - * Lunette load (100 lbf) compatible with human factors even at min crew
 - * Lunette loading compatible with rule-of-thumb towing stability requirements (10% of towed load)
- * Overall LTHD width had to be increased (though still narrower than M198) to maintain tread width necessary for stability when walking beams were moved into trails (to eliminate separate dolly)
- * Latest walking beam configuration has not been analyzed for stability, although we do not think stability has been degraded by
 - * adoption of unequal length walking beams (see Walking Beams)
 - * addition of damping via walking beam actuators (see Walking Beams)
- * Rear of cannon is supported during towing
 - * Cannon is retracted for towing (equivalent to full recoil)
 - * As cannon is retracted, outer breech band engages towing guide mounted to inside bottom of cradle
 - * This is not documented in TDP

Trails

- + Deep section provides
 - + high stiffness-to-weight ratio, necessary to minimize rebound factor in firing stability (see Firing Stability)
 - * large storage area
- * Open truss
 - * Primarily carbon fiber epoxy members with titanium joints
 - * Matches CTE of cradle (important in stow configuration)
 - * Plus a titanium wheel bulkhead
 - + Thermal growth within structure is manageable
 - * Tied into cradle during towing to minimize loading caused by "bump and skid" (going around corner at high lateral g's and hitting bump)
 - * tie-in must be folded away during emplacement to keep from getting in way of cannoneer 1
 - * keeps Cannoneer 1 from experiencing amplified blast overpressure (from reflection)
- * Preloaded in firing position by threaded connection from platform
 - * Thread may be a problem with dirt (although probably no worse than pins used on M198)
 - Can only be opened to one angle

Valves

- * All valves are poppet-style (zero-leakage, contaminant-resistant)
- * Control valve handles must be depressed before position can be changed
- * AZ/QE (4-way) joystick
 - * force feedback throttling
 - * provides "feel"
 - * provides automatic release and relock of BearLocs
 - * May enhance aiming accuracy and speed from "video game skills" of current generation of soldiers

Walking Beams

- + Reduce road shock input
 - + Combination shock absorbers/hydraulic actuators dampen road shock
 - + Simple control valve mounted on each actuator (one per wheel)
 - + Fuzed to main circuit to keep a leak in walking beam actuators from bleeding down main system
 - Problem from high flow rates in actuator possible from road shock input have not yet been resolved
 - / Accumulator in actuator rod, possibly with pilot-operated two-way valve in piston, limiting oil consumption during emplacement while opening volume to accumulator during tow (requires change in actuator control valve porting pattern)
 - + Facilitate raising CG sufficiently to make spade integral to platform
 - + Spade can be removed if more ground clearance than M198 is req'd
- + Moved from separable dolly to integral with trail [B/500pg9] [F/140]
 - + Dolly was hard to align and maneuver
 - + Eliminate loose piece
 - + Dolly would not have been compatible with matured design of cradle
- + Provide multiple additional functions
 - + Meet human factors specs for lunette load with minimum crew while putting sufficient weight on lunette during towing for stability (see Towing Stability)
 - + When locked together (forward and rearward part of walking beam linked)
 - + Rock forward to reduce load on lunette to 100 lbf
 - + Rock backward to increase angle of departure
 - + Rock flat tire up to change tire
 - + Lock in position to run on good tire when other is flat
 - + When not locked together
 - + Lift into trail during firing, increasing stabilizing moment
 - + Lower rear wheels to extract spade
 - + Lower front wheels to balance howitzer during speedshift
- + Provide ability to drop 763 lbm in an emergency
 - + Break or cut fluid connections and release pivots
- * Unequal length walking beams adopted
 - + Rear wheel set to maximize angle of departure
 - + Front wheel set to provide manageable lunette load for emplacement/displacement
 - + Design facilitates easy modification to adjust actual lumette weight after assembly
 - + Space between wheels set to make sure rocks were unlikely to bind up between them
 - + Walking beam pivot set back to maximize size of rock front wheel can go over before binding.
 - + Improves C130 roof clearance [B/200pgHT-3andHT-5] [B/400pg4-7]
- * Service brakes on all four wheels
 - * Two sets per wheel to achieve envelope requirement necessary to retract into trail
 - + Improved redundancy

COMPRESSIONAL BOOKS OF THE BOOK

- * Parking brakes on forward wheels only (used with light lunette load)
- * Axle is pulled (like front wheel of motorcycle) to change wheel or tire
- * Operation of actuators from a pressurized reservoir facilitates lowering wheels during displacement in below freezing weather (a problem with the gravity-fed reservoir on M198)

PHASE II
FORMALLY NOTED
OPENING DEPICIENCIES

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PRELIMINARY

EVALUATION TEAM COMMENTS

FOR FMC

PARCE SUPPLY INFO?

KARLY NORTH

FMC System Operability

Strong Points

- .o Configuration presents a compact package which greatly enhances transportability.
- o Theoretically excellent cross-country capibility. (reference
- o System will, theoretically, significantly reduce the labor intensity associalted with operating towed artillery.

Weak Points-

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o The system appears to be 100% hydraulically controlled and operated. SG-1 Minor hydraulic problems and or combat damage could easily cause this weapon to become totally inoperable.

- 5r-2 o Hydraulic system reliability does not have a good track record in existing ARMY artillery systems.
- o Assuming stored energy is insufficient to emplace the weapon, the time required to manually pump up the system is viewed as an operational 50-3 deficiency.

o The secondary or offset trunnion, appeares to be vertically displaced by approximately 4 ft. The ability to effectively engage direct 50-4 fire targets is questionable due to parallax conditions between the bore and direct fire sight centerlines.

o The configuration necessitates the bore centerline to be approxiametely 18 inches from gound level. The ability to effectively engage 50-5 direct fire targets is questionable at this height due to trees, brush, rocks etc., between the howitzer and target.

- o The firing crew is totally separated from the breech area. Visual 506 inspection to verify complete charge bag insertion is impossible. This is viewed as a safety hazard and an operational deficiency.
- o Visual inspection of the bore area and verification of bore clear 50procedures are extremely difficult; if not impossible.
- o During blackout operations, no visual verification of the loading SU-8 procedure is possible.
- LOAD TRAY o During daylight hours no visual verification of Analy engagement with the breech, correct ramming of the projectile, and correct position of 50-9 the powder charge for ramming is possible.

- o Power ramming a powder charge is viewed as a high risk operation. 50-10 The charge is not rigid, and ripping or tearing the bag is possible if charge alignment is off. In addition, no visual verification by the crew is possible if the charge "hangs up".
- Mechanical ramming by pushing on the powder charge igniter pad is 50-11 viewed as high risk.

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- o Bore swabbing and breech clearing through water spray and brush is 50-12 totally unacceptable.
- 50-13 o No discussion of system disengagement with prime mover is discussed.
- 50-14 o Manually "jockying" around the dolly is viewed as difficult on uneven terrian, considering the weight of this item.

FMC Design Adaquacy and Produceability

- o The effect the metal liner has on the composite structure in the recoil cylinders must be adressed. Thermal coefficients of expansion must YEST TO RONIDE LUPO! be considered in order to preclude de-laminations.
 - o Several problems exist in the slide concept configuration. The thin wall honeycomb sandwich construction is both impact and moisture sensitive. Reference to acceptable aircraft construction does not indicate adequacy for . WILL BES DE WAY? MYBE ETTHORD howitzers.
 - o Postruded rails are very risky. The likelyhood of premature failure due to matrix cracking at the corners is high. This is caused by inability of the fibers to reinforce the corners. The concept of making four plates and joining them with preformed corners is inadequate. O Filament winding in this area would produce a superior structure. Galvanic/corrosion and adhesive selections should be addressed.
 - o Honeycomb core construction in the trails is a concern due to impact damage and moisture absorbtion. Pre-formed woven corners seem inadequate. Complexity and cost would be high with this concept. The choice of a factor of safety approach is not as adequate as hot/wet property data. As MGD STATISTICAL TIME FIRETHING.
 - o The choice of graphite epoxy skin with aluminum honeycomb core in the platform area invites two potential problems. One is the moisture effects on honeycomb construction and two is the shielding of aluminum from graphite to prevent galvanic corrosion. Hand lay-up and autoclave cure is reasonable for these parts.

TAPE WAS TO SIMULATE FILAMENT WINDING?

o The "claws" are constructed as described above and similar problems are forseen. The claw should be designed for damage tolerance since it is expected to interface with the ground. Graphite epoxy skin is inadequate for this purpose.

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- o The 'backbone" is constructed using a hand lay-up procedure rather than being filament wound. This is considered a poor approach for this component. Removing the mandrel to form a concave section may result in a wrinkled, void filled structure that will not possess the desired mechanical properties.
- o All components should be inspectable by ultrasonic techniques. No indication of inspecting for moisture absorbtion exists. This must be addressed.
- o The bolted joint method of integration is acceptable; however, corrosion and sealing of the holes from moisture should be addressed.
- o Bulkiness in trail design is also a concern. It is not clear what weight optimization process ruled out a more reasonable selection of dimensions. The construction does not seem to take damage tolerance and envionmental degradation factors into account.

FMC Structural Verification

- o Beam stress calculations on the supporting trails which allow 70kpsi in a quasi-isotropic lay up of graphite epoxy seem suspect. The allowed stress can not possibly be greater than 50kpsi for a quasi-isotropic lay up, even before introducing moisture effect degredation factors.
- o There is also a disregard for environmental effect, including hot/wet conditions and impact dammage degredation effects.
- o The claimed skin thickness in the trail beam of .019" in the sandwich flanges is suspect.
- o The produced rail design is suspect. The corners are vulnerable to cracking when the structure is loaded in bending.
- o Strength calculations are questionable because of the dry room temperature values used. The knockdown factors are generated from the Tsai-Wu equation; however, the stresses calculated have sketchy rationale. The documentation does not indicate how these stresses are derived.

o No justification has been given for the size of the trails. Collaboration on the design methodology of these components is desired.

FMC System Stability

Strong Points

o Excellent towing stability analysis through computer simulation.

Weak Points

- o A trapezoidal recoil force profile is most desirable but not always obtainable. Maintaining this profile through 8 and one half feet of recoil travel will be difficult, especially in light of the "energy taps" employed for the hydraulic system. Temperature induced gas pressure changes, as they affect recoil force, appear not to be adequately addressed. A "sensitivity"analysis would be helpful.
 - o Counter-recoil stability is mentioned but not fully investigated.
 - o Rifling torque appears not to be included in stability computations.

FMC Cannon / Ammo

Strong Points

- o It appears cannon system is compatible with all 155MM amountion.
- o No adverse impact is anticipated on range or on range precision with the configuration.

Weak Points

Processor Representation

- o Cannon interface was not adequately determined until after the phase I final review.
- o The automatic breech opening mechanism for this design has not been determined.
 - o The primer feed mechanism has not been adequately defined.

FMC - Recoil Mechanism

- o The effects of the external cylinders used for power recovery and storage on the recoil cycle has not been addressed.
- o The manufacturing of an effective ten foot recoil mechanism is of great concern.

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o A trapezoidal shape recoil force vs. time curve is utilized for all analyses. A more conservative approach in which some room is allowed for non-ideal circumstances is preferred.

FMC - Organization

- o There appears to be a reluctance by FMC management to bring forth supporting documentation of their efforts. This position hampers not only the FMC effort, but the program as well by not allowing adequate monitoring of progress made. A change in this area will be most mutually beneficial and desirable.
- o There seems to be a lack of coordination between CEL and Northern Ordnance. It is felt that the FMC composite experience is very limited perhaps to that associated with the Bradley Fighting Vehicle. The primary emphasis and design philosophy brought forth in the Light Weight Towed Howitzer is that of metal construction. If this philosophy and outlook does not change, grave design errors are likely to occur.
- o It is unclear why this effort has both a program manager and project manager. Discussion of this question with Ms. Tia Stackland resulted in a poor explanation.
- o The entire cannon interface portion of this program was poorly executed.
- o Program has undergone a shakeup in management midway through Phase 1. This is apparant in the organization of this entire program. Numerous significant design changes up to the conclusion of Phase 1 indicate that personnel are not clear on their direction.
- o Technical personnel seem well motivated however, numerous design changes apparant at the final review indicate they also are unclear of their direction.

FMC - Quality Assurance

The inspection facilities that FMC has shown are predominantly laboratory testing apparatus. They are able to test coupons and tensile samples, but it is felt that large, full size, howitzer structures are going to pose a problem.

The honeycomb composite structure used throughout the concept has a moisture absorption characteristic that results in decreased mechanical performance. Because the proposed outside skin is an organic composite it will be very difficult to inspect for moisture content.

FMC has given little consideration to producibility aspects - the hand lay-up techniques proposed are very expensive.

The loader feeder for primers is a complicated unproven device - this can be a major RAM problem.

The cantilevering of the gun may cause fire control problems because of vibrational modes and/or instabilities.

The skin of the so called "claws" is proposed to be graphite epoxy. It is hard to foresee this contruction surviving the type of abuse it will receive.

The equilibration cables seem to be a protential RAM problem.

The P.A. plan is scarcely addressed.

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It appears that little consideration has been given to the formulation of an impact damage assessment.

There is no indication of any thought given to the delamination problem which may occur from expansion between the over wrapped cylinders and the composite wrapping.

The recoil slides/cradle is viewed as having three serious potential problems:

- 1. Degradation by moisture absorption.
- 2. The method utilized for joining the four sides appears very similar to ones used for metal fabrication. This structure would seem to be much more efficient if filament wound.
- 3. In the chosen configuration, the glass or graphite pultrusions that are to make up the slides seem very prone to develop cracks due to a lack of reinforcement inherent in the pultrusion process. In addition, residual stresses in the material due to resin shrinkage may be enough to initiate cracks.

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CONTRACT CASCAGO

Date:

4 August 1986

To:

Distribution (See responsibilities below)

From:

Herb Theumer TV

Subject:

Evaluation Team Comment Replies

Attached is a coded copy of the Evaluation Team Comments received at 5 the kick-off meeting for Phase II at ARDEC.

As discussed earlier, we have ample oportunity to reply to these comments by either explaining discrepancies or misunderstandings, or by detailing another approach that may be more appropriate to use than the one originally proposed.

We will share the reply effort by assigning responsibility for various sections to different team members. That does not meam that everyone will help as needed.

The responsibilities are as follows:

System Operability (SO)

. Design Adequacy and Produceability (DAP) Structural Verification (SV)

System Stability (SS)

Cannon/Ammo (CA)

Recoil Mechanism (RM)

Organization (0)

Quality Assurance (QA)

Herb Theumer/ Bart Anderson

Bruce Zierwick Bruce Zierwick

Scott Dacko/ Jeff Ireland Bart Anderson Jeff Ireland/

Scott Dacko
Dave Peterson/
Herb Theumer

Herb Theumer

We will state each comment and follow it with our reply in order to have adequate cross-reference to each section and each comment using the coding shown on the following pages.

The schedule for this effort is as follows:

15 August draft comments to Herb Theumer for compilation and

team review

29 August copies sent to ARDEC for review prior to meeting

5 September Meeting to discuss replies at ARDEC

The first project review meeting for Phase II will be tentatively held at ARDEC on 29 and/or 30 October.

This would require an internal review meeting and dry run about 16 October. Let's plan to make these dates.

Preliminary Evaluation Team Comments for FMC

FMC System Operability

- SU-1 The system appears to be 100% hydraulically controlled and operated. Minor hydraulic problems and/or combat damage could easily cause this weapon to become totally inoperable.
- SO-2 Hydraulic system reliability does not have a good track record in existing ARMY artillery systems.
- SO-3 Assuming stored energy is insufficient to emplace the weapon, the time required to manually pump up the system is viewed as an operational deficiency.
- The secondary or offset trunnion appears to be vertically displaced by approximately 4 ft. The ability to effectively engage direct fire targets is questionable due to parallax conditions between the bore and direct fire sight centerlines.
- The configuration necessitates the bore centerline to be approximately 18 inches from ground level. The ability to effectively engage direct fire targets is questionable due to trees, brush, rocks, etc. between the howitzer and target.
- 50-6 The firing crew is totally separated from the breech area. Visual inspection to verify complete charge bag insertion is impossible. This is viewed as a safety hazard and an operationasl deficiency.
- SO-7 Visual inspection of the bore area and verification of bore clear procedures are extremely difficult if not impossible.
- SO-8 During blackout operations, no visual verification of the loading procedure is possible.
- 50-9 During daylight hours no visual verification of dolly (loading tray) engagement with the breech, correct ramming of the projectile, and correct position of the powder charge is possible.
- SO-10 Power ramming a powder charge is viewed as a high risk operation. The charge is not rigid, and ripping or tearing the bag is possible if charge alignment is off. In addition, no visual verification by the crew is possible if the charge "hangs up".

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- SU-11 Mechanical ramming by pushing on the powder charge igniter pad is viewed as high risk.
- 50-12 Bore swabbing and breech clearing through water spray and brush is totally unacceptable.
- SD-13 No discussion of system disengagement with prime mover is discussed.
- SO-14 Manually "jockying" around the dolly is viewed as dificult on uneven terrain, considering the weight of this item.

FMC Design Adequacy and Produceability

- DAP-1 The effect the metal liner has on the composite structure in the recoil cylinders must be addressed. Thermal coefficients of expansion must be considered in order to preclude de-lamination.
- DAP-2 Several problems exist in the slide concept configuration. The thin wall honeycomb sandwich construction is both impact and moisture sensitive. Reference to acceptable aircraft construction does not indicate adequacy for howitzers.
- Poltruded rails are very risky. The likelyhood of premature failure due to matrix cracking at the corners is high. This is caused by inability of the fibers to reinforce the corners. The concept of making four plates and joining them with preformed corners is inadequate. Filament winding in this area would produce a superior structure. Galvanic corosion and adhesive selections should be addressed.
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- The choice of graphite epoxy skin with aluminum honeycomb core in the platform area invites two potential problems. One is the moisture effects on honeycomb construction and two is the shielding of aluminum from graphite to prevent galvanic corrosion. Hand lay-up and autoclave cure is reasonable for these parts.

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FMC Structural Verification

- SV-1 Beam stress calculations on the supporting trails which allow 70 kpsi in a quasi-isotropic lay-up of graphite expoxy seem suspect. The allowed stress cannot possibly be greater than 50 kpsi for a quasi-isotropic lay-up, even before introducing moisture effect degradation factors.
- SV-2 There is also a disregard for environmental effect, including hot/wet conditions and impact damage degradation effects.
- SV-3 The claimed skin thickness in the trail beam of .019" in the sandwich flange is suspect.
- SV-4 The produced rail design is suspect. The corners are vulnerable to cracking when the structure is loaded in bending.
- SV-5 The graphite epoxy skin of the claws appears to be vulnerable. This area should be addressed utilizing a damage tolerant design criteria.

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SV-6 Strength calculations are questionable because of the dry room temperature values used. The knockdown factors are generated from the Tsai-Wu equation; however, the stresses calculated have sketchy rationale. The documentation does not indicate how these stresses are derived.

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FMC System Stability

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- SS-3 Rifling torque appears not to be included in stability computations.

FMC Cannon/Amro

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- CA-2 The automatic breech opening mechanism for this design has not been determined.
- CA-3 The primer feed mechanism has not been adequately defined.

FMC Recoil Mechanism

- RM-1 The effects of the external cylinders used for power recovery and storage on the recoil cycle has not been addressed.
- RM-2 The manufacturing of an effective ten foot recoil mechanism is of great concern.
- RM-3 A trapezoidal shape recoil force vs. time curve is utilized for all analyses. A more conservative approach in which some room is allowed for non-ideal circumstances is preferred.

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- D-3 It is unclear why this effort has both a program manager and project manager. Discussion of this question with Ms. Tia Stackland resulted in a poor explanation.
- D-4 The entire cannon interface portion of this program was poorly executed.
- D-5 Program has undergone a shakeup in management midway through Phase I. This is apparent in the organization of this entire program. Numerous significant design changes up to the conclusion of Phase I indicate that personnel are not clear on their direction.
- O-6 Technical personnel seem well motivated however, numerous design changes apparent at the final review indicate they also are unclear of their direction.

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- QA-2 The honeycomb composite structure used throughout the concept has a moisture absorptiob characteristic that results in decreased mechanical performance. Because the proposed outside skin is an organic composite it will be very difficult to inspect for moisture content.

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- QA-3 FMC has given little consideration to producibility aspects the hand lay-up techniques proposed are very expensive.
- QA-4 The loader feeder for primers is a complicated unproven device this can be a major RAM problem.
- QA-5 The cantilevering of the gun may cause fire control problems because of vibrational modes and/or instabilities.
- QA-6 The skin of the so called "claws" is proposed to be graphite epoxy. It is hard to foresee this construction surviving the type of abuse it will receive.
- QA-7 The equilibration cables seem to be a potential RAM problem.
- QA-8 The P.A. plan is scarcely addressed.

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PARTIES SESSION CONTRACTOR DESCRIPTION

- QA-9 It appears that little consideration has been given to the formulation of an impact damage assessment.
- QA-10 There is no indication of any thought given to the delamination problem which may occur from expansion between over wrapped cylinders and the composite wrapping.
- QA-11 The recoil slides/cradle is viewed as having three serious potential problems:
 - 1. Degradation by moisture absorption.
 - 2. The method utilized for joining the four sides appears very similar to ones used for metal fabrication. This structure would seem to be much more efficient if filament wound.
 - 3. In the chosen configuration, the glass or graphite pultrusions that are to make up the slides seem very prone to develop cracks due to a lack of reinforcement inherent in the pultrusion process. In addition, residual stresses in the material due to resin shrinkage may be enough to indicate cracks.

PHASE III

FORMAL RESPONSE

TO

OPENING DEFICIENCIES

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PICATINNY (860829)

ACCOUNT PRODUCTS

Russ, C

F M C

REPLIES

to

PRELIMINARY

EVALUATION TEAM COMMENTS

FMC Corporation Columbia Heights Center (CHC) 3989 Central Avenue Minneapolis, Minnesota 55421

29 August 1986

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Design Adequacy and Producibility	16
Structural Verification	28
System Stability	32
Cannon/Ammo	35
Recoil Mechanism	37
Organization	39
Quality Assurance	45

Appendix

FMC System Operability

SO-1 <u>Comment</u> - The system appears to be 100% hydraulically controlled and operated. Minor hydraulic problems and/or combat damage could easily cause this weapon to become totally inoperable.

Reply - The system is essentially 100% hydraulically controlled and operated. The following steps have been taken, to date, to minimize the susceptibility to minor problems and/or combat damage that might render this weapon inoperable. The major reason for selecting hydraulic control is that it provides a design that is significantly lighter than other alternatives investigated.

- 1. All major pressure vessels are enclosed within the slide.
 - Improves protection.
- 2. All major actuators (except recoil and counterrecoil) are kevlar-wrapped. Recoil and counterrecoil are not currently kevlar-wrapped due to heat rejection concerns.
- Improves resistance to external damage resulting in internal malfunction.
- 3. All accumulators are kevlar-wrapped.
 - Improves resistance to externally-induced internal

damage and reduces or eliminates shrapnel due to bullet-induced explosion.

- 4. Equilibration actuators have composite shields.
 - Improves protection.
- 5. Connection of equilibration accumulator to equilibrator actuator is through a slip ring into a manifold.
- Improves structural integrity and resistance to shock loading.
- 6. Equilibration accumulator is of bellows type.
- Improves resistance to external damage resulting in internal malfunction.
- 7. All valving is manifold mounted (cartridge or subplate).
- Improves structural integrity and resistance to shock loading.
- 8. Major arteries between manifolds are bundled and kevlar-wrapped (front of slide to rear of slide and rear of slide to mid-slide).
- Improves structural integrity and resistance to shock loading.
- 9. Pipe threads are not used.
- Improves structural integrity and resistance to shock loading.
- 10. Hydraulic circuit breakers protect the recoil and equilibration systems from fluid loss due to loss of fluid in other sections of the hydraulic system.
- 11. Dual hand-pumps are provided (one for gunner, one for assistant gunner).

- Provides redundancy.

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- 12. "Feather touch" controls are planned in critical control areas.
- Improves resistance to hydraulic system-induced shock loading.
- 13. Ultra-fine, zero bypass filter (high delta psi element).
 - Does not allow contaminated fluid to enter system.
- 14. Dynamic analysis of hydraulic system is planned for Phase II.
- Reduces probability of "water-hammer" and resultant hydraulic system-induced shock loading.

As specifics relative to RAM-D and degraded modes develop, additional measures may be taken. They include:

- 1. Select special hose configurations and materials to maximize durability.
- 2. Provide ability to isolate selected sections of hydraulic system and provide two-20' hoses to facilitate external hydraulic circuitry in the event of artery damage.
- 3. Employment of redundant systems for critical functions (redundancy levels in jet aircraft typically vary from 2 to 5).
- SO-2 <u>Comment</u> Hydraulic system reliability does not have a good track record in existing ARMY artillery systems.

Reply - See SO-1.

SO-3 <u>Comment</u> - Assuming stored energy is insufficient to emplace the weapon, the time required to manually pump up the system is viewed as an operational deficiency.

Reply - The system can be emplaced for firing by locking out the energy recovery accumulator and using the hand pump to extend the platform and elevate the cannon. (Takes 2 men approximately 2 minutes.) The time required for manual pumping is approximately 10% more than with a mechanical actuator. The procedure for locking out the accumulator is defined in the Appendix Al, page 4, under the hand pump.

Comment - The secondary or offset trunnion appears to be vertically displaced by approximately 4 ft. The ability to effectively engage direct fire targets is questionable due to parallax conditions between the bore and direct fire sight centerlines.

Reply - The fact that the tube is further removed from sight results in parallax error. The parallax error, which results from differences of carriage pitch, is negligible. The correction for parallax is included in the direct fire range plate. At 10° carriage elevation the parallax error is 0 and at 0° and 20° carriage elevation the error is

maximum (approximately 8 inches). The bore sight template is included in Appendix B-4.

SO-5 Comment - The configuration necessitates the bore centerline to be approximately 18 inches from ground level. The ability to effectively engage direct fire targets is questionable due to trees, brush, rocks, etc. between the howitzer and target.

Reply - Discussion of the problem:

Since the fire control is mounted on a "secondary trunnion" that is at the same trunnion height as the M198, the problem has been assumed not to be seeing the target, but rather the projectile trajectory. The trunnion height of the FMC LTHD, at 18 inches, is 30 inches below the four foot M198 trunnion height. Thus, the trees, brush, and rocks that will cause the problem will be those that would have been less than 30 inches below the trajectory of direct fire from an M198 at zero QE.

Because the FMC LTHD trunnion is further from the muzzle, this delta goes to zero at QE's above 12 deg. Thus, a second qualifier is for QE's below 12 degrees.

Reason for low trunnion height:

The FMC LTHD trunnion was dropped to 18 inches for firing and towing stability. While the reduced trunnion height is very beneficial toward achieving firing stability, achieving towing stability was a primary driver in reducing the trunnion height.

Initial towing stability analyses clearly indicated the criticality of lowering the center of gravity, particularly in around-the-curve stability while hitting a bump.

Widening the wheels (beyond 96 inches to the 110 inch M198 width) addressed the around-the-curve stability, but it also adversely impacted towing mobility and loading onto C130 aircraft. Thus, the decision to drop the trunnion to 18 inches was made. Towing stability now appears (theoretically) to be superior to the M198 in straight lines as well as around curves (although additional stability is still available by increasing the width to over eight feet).

The Trade-Off:

Towing stability likely to be more critical than a direct fire trajectory 30 inches below that of the M198 at zero QE (15 inches at QE's of 6 deg)?

While at Fort Sill watching M198's being emplaced, we

observed the wheel of one of the three units bounce almost a foot off the ground due to a small bump, at a speed of 4-5 mph, at approximately a 65 foot turning radius, during positioning in the field.

<u>Summary</u> - Given the current doctrinal employment methods, we have found insufficient justification to warrant trading towing stability for 30 inches of trajectory height (at zero QE) to miss bushes, rocks, and trees while engaging a direct fire target.

SO-6 Comment - The firing crew is totally separated from the breech area. Visual inspection to verify complete charge bag insertion is impossible. This is viewed as a safety hazard and an operational deficiency.

Reply - The firing crew is totally separated from the breech area when the maximum rate of fire procedure is necessary. Visual inspection to verify complete bag charge insertion is difficult, thus the Rammer Position gage is provided to indicate when the ram is fully extended. See Appendix B2.

Under the continuous rate of fire procedure option, a cannoneer is stationed beside the breech. Swabbing, bore cleaning and bag insertion are then performed manually.

See Appendix B3.

Tests performed at FMC support the feasibility of power positioning a bag charge. During a study conducted at Northern in 1983, a 155mm Howitzer Brassboard Autoloader was modified to test the concept of mechanically positioning bagged propellant. In addition to determining friction characteristics between the propellant bags and the autoloader components, the test program determined whether the bags could withstand the wear and tear of mechanical handling.

Results include:

- The various ramming tests on the bagged propellants demonstrated that the speeds and impacts encountered caused no apparent damage to the bag material. No ripping or tearing occurred; and the tie strips did not untie, either in the long charge or short charge configurations. Several ram cycles were completed with twisted tie strips with no problem.
- o It was possible to demonstrate consistently repeatable seating distances using a buffing orifice which provides 1/4 G deceleration.
- The bags were durable enough to withstand the speeds

necessary to meet the ramming cycle time constraints.

- o When ramming tests were conducted with a large rammer pawl-to-bag gaps (up to 20 inches), the ramming impact produced no ill-effects on the propellant bags.
- seems that mechanical ramming of bagged propellant is quite feasible. Since these tests did not indicate how sensitive the actual propellant chemicals will be, further testing may be required. If necessary, however, softer starts in the ram cycle can be accomplished by slight changes in the ramming system configuration. Bag "hang-ups" can be eliminated or minimized with a configuration with smooth surface transitions.
- o Test results proved the feasibility of the following propellant rammer force/time profile:

3.0 G acceleration for 0.10 sec, 0.30 Gdeceleration for 1.05 sec. Total stroke length:70 inches, total time: 1.15 sec.

The above results were summarized from FMC E.C. 1165,
"155mm Howitzer Brassboard Autoloader (DSWS), Determination
of Sliding Friction Coefficients for Bagged Propellant." 10

August 1983, and is available upon request. Slow-motion films of the tests are also available.

SO-7 <u>Comment</u> - Visual inspection of the bore area and verification of bore clear procedures are extremely difficult; if not impossible.

Reply - See SO-6.

SO-8 <u>Comment</u> - During blackout operations, no visual verification of the loading procedure is possible.

Reply - Operation recommended here is the same as discussed in the continuous rate of fire procedure discussed in SO-6. Visual verification for blackout is accomplished manually, similar to what is presently done on the M198.

SO-9 <u>Comment</u> - During daylight hours no visual verification of dolly (loading tray) engagement with the breech, correct ramming of the projectile, and correct position of the powder charge for ramming is possible.

Reply - The rammer cannot be positioned for ramming if the loading tray is improperly aligned with the breech. The rammer will not ram--and will be so indicated by the Ram

Position Pressure Gauge--if the rammer position is incorrect. The ram position pressure gauge will indicate full seating of the projectile.

For continuous rate of fire operation, the cannoneer manually installs the charge and monitors the position.

For maximum rate of fire operation, if the charge holder is incorrectly positioned, the rammer will not move and the ram position pressure gage will so indicate.

SO-10 Comment - Power ramming a powder charge is viewed as a high risk operation. The charge is not rigid, and ripping or tearing the bag is possible if charge alignment is off. In addition, no visual verification by the crew is possible if the charge "hangs up".

Reply - See SO-6.

SO-11 <u>Comment</u> - Mechanical ramming by pushing on the powder charge igniter pad is viewed as high risk.

Reply - See SO-6.

SO-12 <u>Comment</u> - Bore swabbing and breech clearing through water spray and brush is totally unacceptable.

Reply - In light of the problems associated with water spray at subzero temperatures, we have modified our concept for bore swab:

For the continuous rate of fire operation, the cannoneer will manually swab and check for a clear breech as indicated in SO-6. For maximum rate of fire operations, the forward end of the rammer has a bristle-brush that brushes the chamber walls when the projectile is being rammed and when the rammer is being retracted, as discussed in Appendix Al-8, Ram.

SO-13 <u>Comment</u> - No discussion of system disengagement with prime mover is discussed.

Reply - Disengagement from the prime mover is accomplished by extending the platform (in order to reduce tongue weight) and then manually disconnecting. This procedure is discussed in detail in Appendix Bl.

SO-14 <u>Comment</u> - Manually "jockeying" around the dolly is viewed as difficult on uneven terrain, considering the weight of this item.

Reply - The dolly weighs approximately 665 pounds of which about 1/3 is rolling weight. For a speedshifting operation, it is necessary to move the dolly forward approximately 20 feet. For direct fire operations below 100 mils, the dolly must be moved forward and to the side. This movement will be more difficult on uneven terrain and alternate methods of movement, such as using a cable to raise two wheels to simplify rotations, are being investigated.

FMC Design Adequacy and Producibility

DAP-1 Comment - The effect the metal liner has on the composite structure in the recoil cylinders must be addressed.

Thermal coefficients of expansion must be considered in order to preclude delaminations.

Reply - The effect of the differences in thermal expansion coefficient of the steel liner versus the composite overwrap in the York Aerospace hydraulic cylinders is the issue. York Aerospace (a potential subcontractor) claims to have had no problems with this in any of their previously designed cylinders. The question is then, has York built cylinders in similar sizes to those proposed on the LTHD. Part size is in direct proportion to any differential expansion problems that may occur. York Aerospace claims that they have built many accumulators with the same diameter as those on the LTHD but only up to about half the length.

York Aerospace has, to date, successfully dealt with thermal expansion problems by developing a process by which proper tension in the composite overwrap is used to keep the steel liner in radial compression at all pressure and temperature extremes. Accumulators they have built for

aircraft have gone through qualification testing in which they were cycled 1,062,000 pressure cycles at temperatures ranging from -65 to +160 degrees fahrenheit with no affect to the unit.

Other tests included a burst test to 12,000 psig, a proof test to 6000 psig, thermal shock from -65 to +160 degrees Fahrenheit, shock and vibration testing per Mil-Std-810C, and a fragmentation test per Mil-C-7905E, which uses an incendiary projectile to fragment the vessel.

The only thermal expansion concern not addressed is that along the length of the vessel. York Aerospace does not believe this to be a problem but we have requested that since they have not built cylinders as long as those proposed for the LTHD that some simple analysis be done to provide a value for the axial stress on the composite overwrap. This stress would be caused by the differential thermal expansion of the steel cylinder and the composite overwrap.

Al/SiC is also a strong contender for the recoil cylinders. See section RM-2.

DAP-2 <u>Comment</u> - Several problems exist in the slide concept configuration. The thin wall honeycomb sandwich

construction is both impact and moisture sensitive.

Reference to acceptable aircraft construction does not indicate adequacy for howitzers.

Reply - The questions on the use of honeycomb material are good ones and well founded based upon some of the Air Force experience with it. However it must be realized that most of this experience relates to repair experience on honeycomb structures built ten or more years ago.

Honeycomb technology has changed considerably in that time. Honeycomb materials were considered in the phase 1 concept mostly for their ability to offer good mechanical properties and low weight in areas of higher shear stress.

FMC will look more closely at alternate core materials for even the high shear stress areas and evaluate whether weight targets can be met with alternate materials. In the event that weight requirements leave honeycomb as an only alternative the following arguments and design procedures will be used to alleviate the review team concerns.

Impact Sensitivity:

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On the impact sensitivity of honeycomb three things should be noted. First, it is standard design practice to use honeycomb with a cell size smaller that 3/16 inch in applications where impact resistance is a factor. Experience has shown that this cell size significantly increases impact resistance as compared to larger sizes.

Second, although some of the skin thicknesses shown in work done for phase 1 are indeed quite thin, the work showed that at least structurally the concept was feasible.

Increased skin thickness will also contribute greatly to impact resistance. Skin thickness can be increased without increases in weight by evaluating the physical size of some of the structures and optimizing the size, weight, and skin thicknesses.

For example, it has been suggested by the review team that the size of the trails seems too big. Reducing section and increasing skin thickness can reduce weight and improve impact resistance. Efforts such as this to maximize skin thickness within weight limits will be made. Thirdly, it is also possible in critical areas where damage may occur to add impact resistant layers of composite over the relatively fragile graphite/epoxy.

Moisture Absorption:

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Moisture absorption of honeycomb structures has been a problem in the past in three areas; corrosion of aluminum honeycomb, water droplets forming in the cells, and difficulty in repair of structures with water in the cells. In the last several years much work has been done in the area of coatings to eliminate corrosion on

aluminum. Aluminum core manufacturers now provide special coatings on the core material which the industry believes will eliminate the corrosion problem. To the extent that aluminum honeycomb is required to meet weight targets FMC will attempt to provide sufficient manufacturer supplied data showing the ability of a particular core product to pass moisture or salt spray corrosion tests.

There is a great deal of discussion within the industry at this time as to how the water gets into the cells and what other negative effects it may have on the structure.

Indications are that in many cases the water enters through damaged spots on the skins which should have been repaired or that the moisture was built into the part by lack of proper humidity control during manufacture. There has been no evidence that the moisture has a negative effect on the structure. Permeation of moisture through the skins has been known to happen after prolonged exposure but this too has been minimized by the use of special sealant coatings.

FMC commonly uses a coating manufactured by Ram Chemicals. The problem of moisture during repair of honeycomb structures can for the most part be eliminated by proper drying of the part prior to repair.

DAP-3 <u>Comment</u> - Pultruded rails are very risky. The likelihood of premature failure due to matrix cracking at the corners

SCORES SECTION

is high. This is caused by inability of the fibers to reinforce the corners. The concept of making four plates and joining them with preformed corners is inadequate. Filament winding in this area would produce a superior structure. Galvanic corrosion and adhesive selections should be addressed.

Reply - It is agreed that pultruded rails are not the best choice for this design. Metal or metal matrix materials are being evaluated for this component. Preliminary finite element analysis is showing that weight budget can be satisfied using conventional materials.

Corner Connections:

FMC is currently working on a slide design that will be made of either two tape laid pieces, a top and a bottom piece, which will be bonded together at the centerline of the rails, or a continuous tape wound part with appropriate cutouts and wound-in joint reinforcements. Both concepts eliminate the corner connections. This could be a filament wound part in production. The cost implications of making this part from a filament winding will be investigated and reported to ARDEC.

Galvanic Corrosion:

It is true that galvanic corrosion will occur if graphite fibers contact aluminum. In all areas where aluminum is used one of a number of film adhesives will be used which are specially made to electrically insulate the graphite fibers in the composite from the aluminum.

DAP-4 <u>Comment</u> - Honeycomb core construction in the trails is a concern due to impact damage and moisture absorption.

Preformed woven corners seem inadequate. Complexity and cost would be high with this concept. The choice of a factor of safety approach is not as adequate as hot/wet property data.

Reply - Impact and Moisture Absorption:

See DAP-2

Corner Connections:

In this particular application FMC felt that a design using four plates connected at the corners is a lower risk approach in terms of cost and schedule as it relates to a single prototype. This construction method has been proven in other applications to be an acceptable method for connection in the presence of shear loads. It is recognized however that it does introduce some weight

penalty and that on a production basis the part would probably be filament wound. FMC is therefore investigating the possibility of tape winding the structure which would be more representative of a filament wound structure.

Factor of Safety Versus Hot/Wet Property Data:

FMC standard procedure is to use nothing but hot/wet property data as material allowables with the traditional factor of safety added on top of this. This was not explained well in the phase 1 report and we apologize for the omission. Hot/wet data was not used in the Phase I analysis because it was not available at that time. Hot/wet data is included in Appendix C2 and will be used during Phase II.

DAP-5 Comment - The choice of graphite epoxy skin with aluminum honeycomb core in the platform area invites two potential problems. One is the moisture effects on honeycomb construction and two is the shielding of aluminum from graphite to prevent galvanic corrosion. Hand lay-up and autoclave cure is reasonable for these parts.

Reply - Moisture and Corrosion

See DAP-2.

DAP-6 Comment - The "claws" are constructed as described above and similar problems are foreseen. The claw should be designed for damage tolerance since it is expected to interface with the ground. Graphite epoxy skin is inadequate for this purpose.

Reply - Damage Tolerance of the Claws:

It is agreed that the claws must be made more damage tolerant than the graphite epoxy material concept proposed in Phase I. This part will probably be made from aluminum or titanium. A tubular space frame is one contender. FMC is also currently working with Astech Corporation to evaluate designing the claws from a steel corrugated core with steel face sheets. This design can be impact hardened at critical locations with welded-in steel inserts. A sample of this material has been forwarded to Adolph Slobodzinski at ARDEC. Weight/cost/function trade-offs will be evaluated and a recommendation made to ARDEC as part of the regular scheduled design reviews.

DAP-7 <u>Comment</u> - The "backbone" is constructed using a hand lay-up procedure rather than being filament wound. This is considered a poor approach for this component. Removing the

mandrel to form a concave section may result in a wrinkled, void filled structure that will not possess the desired mechanical properties.

Reply - Dolly Backbone Construction:

The dolly backbone has been reconfigured to address this concern as well as to minimize/eliminate pinch points.

DAP-8 Comment - All components should be inspected by ultrasonic techniques. No indication of inspecting for moisture absorption exists. This must be addressed.

Reply - Inspection methods will be discussed in the Product Assurance Plan. Inspection methods will be matched to the part design and material selected.

DAP-9 <u>Comment</u> - The bolted joint method of integration is acceptable; however, corrosion and sealing of the holes from moisture should be addressed.

Reply - Sealing of Bolt Holes:

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Corrosion in the areas of bolted joints will be addressed the same way as corrosion of aluminum honeycomb. The

aluminum inserts and shims will be anodized and separated from the graphite fibers with a film adhesive. Most holes will not require sealing in that metal bushings will be bonded into the composite holes to distribute contact stresses. Those holes without bushings will either have metal threaded inserts or will be sealed with a resin sealer.

DAP-10 <u>Comment</u> - Bulkiness in trail design is also a concern. It is not clear what weight optimization process ruled out a more reasonable selection of dimensions. The construction does not seem to take damage tolerance and environmental degradation factors into account.

Reply - The trail section depth was chosen to:

- Maximize resistance to bending (caused by support
 of at-battery cannon weight), which in turn
 maximizes system elasticity, thus minimizing
 propensity to hop at zero QE fire.
- Spread trail bearings as far apart as possible to reduce bearing loads, which in turn minimizes weight.

3. Facilitate torsionally locking the trail to the upper and lower portions of the platform, serving further to maximize stiffness while minimizing weight.

Smaller forward sections are being evaluated to provide a more damage-tolerant structure which is light weight.

See DAP-2 and DAP-4 for additional discussion on the trail design.

FMC Structural Verification

SV-1 Comment - Beam stress calculations on the supporting trails which allow 70 kpsi in a quasi-isotropic lay-up of graphite epoxy seem suspect. The allowed stress cannot possibly be greater than 50 kpsi for a quasi-isotropic lay-up, even before introducing moisture effect degradation factors.

Reply - Allowable Stresses:

In many cases in phase 1 the starting point for our calculations was to take a reasonable stress value and use the force over area formula to arrive at a starting point for the laminate strain calculations. 50,000 psi was chosen as a reasonable value and this can certainly be argued. However in all cases the report contains at the end of each supplementary section a copy of the computer printout in which a ply by ply analysis was done on the proposed laminate. The trail design is being analyzed using a composites laminate optimizer program. This analysis takes into account the resultant dynamic loads as indicated by the dynamic analysis and the actual published ply-by-ply allowable strains. The allowable ply strains are mathematically transformed to the ply axis. The result is an analysis of first ply failure whether it be axial or

any angled plies. A factor of safety is applied to these results to guarantee that all ply strains are significantly below the failure strains. Again, it is agreed that hot/wet property data should be used for the design allowables and will be in the detailed analysis in phase 2. In addition appropriate knockdown factors for impact resistance will be incorporated.

SV-2 <u>Comment</u> - There is also a disregard for environmental effect, including hot/wet conditions and impact damage degradation effects.

Reply - Environmental Effects:

In phase 2 hot/wet property data will be used for the proposed materials and used as the design allowable. We have determined the proper stacking sequence to give maximum strength for minimum weight for Gr/Ep filament wound trails as discussed in reply to SV-1. Loads are being determined using finite element analysis as part of this design process. Preliminary calculations have shown that maximum stresses seen in any lamina is about 15 KSI which is well within strength limits of the materials used. Calculations have also been run for hot/wet conditions to calculate the stress/deflection levels. This

information will be reviewed with ARDEC at the next scheduled design review or by arrangement with program manager.

Impact Effects:

Damage resistants will be determined from published data or by test to evaluate the effect of impact on the proposed sandwich structures.

SV-3 <u>Comment</u> - The claimed skin thickness in the trail beam of .019" in the sandwich flange is suspect.

Reply - Trail Skin Thickness:

The skin thickness is 0.19" not 0.019" for the Gr/Ep filament wound trails. We have used programs available from the University of Delaware to obtain mechanical property values for hot/wet conditions. Changes in part dimensions, loads, hot/wet material data, and impact criteria requirements will be factored into the analysis and design in Phase II.

SV-4 Comment - The pultruded rail design is suspect. The corners

are vulnerable to cracking when the structure is loaded in bending.

Reply - See DAP-3.

SV-5 <u>Comment</u> - The graphite epoxy skin of the claws appears to be vulnerable. This area should be addressed utilizing a damage tolerant design criteria.

Reply - See DAP-6.

SV-6 Comment - Strength calculations are questionable because of the dry room temperature values used. The knockdown factors are generated from the Tsai-Wu equation; however, the stresses calculated have sketchy rationale. The documentation does not indicate how these stresses are derived.

Reply - See SV-1

SV-7 <u>Comment</u> - No justification has been given for the size of the trails. Collaboration on the design methodology of these components is desired.

Reply - See DAP-10

FMC System Stability

SS-1 Comment - A trapezoidal recoil force profile is most desirable but not always obtainable. Maintaining this profile through 8 and a half feet of recoil travel will be difficult, especially in light of the "energy taps" employed for the hydraulic system. Temperature induced gas pressure changes, as they affect recoil force, appear not to be adequately addressed. A "sensitivity" analysis would be helpful.

Reply - The intent is to determine the "ideal trapezoidal recoil force profile" that has enough margin to permit reasonable deviations in actual practice.

The current margin between ideal and tolerable is 12.8% (see Dynamic Analysis Report pages 20, 22). The primary causes of variations are expected to be from:

- 1. Calculation variance (predicted versus actual)
- Impact of temperature upon the nitrogen pressure in the counterrecoil accumulator, the fluid viscosity, and the variations in the components.

- Variations caused by status of energy recovery system during firing
- 4. Manufacturing tolerances

Correction of the calculation variance could be one by-product of the demonstrator. Variation due to ambient temperature fluctuations are compensated for as described in the following paragraph. Variations due to the status of the energy recovery accumulator are expected to be roughly 4.5%. Thus, tolerable variations due to manufacturing tolerances must be less than 8.3% to maintain stability. We feel this is realistic.

The nitrogen pressure in the counterrecoil accumulator can be adjusted via the Recoil System Volume Control Valve (recoil system pressure is monitored by the Recoil System Pressure Gauge). Adjusting this pressure as a function of system temperature will facilitate inclusion of viscosity and dimensional variations which result from ambient temperature variations. (This would probably be most easily accomplished via a gauge face calibrated in system temperature rather than pressure, thus eliminating a conversion table in the manual.)

SS-2 <u>Comment</u> - Counter-recoil stability is mentioned but not fully investigated.

Reply - Section 4.2 of the Dynamic Analysis Report presents the analysis to date in this area. As long as the decelerating force during counter-recoil does not exceed 5,000 lbs, no stability problems are anticipated. Initial analysis indicates that counter recoil stability is not a problem. When the counter-recoil force profile is better defined, this concern will be reevaluated and the results presented to ARDEC.

SS-3 <u>Comment</u> - Rifling torque appears not to be included in stability computations.

Reply - Rifling torque computations were performed. Page 38 of the Dynamic Analysis Report reported the results, but no computations were included. The computations are included in Appendix C3. The calculated side hop is .036" (Dynamic Analysis Report reported "not ... more than 0.25 inches").

FMC Cannon/Ammo

CA-1 <u>Comment</u> - Cannon interface was not adequately determined until after Phase I final review.

Reply - We agree. Information was provided to Russ

Fiscella and Malcolm Dale from Benet Weapons Lab and Steve

Floroff from ARDEC on 13 June 1986 at Benet.

CA-2 <u>Comment</u> - The automatic breech opening mechanism for this design has not been determined.

Reply - We have not provided ARDEC with a drawing of our concept. A concept for a breech opening mechanism was discussed at Benet on 13 June 1986 with above mentioned individuals (CA-1 Reply). Concept was agreed to in principle and details will be provided on the concept interface drawing which will be provided by October 17th to ARDEC.

CA-3 <u>Comment</u> - The primer feed mechanism has not been adequately defined.

Reply - A primer feed mechanism is being developed by FMC for another demonstrator program. The primer inserter mechanism has gone through first generation development prototype testing and is being redesigned based on the results of these prototype test and evaluation results. We are scheduled to have a second prototype available by April 1, 1987. Plans are to fabricate a primer feeder to this design for use on the LTHD program.

Seeses and Property

FMC Recoil Mechanism

RM-1 Comment - The effects of the external cylinders used for power recovery and storage on the recoil cycle has not been addressed.

Reply - The effects of the external cylinders has been addressed in the System Stability section under the SS-1 reply.

RM-2 <u>Comment</u> - The manufacturing of an effective ten foot recoil mechanism is of great concern.

Reply - The 105 inch max recoil stroke length, of which up to 102 inches will be used under worst-case firing conditions, will be 35 inches longer than the M198 recoil mechanism (which has a 70 inch max stroke). Our Advanced Manufacturing personnel (familiar with the M198 recoil mechanism) do not feel the 105 inch recoil stroke by itself is higher risk than the M198 recoil mechanism. We are considering both composite and metal cylinders. If metal matrix (AL/SiC) material is employed, there would be some risk in developing manufacturing processes, although the

dimensional stability issue (primary problem on M198) should improve.

RM-3 <u>Comment</u> - A trapezoidal shape recoil force vs. time curve is utilized for all analyses. A more conservative approach in which some room is allowed for non-ideal circumstances is preferred.

Reply - The use of a trapezoidal shaped recoil force curve has been addressed in the System Stability section under the SS-1 reply.

FMC Organization

O-1 Comment - There appears to be a reluctance by FMC management to bring forth supporting documentation of their efforts. This position hampers not only the FMC effort, but the program as well by not allowing adequate monitoring of progress made. A change in this area will be most mutually beneficial and desirable.

Reply - FMC has provided ARDEC all contractually required data plus approximately 50% more data than was planned and bid under Phase I contract. FMC has made every effort to cooperate with ARDEC to provide requested data. The problem noted in this comment stems from a contract SOW and DD Form 1423 which does not accurately indicate the data desired by ARDEC. This has caused FMC cost and schedule problems particularly in the last month of Phase I.

FMC will work more closely with ARDEC during Phase II to avoid last minute changes and changes of direction. Every effort will be made to provide ARDEC preliminary information for comment for timely feedback and monitoring of progress.

CEL and Northern Ordnance. It is felt that the FMC composite experience is very limited - perhaps to that associated with the Bradley Fighting Vehicle. The primary emphasis and design philosophy brought forth in the Light Weight Towed Howitzer is that of metal construction. If this philosophy and outlook does not change, grave design errors are likely to occur.

0-2

Reply - CEL's involvement in the Phase I effort represented than 10% of the effort and consisted of: 1) creating a finite element model of the system for structural analysis and 2) towing stability simulation. The selection of the materials required based on detailed analysis is planned for Phase II. The background information which was presented at the June 4th review was not intended to answer the detailed application questions which resulted at that review.

FMC has significant experience in composite parts development and has invested several million dollars in equipping a laboratory to build composite parts. A list of experiences include Bradley turnet and turnet basket, M113 hull, torpedo dolly and stowage tray, and MK45 weather shield. FMC has engineering personnel with over 20 years of composite design experience with companies including Boeing, Sikorsky, Fiberite and Beach Aircraft. In

addition, qualified vendors are being selected, an example being York Industries for composite hydraulic cylinders.

We understand the need to utilize composites to meet the weight targets. Composite designs are being investigated for the trails, slide, spade assembly and gimbal; heavy parts which offer the greatest potential for weight reduction. Preliminary designs for these parts are being developed and will be ready for review in late September.

O-3 <u>Comment</u> - It is unclear why this effort has both a program manager and project manager. Discussion of this question with Ms. Tia Stackland resulted in a poor explanation.

Reply - FMC's program management organization has consisted of a program manager from the business side of the organization and a project manager from engineering to direct the technical activities on development projects. This was the program management structure shown in the approved program plan for the LTHD program. FMC recognized that at times this was like dealing with a two-tiered management structure and the customer did not understand the relationship or the necessity for both.

Because of comments such as yours, we have made a change to our project team structure. A program manager who has

unilateral authority and is a senior level person has been assigned singular leadership responsibility. A contract administrator is assigned to assist him or her and handle many of the contract issues. The product manager from business development will continue to handle the marketing activities.

0-4 <u>Comment</u> - The entire cannon interface portion of this program was poorly executed.

Reply - This comment refers to the subcontract relationship which was originally proposed for acquiring a cannon assembly from Watervliet Arsenal (WVA). Watervliet had not worked as a subcontractor previously, but had been told that the relationship would be legally acceptable. Upon exploring the matter further, Gerry Cooper from Rock Island Arsenal indicated that the relationship represented a conflict of interest as it relates to the other competing contractors. It was then decided, by the government, that the cannon assembly should be furnished GFE. This area has been a learning experience for both FMC, WVA, RIA, and ARDEC.

O-5 Comment - Program has undergone a shake-up in management midway through Phase I. This is apparent in the organization of this entire program. Numerous significant design changes up to the conclusion of Phase I indicate that personnel are not clear on their direction.

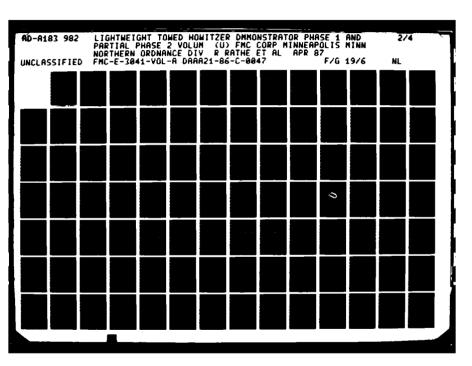
Reply - Every effort was made by FMC management to communicate the reasons for changing product managers in May on this project. The remaining team members were with the program throughout Phase I with the duties proposed in the approved Program Plan. The design changes and changes in direction were the results of FMC's attempts to be responsive to comments from ARDEC, eliminate design deficiencies and provide an acceptable concept. Many of the last minute changes were the result of input at the June 4th review.

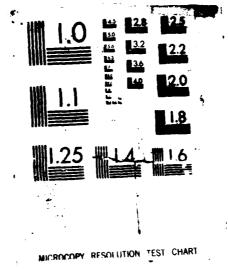
Additional reviews and meetings are planned with the customer during Phase II to provide improved feedback and communications with ARDEC and reduce confusion and chaos caused by engineering changes.

See also reply to comment 0-3 relative to Phase II.

O-6 Comment - Technical personnel seem well motivated however, numerous design changes apparent at the final review indicate they also are unclear of their direction.

Reply - The reason for last minute changes is explained above in 0-5 and were caused by trade-off analysis and our desire to overcome operational deficiencies identified at the June 4th review.





FMC Quality Assurance

QA-1 <u>Comment</u> - The inspection facilities that FMC has shown are predominantly laboratory testing apparatus. They are able to test coupons and tensile samples, but it is felt that large, full size, howitzer structures are going to pose a problem.

Reply - CEL has tested entire Armored Personnel Carriers in its facilities without any difficulties. Therefore, we do not foresee any problem with testing any large, full size, howitzer structure. A detailed layout of the test facility with a description of capabilities will be provided at the September 5th review.

QA-2 <u>Comment</u> - The honeycomb composite structure used throughout the concept has a moisture absorption characteristic that results in decreased mechanical performance. Because the proposed outside skin is an organic composite it will be very difficult to inspect for moisture content.

Reply - The Gr/Ep-Honeycomb structure proposal represents one possible design approach - not necessarily the final

one to be taken. As part of Phase II analysis work, a number of types of materials will be considered and analyzed before final selection is made.

The honeycombed structure was proposed because of its high strength/weight ratio. Material properties used will reflect worst case moisture absorption physical constant test information. See also significant discussion on this topic in the Design Adequacy and Producibility Section.

QA-3 <u>Comment</u> - FMC has given little consideration to producibility aspects - the hand lay-up techniques proposed are very expensive.

Reply - We will use the most cost effective method to produce the demonstrator parts. This will not always mean, that multiple part fabrication would be achieved using the same method. We will, whenever feasible within budget and schedule, include details about different fabrication techniques we would recommend to be used in multiple fabrication. Cost trade-offs will be presented to ARDEC as part of the design review process and they will have input to the manufacturing process selected for prototype parts.

QA-4 <u>Comment</u> - The loader feeder for primers is a complicated unproven device - this can be a major RAM problem.

Reply - We share the concern and are monitoring closely in-house progress as well as progress with primer feeder mechanisms by others in order to provide a reliable component. A primer feed mechanism is being developed by FMC as discussed in the CA-3 reply.

QA-5 <u>Comment</u> - The cantilevering of the gun may cause fire control problems because of vibrational modes and/or instabilities.

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Reply - The entire LTHD system is set up as a finite element model. Alternate materials will be investigated to insure that structure is not excitable at a natural frequency with resulting instability. If preliminary analysis indicates problems, a more detailed modal analysis can be performed.

QA-6 <u>Comment</u> - The skin of the so called "claws" is proposed to be graphite epoxy. It is hard to foresee this construction surviving the type of abuse it will receive.

Reply - We agree. If the claw must be able to withstand hitting by a sledge hammer, a metal material such as aluminum or titanium would be much more damage tolerant. We will include this concern as one of the design

requirements for the claw in Phase II and determine the weight trade-off.

QA-7 <u>Comment</u> - The equilibration cables seem to be a potential RAM problem.

Reply - The equilibration cable design and the RAM implications are discussed in detail in Appendix C-1. The cables are designed to provide redundancy where one cable will support the entire load with a generous safety factor of 8 for maximum static loads. Our analysis shows that the load on the cable never goes to zero. Normal working loads are under 10% of the breaking strength. Cables have been designed as recommended in the design guides and redundancy will provide a safety factor. They are utilized because of their light weight for tension loading applications.

QA-8 Comment - The P.A. plan is scarcely addressed.

Reply - A draft Preliminary Nondestructive Testing and Product Assurance List developed under C.2.C.1.d of the SOW was submitted for review to Mr. Joseph Argento (AMSMC-QAH-T) on 29 May, 1986 (cc. Norm Lionetti, SMCAR-FSA-F). We believed that it satisfied our submission requirement for Phase I. We interpreted C.2.C.2.e as required to be developed in Phase II. The product

assurance plan is presently under development and our goal is to review draft plan with Mr. Argento by the middle of September.

QA-9 <u>Comment</u> - It appears that little consideration has been given to the formulation of an impact damage assessment.

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Reply - We agree. Damage criteria were not established in the first phase and the primary effort was focused on meeting the weight criteria and satisfying operational requirements. We did not cost significant effort in this area in our proposal and we are assessing the impact on program cost. Materials selected will be evaluated for damage resistance during Phase II with the damage criteria established through agreement with ARDEC. An impact damage assessment for mission critical composite parts will be part of our preliminary product assurance plan.

QA-10 <u>Comment</u> - There is no indication of any thought given to the delamination problem which may occur from expansion between over wrapped cylinders and the composite wrapping.

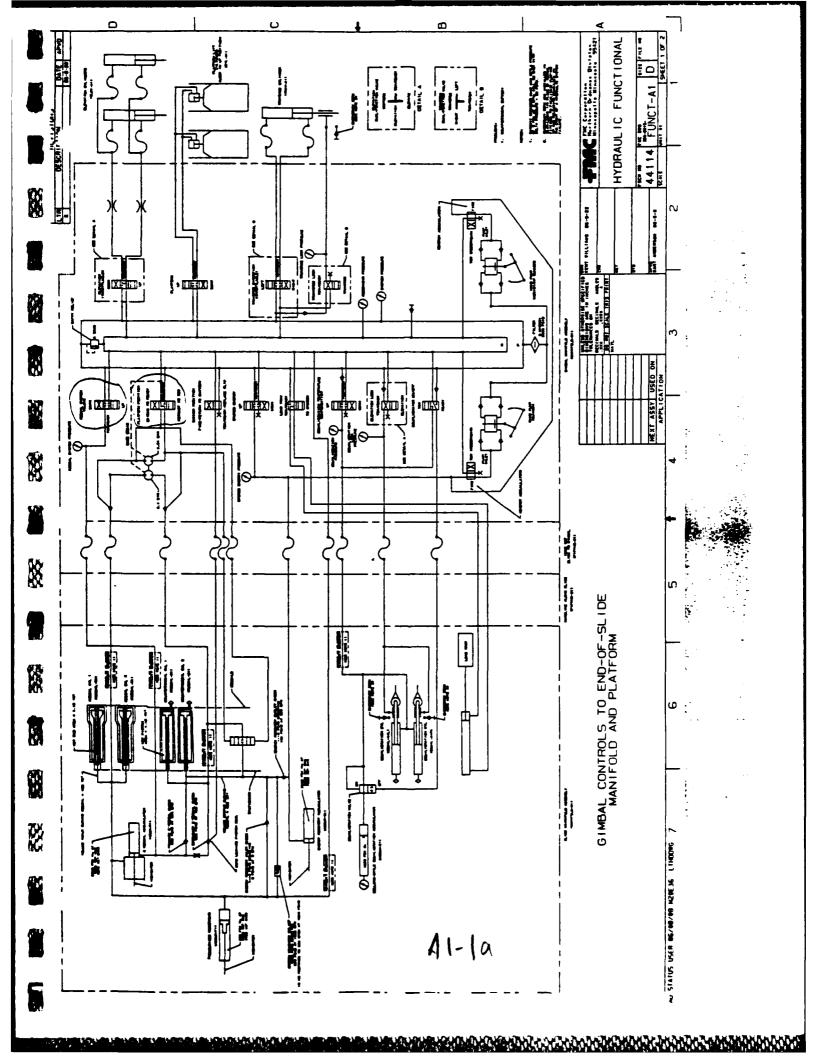
Reply - This comment raises similar issues to that raised previously under DAP-1. See DAP-1 for answer.

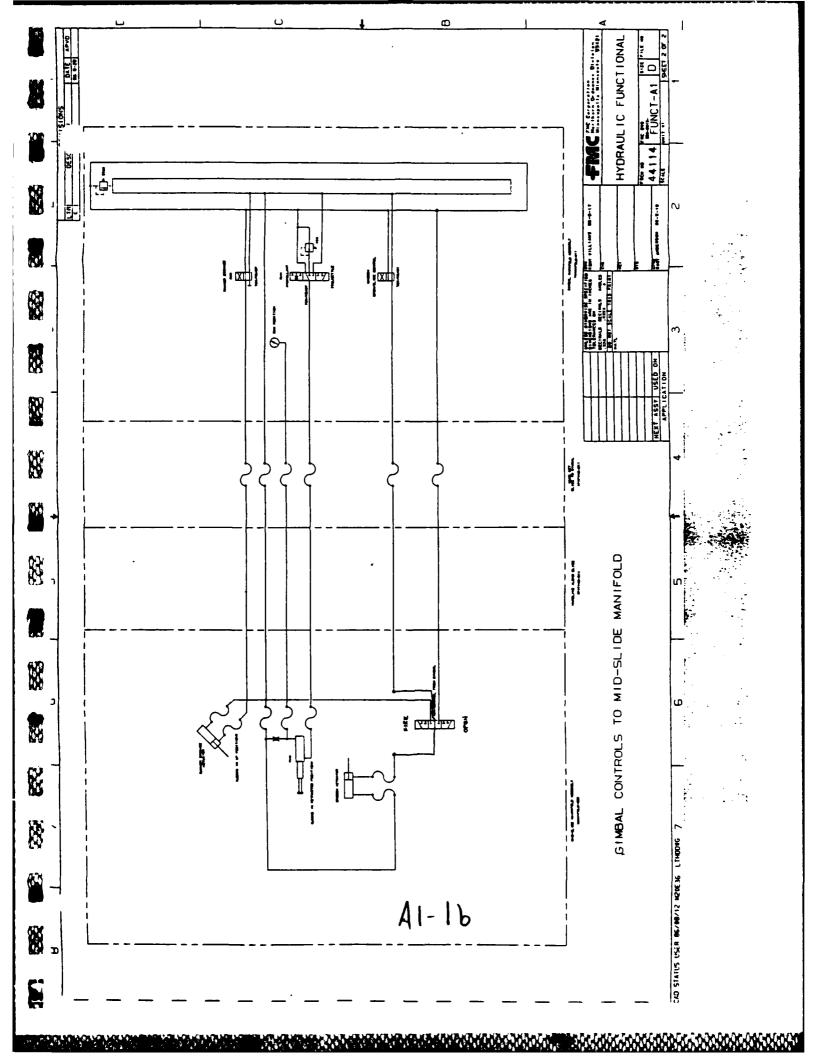
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- QA-11 <u>Comment</u> The recoil slides/cradle is viewed as having three serious potential problems:
 - 1. Degradation by moisture absorption.
 - 2. The method utilized for joining the four sides appears very similar to ones used for metal fabrication. This structure would seem to be much more efficient if filament wound.
 - 3. In the chosen configuration, the glass or graphite pultrusions that are to make up the slides seem very prone to develop cracks due to a lack of reinforcement inherent in the pultrusion process. In addition, residual stresses in the material due to resin shrinkage may be enough to initiate cracks.

Reply - This comment raises similar issues to those raised previously. See DAP-2, 3 and 4 for the answer to this comment.

Device Sender Pomp luce / OUTLET (LIDE This Appendix is prepared to provide a framework for the explanation of the questions relative to FMC LTHD Operations. Accordingly, the following sections support this end. ASD TEMP GAGE . Recoil System Volume......A1-2 Position......A1-3 A1-3 sout in views Keck Esolu 1/2 Equilibrator On/Off......A1-5 Load Tray A1-7 1 VALVE? by interfores MUNAL PLIMER Mid-Slide Controls......A1-9 REPLACEMENT Recoil System......A1-10 Recoil Cylinders......A1-10 Counterrecoil Cylinders......Al-10 Counterrecoil Buffer......Al-10 1 VALVE? Operating Procedures.....B Maximum firing rate (four rounds per minute)......B2 Continuous firing rate option (two rounds per minute).....B3 Note 1: Descriptions assumes that the hydraulic system has sufficient energy storage. The procedures required if this is not the case are covered under the Hand Pump Sections (Al-4 and Al-6). SECOND ECCUATION VALUE - WIVING SYSTEM LOCKOUT





APPENDIX A1

Narrative

Gunner's Controls. These controls are mounted on the left side of the gimbal and move with the gun in traverse. The gunner can operate the hydraulic controls while looking into the eyepiece of the panoramic scope.

Traverse. This is a "T-pattern" dual function control valve.

Tow/Ready. The traverse system is locked.

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- Traverse. Moving the handle upward (against a vertical spring) releases the BearLocs (the Traverse Lock Pressure Gage indicates when the lock releases).
- Traverse-Left. Moving handle to the right (against a horizontal spring) traverses the gun to the left.
- Traverse-Right. Moving handle to the left (against a horizontal spring) traverses the gun to the right.
- Cannon Position. Allows the gun to be caught in recoil for replacement of primer clip.
 - Tow/Ready Replace Clip. The gun position will be held. During firing this position is used to "catch" the gun to facilitate reloading the auto primer clip (every eight rounds). During displacement, this control will hold the gun in tow position until the travel locks are secure.
 - Ready Return to Battery. Moving the handle upward against a spring will return the gum to battery after the recoil cycle.
- Recoil System Volume. Recoil System Volume is measured by the Recoil System Pressure Gage and the position of the Recoil Accumulator Indicator (protruding from the Slide Manifold at the forward end of the Slide).
 - Tow/Ready. The Recoil System is locked up and separated from the Hydraulic System.
 - Up. Pushing the valve to the right against a spring will add oil to the Recoil System.
 - Down. If too much oil is accidentally added, the valve can be moved to the left (against another spring) to bleed off some of the excess oil (back into the Hydraulic System).

- Platform Position. Extends and retracts the platform during emplacement and displacement, as well facilitates removal of stuck projectiles.
 - Tow/Ready. The recoil and counterrecoil cylinder volumes are "locked" (unless the "Cannon Position Control" is being held in the "Ready" position, in which case the gun will go to the battery position).
 - Retract to Tow. To pull gum back into the slide for towing, the Gunner must push the control valve handle down (against a spring) to the the "Retract to Tow" position. Oil is transferred from the Gounterrecoil and Energy Recovery Cylinders to the the Recoil Cylinders (volumes are equal) by a closed loop motor-pump.
 - Extend to Fire. To extend the gun to the firing position, the Gunner must push the control valve handle up (against a spring) to the "Extend to Fire" position. Oil is transferred from the Recoil Cylinders to the Counterrecoil and Energy Recovery Cylinders.

This control can also be used to extract a stuck projectile. The gun is first slightly retracted toward the tow position. Then the pole is set in the bore against the ogive. The strap winch is run around the end of the pole and attached to the forward trail locks. Extending the gun to the firing position facilitates the application of up to 3,500 lbf extraction force. The relative force is monitored by the System Pressure Gage.

- Stored Energy. Facilitates manually adding energy to the Energy Accumulator. The magnitude of energy on tap is indicated by the Stored Energy Pressure Gage.
 - Tow/Ready. In this position, the Energy Accumulator inlet port is connected only to the Energy Recovery Cylinder (manual addition of energy is locked out).
 - Up. Moving the handle upward against a spring, with the Pump Inlet Control in the Tow (Reservoir Input) position, energy can be added to the Energy Accumulator with the Hand Pump.
 - Down. Moving the handle downward against a spring, energy can be bled from the Energy Accumulator (to facilitate servicing the Energy Accumulator).

Pump Inlet. A heavily detented two position control valve that selects the inlet to the hand pump from the Pressurized Reservoir or the Energy Recovery Accumulator.

Tow (Reservoir). The valve should be left in this position during tow operations to ensure that inadvertent operation of valves do not produce unplanned events. Also used when manually pumping up Energy Accumulator (see Hand Pump Section).

Ready (Energy Accumulator). Normally during firing, energy will be supplied by the Energy Accumulator. Oil will automatically flow from the Pressurized Reservoir in the event that the Energy Accumulator's supply goes to zero.

Hand Pump. Balanced area, double-acting, lever-actuated hand pump.

When a control valve is opened (depleting the pressure header of oil), additional oil will flow through this pump (when the Pump Inlet Control is set to Ready).

If the pressure from the Energy Accumulator is insufficient to achieve the desired effect, additional pressure can be added by pumping the handle of the hand Pump.

To manually recharge the Energy Accumulator, set the Pump Inlet Control to Tow (Reservoir Input), set the Stored Energy Control to Up, and pump one or both the Hand Pump(s). As energy is added to the Energy Recovery Accumulator, the Stored Energy Pressure Gage will so indicate.

- Assistant Gunner's Controls. These controls are mounted on the right side of the gimbal and move with the gun in traverse. The Assistant Gunner can operate the hydraulic controls while looking into the eyepiece of the direct fire scope.
 - Elevation. This is "T-pattern" (laid down) dual function control valve that operates two Elevation Cylinders. The Equilibration Cylinders must be unlocked before the gun can be elevated.
 - Tow/Ready. The elevation system is locked.
 - Elevation. Moving the handle to the left (against a horizontal spring) releases the BearLocs (the Elevation Lock Pressure Gage indicates when the lock releases).
 - Up. Moving handle downward (against a vertical spring) elevates the gun.
 - Down. Moving handle upward (against a vertical spring) depresses the gun.
 - Equilibrator Temperature Compensation. This valve is used to adjust the equilibrators for changes in ambient temperature. The Equilibration Pressure Gage monitors the equilibration pressure as it changes with ambient temperature.
 - Tow/Ready. The Equilibration System is locked.
 - Up. Moving the handle upward against a spring adds oil to the Equilibration System, thus increasing equilibration (necessary when ambient temperature drops).
 - Down. Moving the handle downward against a spring removes oil, thus decreasing equilibration (necessary when ambient temperature rises).
 - Equilibrator On/Off. This is a heavily detented two position control valve that turns the equilibrators on and off.
 - Tow. Turns the equilibrators off to facilitate lifting the trails off the ground (when the gun is on the dolly) for towing.
 - Ready. Turns the equilibrators on to elevate the barrel.

- Platform. Used to lift the Platform and in turn lift the main spade out of the ground.
 - Tow/Ready. In this position, the Platform Lift Cylinders are locked in the up position.
 - Up. Moving the valve handle upward against a spring will extend the Platform Lift Cylinders and Raise the Platform (and spade) out of the ground.
 - Down. Moving the valve handle downward against a spring will retract the Platform Lift Cylinders and Lower the Platform (and spade) into the ground.
- Pump Inlet. Identical to one on Gunner's side. Heavily detented two position control valve that selects the inlet to the hand pump from the Pressurized Reservoir or the Energy Recovery Accumulator.
 - Tow (Reservoir Input). The valve should be left in this position during tow operations to ensure that inadvertent operation of valves produce no unplanned events. Also used when pumping up manually pumping up Energy Accumulator (see Hand Pump Section).
 - Ready (Energy Accumulator). Normally during firing, energy will be supplied by the Energy Accumulator. Oil will automatically flow from the Pressurized Reservoir in the event that the Energy Accumulator's supply goes to zero.
- Hand Pump. Identical to one on Gunner's side. Balanced area, double-acting, lever-actuated hand pump.

When a control valve is opened (depleting the pressure header of oil), additional oil will flow through this pump (when the Pump Inlet Control is set to Ready).

If the pressure from the Energy Accumulator is insufficient to achieve the desired effect, additional pressure can be added by pumping the handle of the Hand Pump.

To manually recharge the Energy Accumulator, set the Pump Inlet Control to Tow (Reservoir Input), set the Stored Energy Control to Up, and pump one or both the Hand Pump(s). As energy is added to the Energy Recovery Accumulator, the Stored Energy Pressure Gage will so indicate.

Cannoneer 1's Controls. These controls are mounted on the top of the gimbal (except the lanyard, which is mounted on the right inside wall) and move with the gun in traverse. Cannoneer 1 can operate the hydraulic controls while watching all loading operations. Cannoneer 1 can also see if the bore is clear up to 150 mils, if the bag is in position, and if the breech marks are aligned.

Breech. This heavily detented valve hydraulically opens and closes the breech. The breech can only be opened hydraulically when at battery. At any other position it must be opened with the manual handle provided with the basic issue items.

The breech can also be controlled from just behind the breech, inside the slide, and in front of the Rammer (see Mid-Slide Controls section).

Tow/Ready. In this position, the breech is closed.

Open/Control from Slide. Pushing the valve upward opens the breech. This is also the position the valve should be in if the breech is to be opened and closed from the Mid-Slide Controls location.

Should this control be accidentally moved to the Tow/Ready position while the breech is being controlled from the Mid-Slide Controls location, the breech will not close as long as the Mid-Slide Control for the Breech is in the Open position.

Load Tray. Advances and retracts the load tray.

Tow/Ready. The load tray is fully rearward.

To Breech. Raising the valve to this position (against a spring) advances the load tray to the breech face. A mechanical lockout must first be released (to eliminate the possibility of accidentally advancing the Load Tray when the cannoneer is behind the breech (see Continuous Firing Procedure Option, Section B3).

In doing so, the brush on the forward part of the load tray passes under the mushroom, cleaning its rearward face.

There is also a pad on the load tray pilot into the breech that cleans the gas check seat.

When the control lever is released, the load tray is retracted. As the load tray retracts, the brush cleans the forward face of the mushroom.

Rammer Stowage. Moves the Rammer from stow to ram position and back.

Tow/Ready. The ramming cylinder is in the up (stow) position.

Load. Pushing downward on the lever (against a spring) positions the Rammer on the centerline of the chamber (Load Tray must be in the Breech position first.

Ram. Two stage cylinder. Both stages are energized to ram the projectile. Reduced pressure is used to extend only one stage, which positions the propellant. The Ram Position Gage indicates when the ram-projectile and position-propellant stages are complete.

The forward end of the Ramming Cylinder Rod has a rearward facing "bristle brush" that brushes the inside diameter (and down into the Swiss notch) of the combustion chamber when the projectile is being rammed and again when the ram is retracting.

Tow/Ready. The Ramming Cylinder is fully retracted.

Projectile. By moving a "safety gate" to the side and pushing the valve upward against a spring, the Ramming Cylinder extends both stages and rams the projectile.

Propellant. Pushing the valve lever downward against a spring, the Ramming Cylinder extends only one stage, and positions the propellant a nominal 0.25 inches beyond the Swiss notch.

Lanyard. This manually pulled 13 foot cable is attached on the inside of the gimbal at a height of about 3 feet. It is routed (through an eyelet on the underside of the Ramming Cylinder mount) and semi-permanently attached to the auto primer.

Mid-Slide Controls.

Breech. This heavily detented valve hydraulically opens and closes the breech. The breech can be opened hydraulically only when at battery At any other position it must be opened with the manual handle provided with the basic issue items.

The breech can also be controlled from behind the gimbal (see Cannoneer 1's Controls section).

Tow/Control from Gimbal. In this position, the breech is closed. This is also the position the valve should be in if the breech is to be opened and closed from the gimbal.

Open. Pushing the valve upward opens the breech.

Should Cannoneer 1's Control for the Breech be accidentally moved to the Tow/Ready position while the breech is being controlled from the Mid-Slide Controls location, the breech will not close as long as the control lever is in this position.

Ready. Pushing the valve lever downward closes the breech.

- Recoil System. The Recoil System is part of the Hydraulic System. As a result, the Recoil System can be monitored and replenished with oil as required by the Gunner. However, the Recoil System has hydraulic circuit breakers that protect it from loss of oil, should a major rupture occur elsewhere in the Hydraulic System.
 - The recoil stroke can be fine tuned by adjusting the pressure in the high pressure side of the counterrecoil accumulator. The Recoil System Volume Control would provide the adjustment means, the Recoil System Pressure Gauge provides the measure.
 - Recoil Cylinders. Two identical recoil cylinders are mounted on either side of the cannon. Cavitation is minimized by connection to the Pressurized (to 200 psi) Reservoir.
 - Counterrecoil Cylinders. One is traditional (fed by a counterrecoil accumulator). The other is fed by the pressurized reservoir, and thus supplies minimal return-to-battery force. But on recoil, the oil from this second cylinder is diverted into the Energy Recovery Accumulator. As the first cylinder returns the gun to the battery position, the second is recharged with oil from the Pressurized Reservoir, in preparation for another recoil cycle.
 - Counterrecoil Accumulator. Self-displacing 2:1 (low:high pressure) provides counterrecoil pressure while minimizing cavitation in the Recoil Cylinders during recoil.
 - Counterrecoil Buffer. Twelve inch stroke shock absorber with profiled resistance to bring cannon to minimal velocity during counterrecoil.
 - Energy Recovery Accumulator. When the Energy Recovery Accumulator is full, additional fluid is bypassed to the Pressurized Reservoir via the Energy Recovery Relief Valve.

If the Energy Recovery Relief Valve fails, the nitrogen in the Energy Recovery Accumulator would go to roughly 6000 psi. The Stored Energy Pressure Gauge would indicate an excess of Stored Energy, and the Safety Relief would bleed off the excess energy (back to the Al-10 Pressurized Reservoir) at a rate suitable for the maximum rate of fire.

Pressurized Reservoir. Serves as a source of oil for the Energy Recovery Accumulator in preparation of a recoil cycle, and as a general source of fluid to the total Hydraulic System.

Equilibration System. The Equilibration System is part of the Hydraulic System. As a result, it can be monitored and replenished with oil as required by the Assistant Gunner. However, the Equilibration System has a hydraulic circuit breaker that protects it from loss of oil, should a major rupture occur elsewhere in the Hydraulic System.

In order to elevate manually (strap winch at the elevation cylinders), the BearLocs must be disabled by removing the BearLoc fittings and installing the grease zerks mounted next to them, and pressurizing with a grease gun.

Load Tray.

The Load Tray is loaded with the propellant and projectile as follows (procedure is for maximum rate of fire).

The propellant is slid into the propellant tube, and the tube is pushed over (to the left, against a spring) to make room for the projectile. The propellant tube is slightly larger than the M203 and is hinged at the bottom.

While holding the propellant tube to the left (against its spring) the projectile is slid into the load tray. Once in place, the projectile holds the propellant tube in position.

Once the load tray is advanced to the breech, the rammer is swiveled down to a position directly behind the base of the projectile.

As the projectile is rammed, disks on the side of the rammer rod hold the propellant tube in position (in the absence of the projectile).

When the rammer is fully retracted to behind the face of the propellant tube, the propellant tube is swiveled in front of the rammer by the propellant tube spring.

At that time, the rammer is energized (at a lower pressure - low enough to extend only one stage of the rammer) and the contents of the propellant tube enter the chamber. The gaps between the end-of-the-propellant-tube and the pilot-into-the-breech (on the load tray) as well as the gap between the pilot-into-the-breech and the breech-face are well under l inch with smooth transitions, to eliminate snags.

When the rammer bottoms, the propellant will be just beyond the Swiss notch, at which time the rammer is retracted.

The rammer is swiveled up, the load tray is retracted, the breech is closed, and the gun is ready to fire.

The procedure for the continuous rate of fire option is identical except the propellant is manually inserted by the cannoneer positioned near the breech.

APPENDIX B

OPERATING PROCEDURES

Disconnection from truck and emplacement

Step	Valve	Valve Position
Set manual brakes	•••••	* * * * * * * * * * * * * * * * * * * *
Disconnect electrical from truck	• • • • • • • • • • • • • • • • • • • •	
Release air lines from truck		
Release air lines (slide to dolly)		
Release out-of-battery latch		
Extend platform	Platform Position	Extend to Fire
5 Set lunette on ground	•••••	•••••
Winlatch claws	• • • • • • • • • • • • • • • • • • • •	• • • • • • • • • • • • • • • • • • • •
Swivel clavs down	• • • • • • • • • • • • • • • • • • • •	
h Unlatch trail pivot locks	• • • • • • • • • • • • • • • • • • • •	
Spread trails	• • • • • • • • • • • • • • • • • • • •	•••••
*Latch trail pivot locks	• • • • • • • • • • • • • • • • • • • •	
Set trails on ground	Elevation	Up
Turn on equilibrators	Equilibrator On/Off	On
Elevate gun	Elevation	Up
	***************************************	•••••
Remove dolly		
Lift platform (to dig in spade)	Platform	Up
Dig in spade (if penetration under 6		• • • • • • • • • • • • • • • • • • • •
Lay weapon	•••••	• • • • • • • • • • • • • • • • • • • •
Load auto primer clip	• • • • • • • • • • • • • • • • • • • •	• • • • • • • • • • • • • • • • • • • •

Go to firing procedure.

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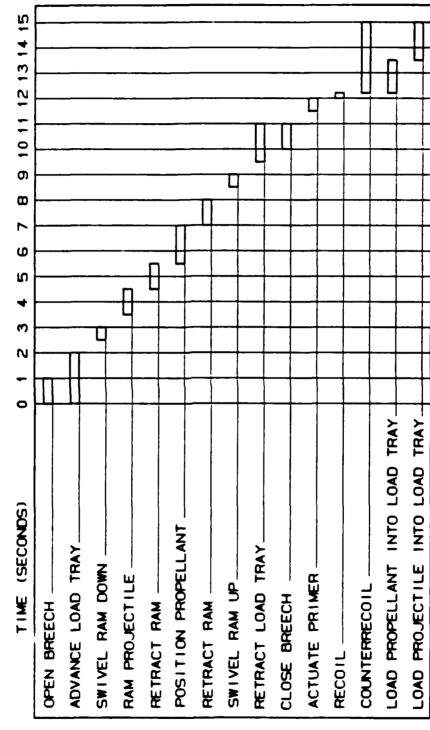
APPENDIX B2

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MAXIMUM RATE OF FIRE PROCEDURE (FOUR ROUNDS PER MINUTE)



APPENDIX B3

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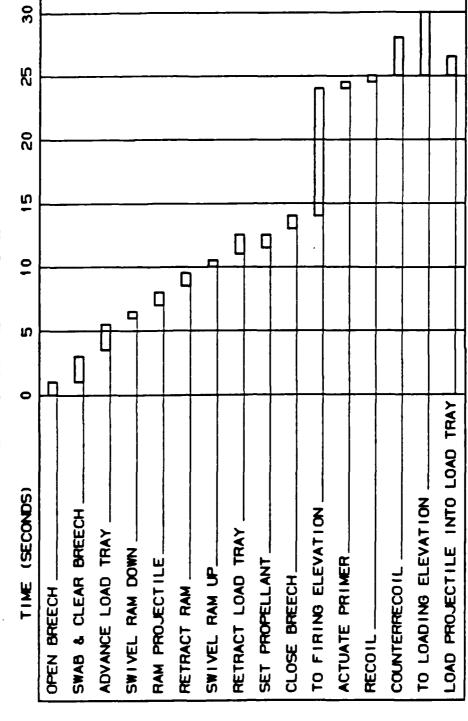
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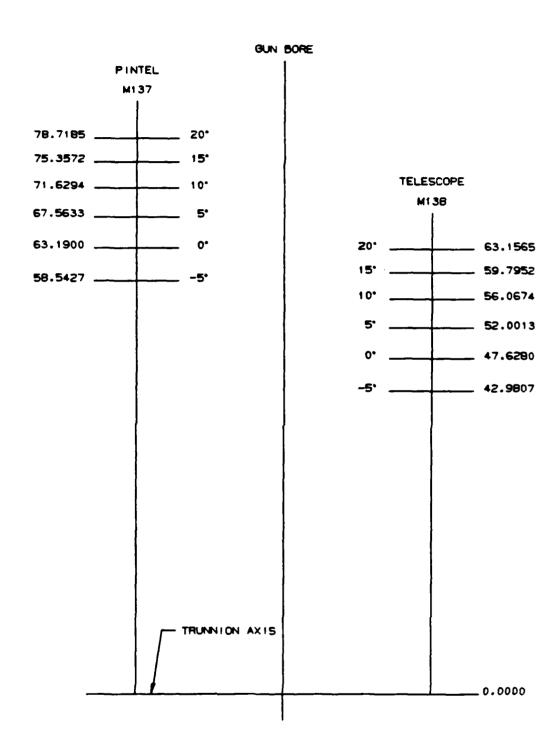
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CONTINUOUS RATE OF FIRE PROCEDURE OPTION (TWO ROUNDS PER MINUTE)



APPENDIX B4



APPENDIX C

SUPPORTING ANALYSIS AND DATA

Appendix Cl - Reliability Analysis of Equilibration Cables

1. System Description

Two pre-tensioned Kevlar-29 cables 11 feet long connect the LTHD equilibration cylinders to the upper platform. The cables have a continuous torque-balanced weave with loops at both ends which are permanently wrapped around 5.5 inch diameter pulleys. The cable diameter at the pulleys is 1.28 inches, the cable diameter between pulleys is 1.64 inches. The pulleys at the equilibration cylinder end do not rotate; the pulleys attached to the upper platform are able to rotate.

2. Static Loading

The maximum static load on the cables occurs when the recoil mass is in battery position. Maximum static load values per cable are 17,230 lbs for 0 degrees elevation and 14,880 lbs for 72 degrees elevation. The break strength of each cable is 275,680 lbs. Thus, a safety factor of 16:1 is provided for maximum static loads. If the LTHD is fired in a degraded condition with one cable, the safety factor is reduced to 8:1. Dynamic loads are discussed in the following paragraphs.

3. Dynamic Loading

The cable materials' flexibility allows it to withstand dynamic loading, including impact loading. Dynamic loading conditions include: changing gun elevation prior to firing; and during firing, which includes the impact from the rifling torque imparted to the slide assembly (which in turn, is partially imparted to the cables), and the change in cable tension resulting from a shift in supported weight CG.

As the CG of the supported weight shifts due to recoil, preliminary dynamic analysis had shown that the cable tension was reduced to a minimum of 7,008 lbs for 0 degree QE firing and 4,840 lbs for 72 degree QE firing. Thus, the cable tension is never zero. The complete dynamic analysis is shown in Appendix X of the Supplementary Information to the Dynamic Analysis Report.

A slight change in cable tension will also occur as a result of rifling torque. The barrel will impart a torque to the slide assembly. The reduced torque which reaches the equilibration cylinders will be countered by the equilibration system cables. The tension in one cable will increase slightly, the other will decrease slightly. The amount of change in cable tension will depend on the stiffness of the slide assembly.

4. Acceptance Requirements/Material Properties

Acceptability requirements for Kevlar parachute cables (rated up to 10,000 lbs/cable) are covered by MIL-C-87129A. Since Kevlar is ultra-violet sensitive, a cable covering is required. The LTHD cables will have braided nylon jackets, which can be dyed to any color. The strength of Kevlar is not affected by contact with hydraulic oils or gasoline. Contact with sea water for 12 months results in a strength decrease of 1.5%.

5. Failure Modes

The Kevlar cables will elongate noticeably before snapping.

Ultimate elongation is 3.8 to 4% of length. Should a cable snap, the Kevlar material has low snap-back properties. The most probable location for failure would be where the cable is in contact with the pulleys. To minimize failure, a pulley diameter to cable diameter ratio of 3 to 4 is recommended in the literature. The 5.125 inch pulleys provide a ratio of 4.0.

6. Preventive Maintenance

Preventive maintenance would include visual inspection for cable deterioration. For all areas, flatness, holes and cuts can be found by visually inspecting the protective nylon covering. If there is a significant increase in cable length (up to 4%), replacement will be needed.

7. Fatigue/Durability

When normal working loads are under 10% of the cable break strength, the cables are rated to last 1,000,000 cycles and when working loads are under 20%, the rated life is 100,000 cycles. A general Kevlar fatigue curve states that cable life will be 18,000 cycles (the design life) when the working loads are 29% of the break strength. (Design life is defined as 15,000 rounds fired plus 3,000 emplacement/displacements.) Static loads for the LTHD cables are at 6.25% of break strength and dynamic loads are not expected to be significantly higher. Thus, if a stress concentration factor of 2.0 at the pulleys is assumed, the safety factor (for fatigue, with stress concentration included) over the cable life will always be at least 29/(2*6.25) or 2.32.

APPENDIX C2

X

MAGNAMITE®

AS4/3501-6

GRAPHITE PREPREG TAPE AND FABRIC MODULE

JULY 1985



AEROSPACE PRODUCTS GROUP Bacchus Works • Magna, Utah 84044

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PHYSICAL CHARACTERIZATION

8

PREPREG TAPE

TAPE DESCRIPTION

Magnamite® AS4/3501-6 graphite prepreg tape is an amine-cured epoxy resin reinforced with unidirectional graphite fibers. Hercules 3501-6 resin was developed to operate in temperature environments up to 350°F (177°C).

REINFORCEMENT

Hercules Magnamite continuous, type AS4 (high strength), pan-based graphite fiber. The fiber has been surface treated to increase composite interlaminar shear and transverse tensile strength. The surface treatment is not a coating or sizing and does not leave any residue.

RESIN

N.

Hercules 3501-6 (350°F) epoxy resin system consists of:

Base epoxy: Tetraglycidyl Methylene Dianiline (TGMDA)

BIS (p-aminophenyl) Sufone (DDS) Curative: Catalyst:

Boron Trifluoride Lewis Salt

TAPE CHARACTERISTICS

	English Units	Metric Units
Fiber areal weight, graphite only	4.3 oz/yd ²	145 g/cm ²
Standard width	12 in.	30.5 cm
Approximate yield	18.8 ft/lb	12.6 m/kg
Standard resin content, % by wta	35 (± 3)	35 (± 3)
Gel time at 350°F (177°C)	6-12 minutes	6-12 minutes
Volatile content, % by wt	1 maximum	1 maximum
Out time at room temperature	10 days minimum	10 days minimum
Shelf life at 0°F (-18°C)	12 months	12 months
Approximate cured-ply thickness (62% F _V , A.W. = 145 g/m ²)	5.2 mils	0.13 mm
Neat resin density	0.0457 lb/in ³	1.265 g/cm ²

PREPREG FABRIC

FABRIC DESCRIPTION

Magnamite prepreg fabrics are a balanced weave of AS4 high-strength fiber impregnate with Hercules 3501-6 epoxy resin. Hercules 3501 is an amine-cured resin, developed to operate in temperature environments up to 350°F (177°C). The fabric has essentially balanced properties in both warp and fill directions, with warp properties typically slightly higher. A Kevlar tracer thread can be inserted in the weave for ease of determining fabric alignment.

FABRIC CHARACTERISTICS

Prepreg Designation	A193-P3501-6	A370-5H/3501-6	A370-8H3501-6
FABRIC ONLY			
Fabric style	A193-P	A370-5H	A370-8H
Fabric weave	Plain	5-harness satin	8-harness sat n
Fiber areal wt	193 g/m² (5 7 oz/yd²)	370 g/m² (10 9 oz/yd²)	370 g/m² (10 9 oz/yd²)
Construction, warp x fill	45 x 45 ± 0 2 yarns/cm (115 x 115 ± 0 5 yarns/in)	43 x 4.3 ± 0.2 yarns/cm (11 x 11 ± 0.5 yarns/in)	85 x 85 ± 0 4 yarns/cm (21.5 x 21 5 ± 1 0 yarns/in)
Yarn filament count	3000	6000	3000
PREPREG FABRIC			
Nominal cured-ply thickness (62% FV)	0 18 mm (7 0 mil)	0 34 mm (13 5 mil)	0.34 mm (13.5 mil)
Standard width	107 cm (42 in)	99 cm (39 in.)	124 cm (49 in)
Approx fabric yield (35% resin content)	3 37 m ² /kg (1 83 yd ² /lb)	1.76 m²/kg (0.95 yd²1b)	1.76 m ² /kg (0.95 yd ² /lb)
Resin content, % by wt?	32-42 (± 3)	32-42 (± 3)	32-42 (± 3)
Gei time at 350°F (177°C)	6-12 minutes	6-12 minutes	6-12 minutes
Volatiles, % by wt	1.5 maximum	1.5 maximum	1.5 maximum
Out time at room temp	10 days minimum	10 days minimum	10 days minimum
	12 months	12 months	12 months

MECHANICAL PROPERTIES

Hercules AS4/3501-6 mechanical properties listed in Table 1 and Table 2 have been determined from laminate testing data generated by two independent firms. Table 1 data were generated by Delsen Labs in Glendale, California. The data used in Table 2 were generated by McDonnell Aircraft Company of St. Louis, Missouri. The same five prepreg lots were tested by both firms, and the results normalized to 5.2 mils per ply thickness unless otherwise noted.

The A370-5H/3501-6 fabric properties listed in Table 3 were derived from McDonnell Aircraft. Company qualification data. Six prepreg lots are represented with results normalized to 14.0 mils per ply thickness.

The A and B allowable values shown are understood to be values above which a certain percentage of the population is expected to fail. The percentages for A and B allowables are 99% and 90%, respectively, with a confidence of 95%.

8

TABLE 1. DESIGN ALLOWABLE DATA FOR AS4/3501-6 UNDIRECTIONAL TAPE

	X	A	8	0	n
0°F flex strength (ksi)					
77°F	305 3	240.7	269.5	19.5	26
170°F weta	235 2	109 2	209 1	14 6	30
O°F flex modulus (msi)		1			
77°F	18.5	16.3	17 2	0 7	26
170°F weta	19.0	16 5	17 6	0 9	30
0° tensile strength (ksi)					
-67°F	301.1	217.9	252.7	26 1	25
77°F	312.7	253.2	278.2	19.2	29
250°F	290.2	208.8	242.8	25 6	25
0° tensile modulus (msi)					
-67°F	20.7	18.2	19 2	0.8	25
77°F	20 7	17.9	19.1	0.9	29
170°F weta	21.2	18.1	19.4	1.0	29
250°F	21 6	17.2	19 0	14	25
0° tensile strain (%)				1	
-67°F	1.34	1.05	1.17	0.09	25
77°F	1.40	1.10	1.20	0.08	28
170°F weta	1.30	1.00	1.10	0.08	28
250°F	1.23	0.82	0.99	0 13	25
0° compressive strength (ksi)			1		
77°F	226.3	183.9	201.5	13.0	22
170°F weta.b	158.3	107 6	128.8	15 8	24
Of compressive modulus (msi)		Ţ		1	
77°F	20 2	16.9	18.3	1.0	22
170°F weta	20 0	15.5	17.4	14	24

*

TABLE 1 (Cont)

	X	A	В	0	n
0° compressive strain, (%)					
77 ° F	1.3	1.0	1.1	0.8	22
170°F wet*	0.81	0.49	0.62	0.10	24
Short beam shearb (ksi)					
77°F	17.5	14.4	15.7	1.0	30
170°F weta	12.0	10.1	10.5	0.2	30
90° tensile strength (ksi)					
77°F	7.84	4.14	5.65	1.2	30
170° weta	3.12	2.16	2.56	0.3	35
90° tensile strength (msi)					1
77°F	1.4	1.1	1.2	0.1	
170°F	1.33	1.03	1.15	0.10	35
90° tensile strain (%)					
77°F	0.674	0.302	0.458	0.121	
170°F weta	0.242	0.146	0.204	0.022	35

y = 0.024 Poissons ratio in 90° direction.

 $[\]gamma = 0.30$ Poissons ratio in 0° direction.

a. Wet: moisture content of test specimens 1.1%; specimens conditioned in a chamber with 87% RH at 190°F

b Most of the failures were TAB failures

TABLE 2. SUMMARY OF AS4/3501-6 MECHANICAL PROPERTIES OF AVERAGES WITH A AND B ALLOWABLES

	X	A	B	0	n
Short beam shear (ksi)					
-67°F	22.7	17.0	19.3	1.6	15
77°F	18.0	14.1	15.7	1.2	23
250°F	13.2	9.7	11.2	1.1	25
250°F wet*	11.8	10.0	10.8	0.5	15
Flatwise tension (ksi)					
-67°F	2.75	0.75	1.57	0.55	14
77°F	3.48	1.90	2.56	0.50	25
250°F	2.05	1.17	1.54	0.24	14
90° tensile strength (ksi)					
-67°F	9.4	6.2	7.5	0.9	15
77°F	10.0	6.8	8.2	1.0	25
250°F	9.5	5.6	7.2	1.1	15
90° tensile modulus (%)					
-67°F	2.1	1.4	1.7	0.2	15
77°F	2.0	1.4	1.6	0.2	25
250°F	1.7	1.3	1.5	0.1	15
90° tensile strain (%)	7				1
-67°F	0.471	0.260	0.347	0.059	15
77°F	0.565	0.302	0.411	0.083	25
250°F	0.639	0.299	0.439	0.095	15
0° compressive strengthb (ksi)					
-67°F	311	201	246	30.7	15
77 ° F	292	202	240	28.3	25
250°F	240	145	184	26.4	15
O° compressive modulusb (msi)					Ţ ·
-67°F	21.2	17.6	19.1	1.0	15
77°F	1 21.1	18.9	19.8	0.7	25
250°F	20.4	16.5	18.1	1.1	15
0° compressive strain (%)					1
-67°F	1.858	0.939	1.317	0.250	15
77°F	1.752	1.064	1.351	0.216	25
250°F	1.411	0.786	1.043	0.174	15

a. Specimens were subjected to a 24-hour water boil before test. The 0° compressive and 90° tensile properties were tested using sandwich beams.

b. Type 1 material: normalized to 5.2 mils per ply thickness.

Type 2 material: normalized to 10.4 miles per ply thickness.

TABLE 3. DESIGN ALLOWABLE SUMMARY, A370-SH/3501-6 FABRIC QUALIFICATION TO MMS-554

(2) (2)

	T	T			
	X	A	В		n
Warp tensile strength (ksi)				ł	}
-65°F	94.3	54.1	70.8	12.2	20
77°F	113.3	86.8	97.9	8.6	30
250°F	128.1	89.3	105.6	12.6	30
Warp tensile modulusa (msi)					
-65°F	9.8	8.8	9.2	0.3	20
77°F	10.3	9.7	9.9	0.2	30
250°F	9.7	8.8	9.2	0.3	30
Warp tensile strain (%)				<u> </u>	
-65°F	0.96	0.61	0.76	0.12	40
77°F	1.12	0.89	0.99	0.08	59
250°F	1.28	1.05	1.15	0.08	47
Fill tension strengtha (ksi)					
77°F	99.4	72.1	83.4	8.2	20
250°F	104.1	64.5	80.9	11.9	20
Fill tension modulusa (msi)					
77°F	9.9	9.2	9.5	0.2	20
250°F	9.4	8.0	8.6	0.4	20
Fill tensile strain (%)					
77°F	1.02	0.78	0.88	0.08	40
250°F	1.11	0.79	0.92	0.11	40
Warp compression strength ^a (ksi)					
-65°F	124.8	81.8	99.7	12.9	20
77°F	109.1	91.4	98.8	5.7	29
250°F	85.1	67.8	75.1	5.6	30
Warp compression modulusa (msi)					
-65°F	9.6	8.3	8.8	0.4	20
77°F	9.8	8.8	9.2	0.3	25
250°F	9.5	8.0	8.6	0.5	30

TABLE 3 (Cont)

8

R

	x	A	8	0	n
Warp compression strain (%)					
-65°F	1.58	0.86	1.17	0.25	50
77°F	1.32	0.72	0.97	0.21	50
250°F	1.07	0.75	0.89	0.11	47
Fill compression strengtha (ksi)					
77°F	98.0	76.0	85.1	6.6	20
250°F	75.5	55.2	63.6	6.1	20
Fill compression modulus ² (msi)					
77°F	9.3	8.3	8.7	0.3	19
250°F	9.1	8.1	8.5	0.3	20
Fill compression strain (%)					
77 ° F	1.24	0.47	0.79	0.26	38
250°F	0.96	0.63	0.77	0.11	39
Warp short beam shear (ksi)					
-65°F	11.4	7.4	9.1	1.3	30
77°F	11.3	7.6	9.2	1.2	30
250°F	8.9	6.1	7.3	0.9	30
250°F wet ^b	6.5	4.6	5.4	0.6	30
Fill short beam shear (ksi)					
77°F .	11.1	8.1	9.4	0.9	20
250°F	8.6	5.9	7.0	0.8	20
Flatwise tensile strength (psi)					
-65°F	1.463	1.093	1.247	0.111	20
77°F	1.810	1.061	1.372	0.225	20
250°F	2.482	1.146	1.700	0.401	20
Tension bearing load* (lb)					
0°F	7.315	6.496	6.835	0.243	19
Tension bearing modulus					
0°F	7.6	6.0	7.0	0.3	20
Tension bearing strain (%)					
0°F	0.51	0.42	0.46	0.03	38

TABLE 3 (Cont)

	X	A	В	o	n
0/45 short beam shear (ksi)	1	İ			
77°F	9.3	6.3	7.6	0.9	20
Pin bearing load ^b (lb)					
200°F	2.877	1.958	2.339	0.276	20
0/45 compression strength a.c (msi)	1				
200°F	54.2	32.0	41.2	6.6	19
220°F	48.7	26.0	35.4	6.8	20
0/45 compression modulusa.c (msi)			1		
200°F	6.7	5.7	6.1	0.3	19
220°F	6.7	6.0	6.3	0.2	20
0/45 compression strain ^c (%)					
200°F	0.86	0.50	0.65	0.12	38
220°F	0.81	0.40	0.57	0.14	40
Warp hole tensile load (lb)					
77°F	1.597	1.442	1.506	0.046	20
Warp hole tensile modulus (msi)					
77°F	6.1	5.8	6.0	0.1	20
Warp hole tensile strain (%)	1				
77°F	0.66	0.14	0.47	0.1	20
Warp hole compression load (lb)	1				
77°F	1.903	1.627	1.741	0.388	20
Warp hole compression modulus (msi)					
77°F	5.8	4.8	5.2	0.1	4
Warp hole compression strain (%)					
77°F	0.88	0.65	0.74	0.05	8

a. Normalized values using 14.0 mils per ply thickness.

b. 24-hour water boil.

c. Specimens exposed to 160°F, 95% RH, to equilibrium.

TESTING PROCEDURES

<u>0° Tensile</u> (unidirectional)

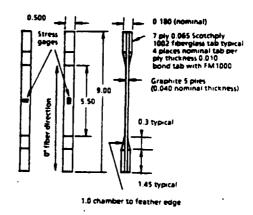
Test method Specimen thickness: ASTM D-3039 0.040 to 0.049 in.

Specimen width:

0.500 in. 5.50 in.

Gage length: Rate of test:

0.05 in/min



<u>Flexure</u>

Test method:

ASTM D-790

Specimen thickness:

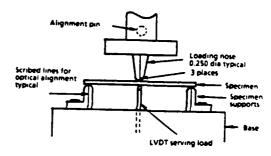
0.081 to 0.095 in. 0.5 in.

Specimen width: Span:

2.712 in.

Rate of test:

0.05 in/min



Short Beam Shear

Test method:

ASTM D-2344

Specimen thickness:

0.083 to 0.095 in.

Specimen width:

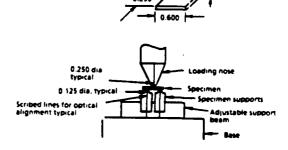
0.25 in.

Span:

0.376 in.

Rate of test:

0.05 in/min



TESTING PROCEDURES (CONT)

90° Tensile (unidirectional)

Test method: ASTM D-3039

Specimen thickness: 0.083 to 0.095 in. (Figure 1)

0.080 in. (Figure 2)

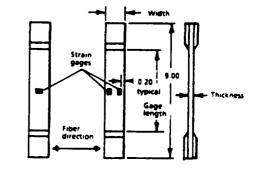
Gage length: 5.000 in. (Figure 1)

6.000 in. (Figure 2)

Width 1.000 in. (Figure 1)

0.750 in. (Figure 2)

Rate of test: 0.05 in/min



Flatwise Tensile

Test method: McDonnell Aircraft

MMS-549

Specimen thickness:

0.040 in.

Width:

1.7 in.

Length:

1.7 in.

Rate of test: 0.01 in/min

Panel t = 0.040 1.7 x 1 7 in

Panel layup 2 45, 0, 90 s

11 01010101010 0 0 0 0 0 0 0

0° Tensile (fabric)

Test method: Boeing Aircraft

BMS-212

Neck width:

0.496 in.

Nominal length:

0.496 in.

Nominal thickness:

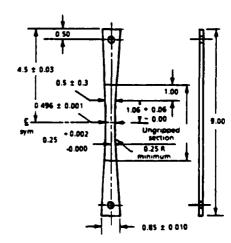
9.0 in.

NOMINAL UNICKNO

0.11 in.

Rate of test:

0.05 in/min



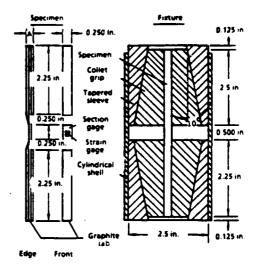
TESTING PROCEDURES (CONT)

0° Compression (Figure 1)

Test method: Specimen thickness: Gage length: Width:

Rate of test:

ASTM D-3410-75 0.097 to 0.121 in. 0.500 in. 0.250 in. 0.05 in/min



0° Compression (Figure 2)

Test method:

McDonnell Aircraft MMS-549

Span:

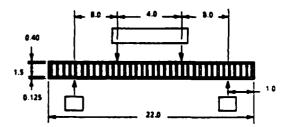
20.0 in.

Width:

1.000 in.

Rate of test:

0.25 in/min



PROCESSING

CHARACTERISTICS

**

BLEEDER REQUIREMENT

Formula for calculating bleeders to be used:

$$A = \left(\frac{R.C.}{F.C.}\right) W_f - \left(\frac{R.V.}{F.V.}\right) \left(\frac{P_r}{P_f}\right) W_f$$

where

A = amount of resin to be removed $\left(\frac{lb}{in^2 - ply}\right)$

R.C. = prepreg resin content (w/o)

F.C. = prepreg filler content (w/o)

R.V. = laminate resin volume required (v/o)

F.V. = laminate fiber volume required (v/o)

 $P_r = resin density (lb/in^3)$

 $P_t = fiber density (lb/in^3)$

 $W_f = prepreg fiber weight (lb/in²)$

When "A" has been determined, the number of bleeders required for the panel is calculated as follows:

5-mil prepreg
$$N_{15} = \frac{(A)(n)}{K} - 2 \text{ (for prepreg resin content of } 45\% \text{ max})^a$$

$$N_8 = \frac{(A)(n)}{K} - 1 \text{ (for prepreg resin content of } 45\% \text{ max})^a$$

$$N_8 = \frac{(A)(n)}{K} - 2 \text{ (for prepreg resin content of } 45\% \text{ max})^a$$

$$N_4 = \frac{(A)(n)}{K} - 1 \text{ (for prepreg resin content of } 45\% \text{ max})^a$$
where
$$N_4 = \frac{(A)(n)}{K} - 1 \text{ (for prepreg resin content of } 45\% \text{ max})^a$$

$$N_{15} = \text{number of bleeders for } 15 - \text{ply laminate}$$

$$N_{15} = \text{number of bleeders for } 15 - \text{ply laminate}$$

prepreg
$$N_8$$
 = number of bleeders for 8 - ply laminate N_8 = number of bleeders for 8 - ply laminate N_4 = number of bleeders for 4 - ply laminate

A = amount of resin to be removed
$$\left(\frac{lb}{in^2 - nlv}\right)$$

n = number of plies in laminate

 $K = 8 \times 10^{-5} lb/in^2$ for type 120 glass

 $K = 17 \times 10^{-5} lb/in^2$ for type CD 1850 Machberg paper

$$N_n = \frac{(A)(n)}{K}$$

a. For prepreg resin content above 45 w/o, the equations become:

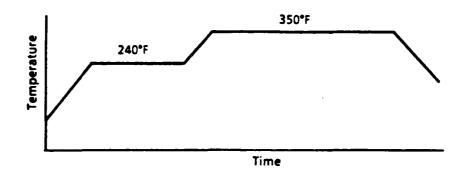
Experience has shown that a required fiber volume of 60 to 64 v/o can be consistently met if a f.V. of 59 is used in the calculation of "A". These formulae are based upon 3 x 10 in. acceptance laminates.

AS4/3501-6 RESIN

CHARACTERISTICS

CURE CYCLE FOR AS4/3501-6 TAPE & FABRIC

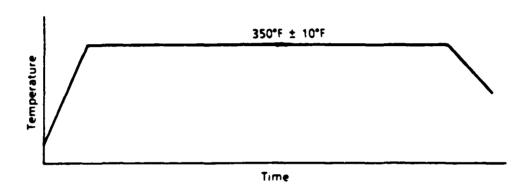
A suggested cure cycle is shown below. Other cure cycles have been used successfully with this resin system.



- 1. Place vacuum bagged layup in autoclave and close clave.
- 2. Apply minimum vacuum of 25 in. of Hg.
- 3. Apply 85 ± 5 psig.
- 4. At a rate of 3°F/minute to 5°F/minute, raise the laminate temperature to 240°F ± 10°F, while holding 85 ± 5 psig autoclave pressure and 20-29 in. of HG vacuum.
- 5. Hold at 240°F ± 10°F, 85 psig ± 5 psig and 20-29 in. of Hg for 60 to 70 minutes.
- 6. Raise pressure to 100 ± 5 psig and vent the vacuum bag to ambient atmospheric pressure.
- 7. Raise temperature at a rate of 3°F/minute to 5°F/minute to 350°F ± 10°F. Hold for 120 ± 10 minutes under 100 ± 5 psig autoclave pressure.
- 8. At a rate of 5°F ± 1°F/minute lower laminate temperature to 200°F. Release autoclave pressure.
- 9. Remove from autoclave and unbag.

POSTCURE FOR AS4/3501-6 TAPE & FABRIC

For high temperature applications, a postcure cycle is recommended to raise the laminate glass transition temperature (Tg). This procedure yields increased mechanical properties at elevated temperatures.

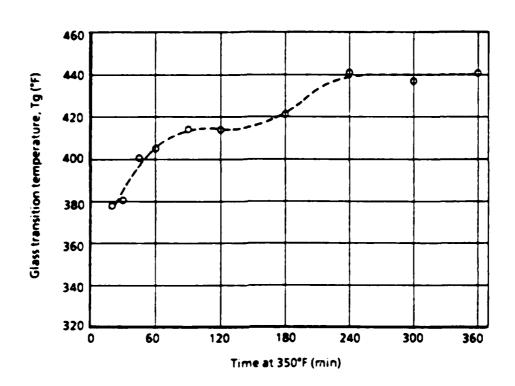


Postcure laminates as follows:

- 1. Heat to 350°F ± 10°F for 30 minutes or more
- 2. Hold for 2 to 4 hours
- 3. Cool to 200°F in a minimum of 30 minutes with oven doors closed
- 4. Remove from oven

STATE BASSES

TIME AT TEMPERATURE VS TG AND MODULUS OF COMPOSITE AS4/3501-6



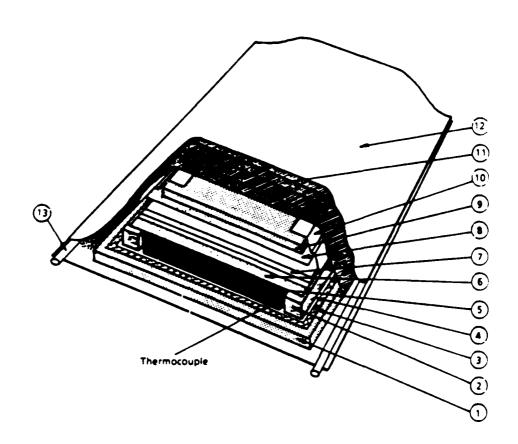
The glass transition temperature specimens were prepared as follows:

1. 20-minute sample used a 350°F press cure with no vacuum.

Branch States

- 2. Autoclave samples were taken directly to 350°F with no-hold step
- 3. 30-180 minute samples came from same panel autoclave cured for 1/2 hour at 350°F with pressure and vacuum. Specified time at 350°F was achieved by postcuring each sample at 350°F.
- 180-360 minute, samples came from same panel autoclave cured for 2 hours at 350°F with pressure and vacuum. Specified time at 350°F was achieved by postcuring each sample at 350°F.
- 5 180 minute data point is the average of the preceding two samples.

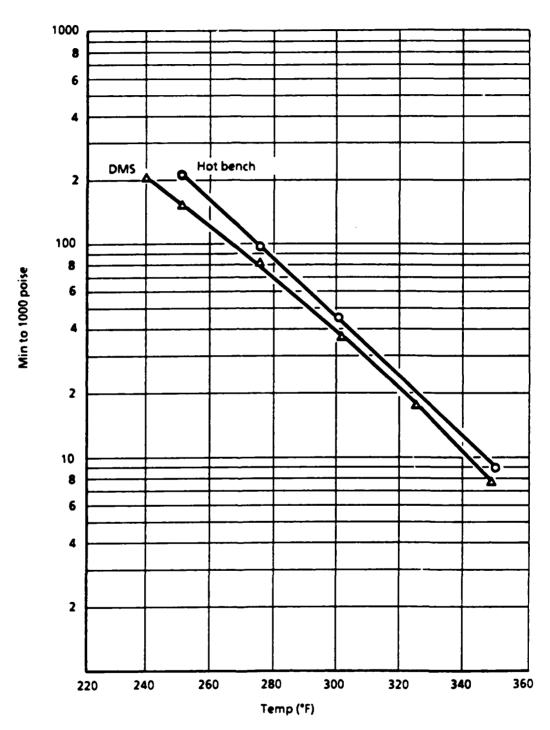
TYPICAL ACCEPTANCE PANEL FABRICATION SEQUENCE



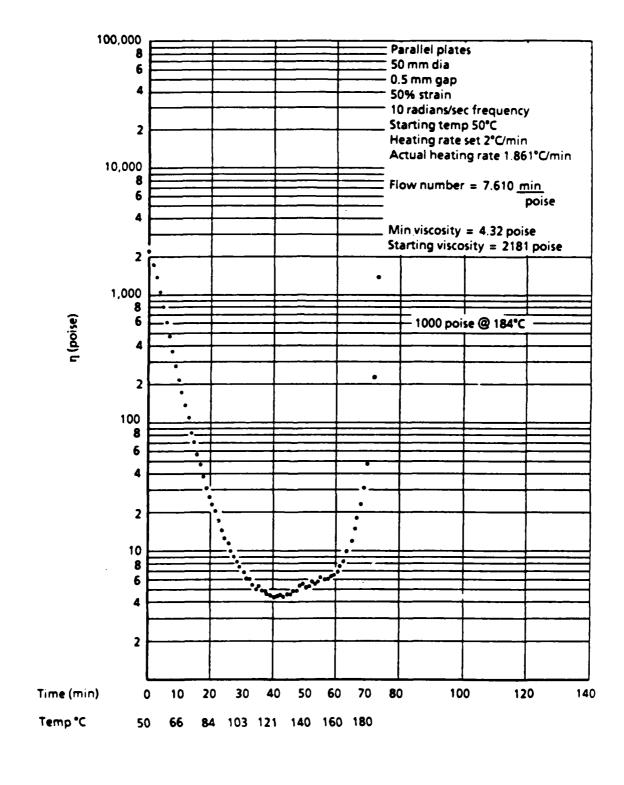
- 1. Base plate: Aluminum 1/4 to 1/2 in, thick
- 2. Cork dam: Cork 1/8 x 1 in. with pressure-sensitive adhesive backing (corprene)
- 3. Release film: Teflon, nonperforated 0.001-0.004 in. thick
- 4. Release fabric: Fabric enfab TX 10-40 release (porous)
- 5. Prepreg layup
- 6. Release fabric: Fabric enfab TX 10-40 release (porous)
- 7. Resin bleeders: Cloth, fiberglass no. 120 (prepreg to bleeder ply ratio 4:1 for 40 ± 2% resin content and 4:1.5 for 42-44° resin content (following calculation on next page)
- 8. Release film: Teflon, nonperforated 0.001-0.004 in. thick
- 9. Caul plate: Aluminum, 0.080, 0.030 minimum thickness
- 10. Tape: Pressure-sensitive, green polyester silicone
- 11. Air bleeder: 1581 style glass; 4 plies over layup
- 12. Vacuum bag: Film capron 80, hi-temp nylon, 0.002 in. thick
- 13. High-temperature sealant: Schnee Morehead

Mold release: Frekote-33 or equivalent

GEL TIME VS TEMPERATURE OF A\$4/3501-6 RESIN



DYNAMIC VISCOSITY OF AS4/3501-6 RESIN



\$ ty sidehop.bas 10 REM THIS PROGRAM CALCULATES WORST-CASE SIDE HOP 20 REM S. DACKO 30 REM 35 DIM T(50) 40 J = 1461REM NET MOMENT DATA 50 55 DATA 0 DATA 8750,13730,16000,12750 60 DATA 9750,4750,1750,-2250 70 80 DATA -3250,-4250,-10250 90 FOR I = 1 TO 10 100 READ T(I) 110 NEXT I 120 FOR I = 11 TO 50 130 T(I) = -10250140 NEXT I 145 REM 146 TI = 0150 REM START THE SIMULATION 155 PRINT "TIME,","ALPHA","V0,","THETA","HOP HT," 156 PRINT "MSEC ","R/S^2","R/S","RADS ","INCHES" 157 PRINT "----","----","----","-----","-----" 160 REM 161 T = .001 $162 \times 0 = 0$ $163 \ V0 = 0$ 164 REM 165 FOR I = 1 TO 50170 ALPHA = T(I)/J180 V = V0 + ALPHA*T190 $X = X0 + V0*T + .5*ALPHA*T^2$ 196 HOPHT = ATN(X)*88200 PRINT TI, ALPHA, VO, X, HOPHT 210 X0 = X

\$ run sidehop

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220 V0 = V 230 TI = TI+T 240 NEXT I 250 END

TIME, MSEC	ALPHA R/S ²	VO, R/S	THETA RADS	HOP HT, INCHES
0			0	0
•	•	0	• • • • • • • • • • • • • • • • • • •	· · · · · · · · · · · · · · · · · · ·
.001	5.98 905	0	.299452E-05	.263518E-03
.002	9.41136	.598905E-02	.136893E-04	.120465E-02
.003	10.9514	.154004E-01	.345654E-04	.304175E-02
.004	8.7269	.263518E-01	.652806E-04	.57447E-02
.005	6.67351	.350787E-01	.103696E-03	.912526E-02
.006	3.2512	.417522E-01	.147074E-03	.129425E-01
.007	1.19781	.450034E-01	.192676E-03	.169555E-01
.008	-1.54004	.462012E-01	.238107E-03	.209535E-01
.009	-2.2245	.446612E-01	.281656E-03	.247858E-01
.01	-7.01574	.424367E-01	.320585E-03	.282115E-01
.011	-7.01574	.354209E-01	.352498E-03	.310198E-01
.012	-7.01574	.284052E-01	.377396E-03	.332108E-01
.013	-7.01574	.213895E-01	.395277E-03	.347844E-01
.014	-7.01574	.143737E-01	.406143E-03	.357406E-01
.015	-7.01574	.735798E-02	.409993E-03	.360794E-01

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.016
              -7.01574
                               .342232E-03
                                              .406827E-03
                                                              .358008E-01
.017
              -7.01574
                              -.667351E-02
                                              .396646E-03
                                                              .349049E-01
.018
              -7.01574
                              -.136893E-01
                                              .379449E-03
                                                              .333915E-01
                                                              .312608E-01
.019
              -7.01574
                              -.020705
                                              .355236E-03
                              -.277207E-01
.02
              -7.01574
                                              .324007E-03
                                                              .285127E-01
              -7.01574
                              -.347365E-01
.021
                                              .285763E-03
                                                              .251471E-01
.022
              -7.01574
                              -.417522E-01
                                              .240503E-03
                                                              .211643E-01
.023
              -7.01574
                              -.048768
                                              .188227E-03
                                                              .016564
.024
                              -.557837E-01
                                              .128936E-03
              -7.01574
                                                              .113463E-01
.025
                              -.627995E-01
              -7.01574
                                              .626282E-04
                                                              .551128E-02
.026
              -7.01574
                              -.698152E-01
                                             -.106949E-04
                                                            -.941148E-03
.027
              -7.01574
                              -.768309E-01
                                             -.910337E-04
                                                            -.801096E-02
.028
              -7.01574
                              -.838467E-01
                                             -.178388E-03
                                                            -.156982E-01
.029
              -7.01574
                              -.908624E-01
                                             -.272759E-03
                                                            -.240028E-01
.03
                              -.978782E-01
                                                            -.329247E-01
              -7.01574
                                             -.374145E-03
.031
              -7.01574
                              -.104894
                                             -.482546E-03
                                                            -.424641E-01
              -7.01574
                              -.11191
                                             -.597964E-03
                                                            -.526208E-01
.032
                              -.118925.
.033
              -7.01574
                                             -.720397E-03
                                                            -.633949E-01
.034
              -7.01574
                              -.125941
                                                            -.747865E-01
                                             -.849846E-03
.035
              -7.01574
                              -.132957
                                             -.986311E-03
                                                            -.867953E-01
.036
              -7.01574
                              -.139973
                                             -.112979E-02
                                                            -.994216E-01
.037
              -7.01574
                              -.146988
                                             -.128029E-02
                                                            -.112665
.038
              -7.01574
                              -.154004
                                             -.14378E-02
                                                            -.126526
.039
              -7.01574
                              -.16102
                                             -.160233E-02
                                                            -.141005
              -7.01574
                              -.168036
                                             -.177387E-02
                                                            -.1561
.04
                                                            -.171814
.041
                              -.175051
              -7.01574
                                             -.195243E-02
.042
              -7.01574
                              -.182067
                                             -.213801E-02
                                                            -.188144
.043
              -7.01574
                              -.189083
                                             -.23306E-02
                                                            -.205092
                              -.196099
.044
              -7.01574
                                             -.25302E-02
                                                             -.222657
              -7.01574
                              -.203114
                                             -.273682E-02
                                                            -.24084
.045
              -7.01574
                              -.21013
                                             -.295046E-02
                                                            -.25964
.046
                                             -.317112E-02
.047
              -7.01574
                              -.217146
                                                            -.279057
                                             -.339879E-02
                                                            -.299092
.048
              -7.01574
                              -.224162
.049
              -7.01574
                              -.231177
                                             -.363347E-02
                                                            -.319744
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* CALCULATED WORST-CASE SIDE HOP FROM RIFLING TORQUE
15 0.036 INCHES.

DESCRIPTION: OPERATIONAL PROCEDURES

ETATUS: The complete set of operational procedures contained in this section reflect the current configuration and are not e pested to change.

Some of the procedures may require minor re-wording, different terminology or clean illustrations, however.

AUTHOR: Scott Dacko

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CAPT COLORGE ANGEORIES PARAMONE CONSTANT

Operational Procedures

200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 | 200 |

A complete summary of LTHD operational procedures can be found in the TDF. Dwgs. 12585710-825, pp. 1-26. Illustrations, timelines and step-by-step procedures in the form of "who, what, how, where, when, how long" have been created for emplacement, maximum rate of fire, sustained rate of fire, misfire, speedshift and displacement operations. Except for some terminology and precise loading control positions, the procedures are up-to-date.

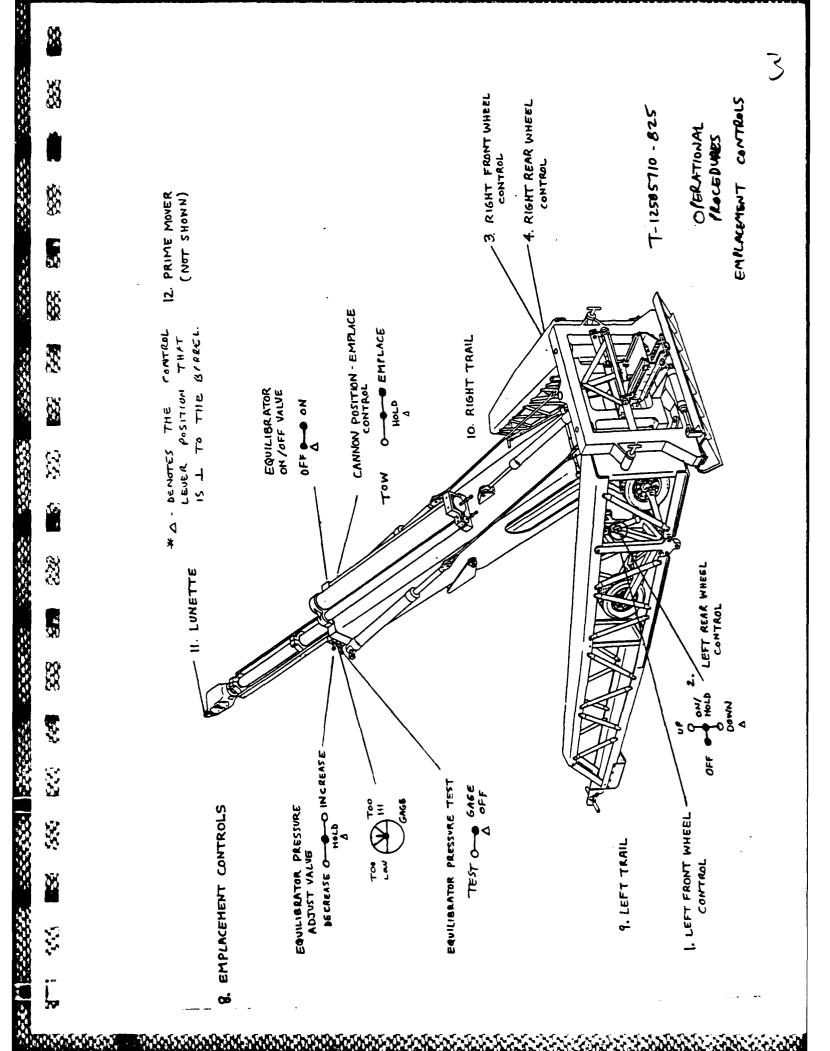
These procedures—and the complete LTHD—numan interface— has been developed with input and feedback from human factors, safet, and other systems engineers at FMC. Members of the Fort Sill user community along with ARDEC personnel have also provided constructive input to the procedures.

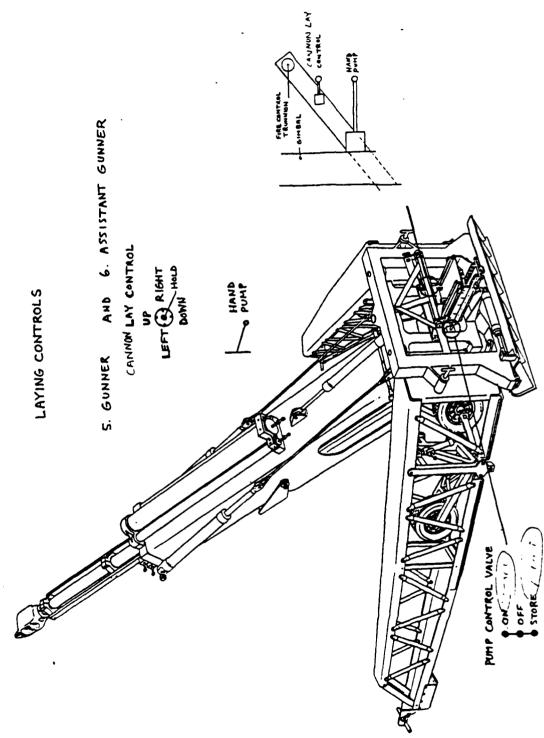
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Operational Design Considerations

The number of areas in which operational considerations played an important role in various aspects of the LTHD design are far too numerous to list. However, some of the considerations which had a significant impact on the design, along with the approaches taken, are discussed below.

- . Holdnette that a light enough for two 95th percentile men to lift off the truck pintle during emplacement and onto the pintle during displacement. The walking beam dimensions position the front wheels to allow a roughly 100 lb. lunette weight.
- . Sufficient space for cannoneer 1 at all QE's and traverse angles. The trails are spread 35 degrees from center to allow norm for a 95th percentile cannoneer 1 wearing a cold-weather. We suit.
- . Inading the bag charge into the breech at all loading OE's and traverse angles. A maximum loading QE of 600 mils provides the upper limit for a cannoneer with 95th percentile reach.
- . Operating the fire control equipment, handpumps and tube lay controls at all DE's and traverse angles. Proper positioning and preamances provide the acceptable reaches and forces to be exerted for the 95th percentile man.





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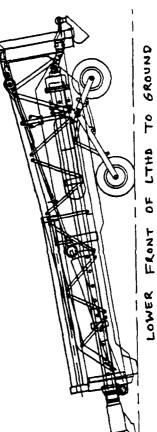
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FMC LTHD - EMPLACEMENT PROCEDURE - ILLUSTRATED



TOW POSITION

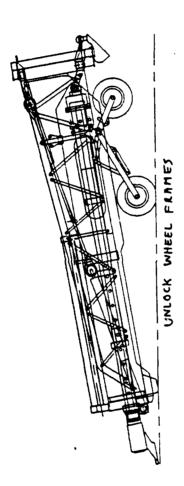


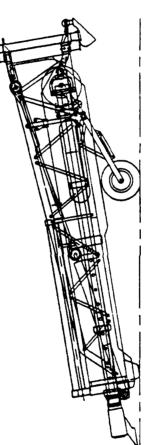
EMPLACEMENT PROCEDUKE -ILLUSTRATED

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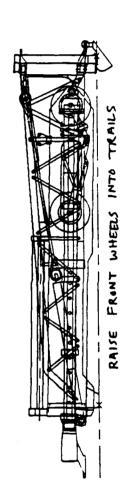
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EMALEMENT PROCEDURE - ILLUSTRATED - CONT'O.





RAISE REAR WHEELS INTO TRAILS



ENFLACENENT PROCEDURE -ILLUSTRATED

FHRACEHENT PROCEDURE - ILLUSTRATER - CONTID.

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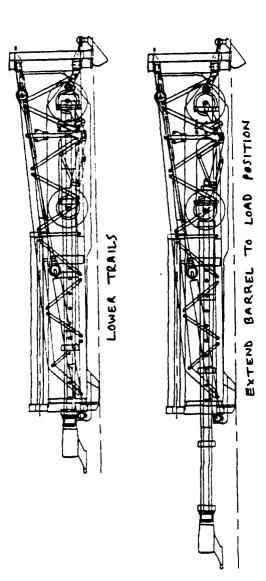
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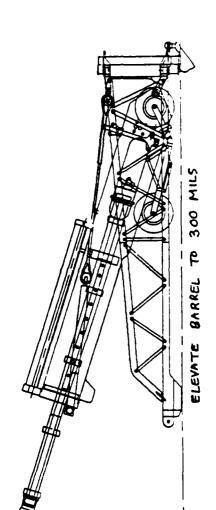
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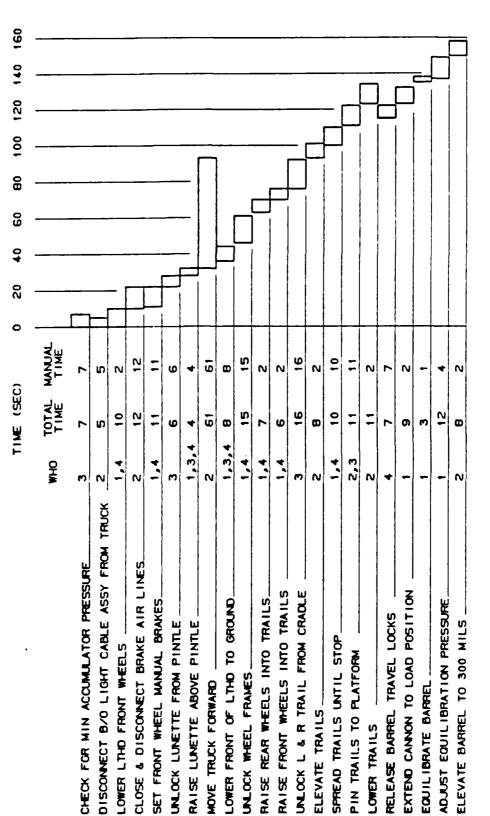
NOT SHOWN - ELEVATE TRAILS
SPREAD TRAILS
PIN TRAILS TO PLATFORM





FINACENENT FLOCEDURE -

FMC LTHD EMPLACEMENT TIMELINE



EMPLACEMENT

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FAC LTHD - EMPLACEMENT PROCEDURE

OH:	MHAT	HOM	WHERE	KHEK	#0# -
m	CHECK FOR MIN. ACCUMULATOR VOLUME	LOOK THRU HOLE ON L,R SIDE OF CRADLE AT ACCUMULATOR VOLUME INDICATOR. ADD UP TOTAL NO. OF NOTCHES SHOWING. IF TOTAL NO. OF NOTCHES GREATER THAN :	CRADLE - L,R 51DE	•	9 8 8
		MAN, MOVE PUMP CONTROL VALVE FR DFF TO ON. IF TOTAL NO. OF NOTCHES LESS THAN .: A. MAN, MOVE PUMP CONTROL VALVE FROM OFF TO STORE. B. MAN, PIISH ON MAND-PIMP HANDLES	د	<u>:</u>	61
		ABOVE MIN. PRESSURE. C. READ ACCUM. PRESSURE GAGE. IF ARRUE MIN. DRESCURE GAGE.			
		O. MAN. HOVE PUMP CATRL VALVE FROM STORE TO DN.			
2	DISC. BLACKOUT LIGHT CABLE ASSY FROM PRIME MOVER	A. MAN. DISCNCT P.M. B/O LT CABLE ASSY. B. MAN. CONNECT ASSY TO DUMMY COUPLING.	0 0	2	2 5
•••	LOWER LTHD FRONT WHEELS	COMCURRENT ON LEFT AND RIGHT: A. MAN. MOVE FRI WHL CNTRL FR OFF THRU ON/HOLD TO DOWN UNTIL REAR WHLS ARE OFF GRAIMED ARRITY & INCHES	1,3	0	•
		B. FRT WHLS LOWER TO LIFT REAR WHLS OFF GND. C. MAN. REL. FRT WHL CONTROL. RET'S TO CN/HOLD.	1,3	- 6	æ
7	CLOSE & DISC. BRAKE AIR LINES	A. MAN. CLOSE SERVICE AIR LINE C/O COCK. B. MAN. CLOSE ENERGENCY AIR LINE CUTOUT COCK. C. MAN. UNCOUPLE SERVICE AIR LINE COUPLING.	. 12	2 2 2	7 7 7
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OH M	WHAT	HOW	WHERE	RHEN	HOM
-	SET FRONT WHEEL MANUAL BRAKES	CONCURRENT ON LEFT AND RIGHT: A. MAN. TAKE BRAKE HANDLES FR CLAMPED POS & PLACE IN SOCKET	1,3	=	-
		B. MAN. PUSH DOWN ON HANDLE UNTIL STOPS. C. MAN. REPLACE HANDLE TO CLAMPED POS.	1,3	51 81	m a
~	UNLOCK LUNETTE FROM PINTLE	A. MAN. REMOVE PINTLE COTTER PIN. B. MAN. RELEASE TOWING PINTLE LATCH.	21	22 26	4 2
1,3,	1,3,4 RAISE LUNETTE AROVE PINTLE	CONCURRENT ON LEFT AND RIGHT: MAN. HOLD HANDLES AT TRAIL ENDS, LIFTING VERT. TO RAISE LUNETTE UNTIL ABOVE PINTLE.	9,10	28	-
7	MOVE IRUCK FORWARD	DRIVE TRUCK UNTIL CLEAR OF LTHD.	13	22	19
1,3,	1,3,4 LOWER FRONT OF LTHD TO GROUND	CONCURRENT ON LEFT AND RIGHT: MAN. HOLD HANDLES AT TRAIL ENDS, LOWER FRT OF LTHD TO GROUND.	9,10	36	85
*	UNLOCK WHEEL FRAMES	CONCURRENT ON LEFT AND RIGHT: A. PULL OUTWARD ON FRAME HANDLE. B. MAN. SWIVEL FRAME DOWN AGAINST FRT FRAME. C. RELEASE FRAME HANDLE.	2,4 2,4 1,3	55 56 56	טטט
* .	RAISE REAR WHEELS INTO TRAILS	CONCURRENT ON LEFT AND RIGHT: A. MAN. MOVE REAR WHL CONTROL FR OFF THRU ON/HOLD TO UP UNTIL RR WHLS RAISE & STOP. B. REAR WHLS RAISE INTO TRAILS AND STOP. C. MAN. REL. REAR WHL CONTROL. RET'S TO ON/HOLD.	7 2,4	63 64 64	40 KD
₹ :	RAISE FRONT WHEELS INTO TRAILS, (REAR OF LTHD LOWERS TO GROUND)	CONCURRENT ON LEFT AND RIGHT: A. MAN. MOVE FRT WHL CONTROL FR ON/HOLD TO 1,3 UP UNTIL FRT WHLS RAISE INTO TRAILS. B. FRT WHLS RAISE INTO TRAILS, REAR OF LTHD LOWERS TO GROUND. C. MAN. REL. FRT WHL CNTRL. RET'S TO ON/HOLD. 1,3	<u> </u>	70 1.7 2.5	N 4

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FREENRE

EMPLYCENE NIT PROCEDURE - CONT'S

								EMPLACE MENT PROCEDURE
HON	1 3 3	7 6	10	100	9 6	2 2	80 ~ =	2
NHEN	76 77 79 81	93	100	111	123 124 133	115	123	135 136
WHERE	6666	5/6	9,10	5,6 5,6	5/6	& &	صا <u>ح</u>	a !
NON	A. MAN. PUSH ON LEFT PLUNGER PIN WITH THUMB. B. MAN. PULL & REMOVE PLUNGER PIN. C. MAN. SWIVEL TRAIL LOCK OUT & UP. D. MAN. REPLACE PLUNGER PIN IN HOLE. E. REPEAT ON RIGHT SIDE OF CRADLE.	A. MAN. MOVE CANNON LAY CNTRL FROM HOLD TO TO DOWN UNTIL TRAILS RAISE AND STOP. B. TRAILS RAISE AND STOP. C. MAN. REL CANNON LAY CNTRL,RET'S TO HOLD.	CONCURRENT ON LEFT AND RIGHT: Man. Push Horiz & Perp to trails at ends & Walk until trails hit stops.	CONCURRENT ON LEFT & RIGHT: A. MAN. PUSH ON SPRING-LOADED BOLT. B. MAN. ROTATE BOLT T-HEAD CM UNTIL TIGHT.	A. MAN. MOVE CANNON LAY CNTRL FR HOLD TO UP UNTIL TRAIL PADS PUSH ON GROUND. B. TRAILS LOWER, PADS PUSH ON GROUND. C. MAN. REL CARNON LAY CNTRL, RET'S TO HOLD.	A. MAN. PULL OUTWARD ON LEFT TRAVEL LOCK LEVER UNTIL STOPS. B. MAN. PULL OUTWARD ON RIGHT TRAVEL LOCK LEVER UNTIL STOPS.	A. MAN, MOVE CANNON POS-EMPLACE CNTRL FR HOLD TO EMPLACE UNTIL BARREL STOPS. B. BARREL MOVES FORWARD, BARREL STOPS. C. MAN. REL. CANNON POS-EMPLACE LEVER.	A. MAN, MOVE EQUIL ON/OFF VALVE FR OFF TO ON. B. Barrel Equilibrates.
WHAT	UNLOCK L & R TRAIL FROM CRADLE	ELEVATE TRAILS	SPREAD TRAILS UNTIL TRAILS STOP (Trail angle 15 35 DEG)	LOCK TRAILS TO PLATFORM	LOWER TRAILS	RELEASE BARREL TRAVEL LOCKS	EXTEND CANNON TO LOAD POSITION	EQUILIBRATE BARREL
OHM	m	~	* :	2,3	2	₩		-

ENPLACUILAT PROCEDURE - CONT'D

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NH.	WHAT	MON	WHERE	MHEN	HON
-	ADJUST EQUILIBRATION PRESSURE	8. MAN. MOVE EQUIL PRESSURE TEST LEVER TO TEST AND READ EQUIL PRESSURE GAGE. TE FOLLI PRESCUIRE RACE DIT DE RANGE.	æ	137	2
		C. MAN. NOVE EQUIL PRESS ADJ VALVE FR HOLD TO B DECR UNTIL INDIC IS IN ACCEPT RANGE, OR: MOVE EQUIL PRESS ADJ VALVE FR HOLD TO INDER, MAN. PUMP UNTIL INDIC IS IN ACCEPT RANGE.	8 .	1	7
		E S	1	142	9
		E. MAN. MOVE EDUIL PRESS ADJ VALVE FROM INCR/DECR TO HOLD.	&	148	_
₩	SET ACCUMULATOR VOLUME DIAL	A. LOOK THRU HOLE ON L,R OF CRADLE AT VOL. INDICATOR. ADD UP TOTAL NO. OF NOTCHES SHOMING.	CRADLE- L,R SIDE	149	•
4-		B. ROTATE ACCUMULATOR VOLUME DIAL UNTIL NO. OF NOTCHES ON DIAL LINES UP WITH PRESSURE GAGE NEEDLE.	5	155	2
2	ELEVATE BARREL TO 300 MILS	A. MAN. NOVE CANNON LAY CNTRL FR HOLD TO UP UNTIL APPROX 300 MILS REACHED.	9/8	157	7
		B. BARREL MOVES TO APPROX 300 MILS.	;	158	9
		C. MAH. REL CANNON LAY CNTRL. RET'S TO HOLD. 5/6	2/6	164	_
		10	TOTAL TIME:	SGWOD3S S91	SONC

DIGGING TRENCH FOR SPADE:

IF GROUND IS NOT EXTREMELY HARD, SPADE WILL PENETRATE GROUND AND THERE IS NO NEED TO DIG HOLE. Ground pressure at lind spade point is approx. 5% Higher than Migb before barrel is extended during EMPLACEMENT. IF GROUND IS EXTREMELY HARD AND SPADE DOES NOT PENETRATE ADEQUATELY EVEN AFTER BARREL IS ELEVATED TO 72 DEG OE, BEGIN SPEEDSHIFT PROCEDURE, AND DIG TRENCH FOR SPADE BEFORE COMPLETING THE FINAL SPEEDSHIFT ROTATION.

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TIME	TOTAL	TIKE	. 22	1.78	1.0	2.9	4.8	3.5	2.5	1.5	1.9	1.9	0.8	15.0	15.0
		品	}	!		-	-			-				m	2
				C'RECOIL TO LOAD POSITION	DPEN BREECH	SHAB CHAMBER	RAM AND RETRACT	LIFT CHARGE TO WINDOW	POSITION CHARGE IN BREECH	CLOSE BREECH	MOVE BARREL INTO BATTERY	CYCLE PRIMER AUTOLOADER	FIRE PROJECTILE	STAGE PROJECTILE	STAGE BAG CHARGE

MAX PATE OF FIRE TIMELINE

13

FHO LIND - LONDING PROCEDURE, MAN RATE OF FIRE

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WHEN FIRING BELOW 600 MILS: WHO WHAT	H11.5:	#64	13 13 13 13	HE HE	75. 12.
			i	Misas	980T
≝ :₌	HERITZER RECEILE TO 8,75 FT MAX. (8,5 FT WY HET JONE 85)	PSGJESTILE FIFES, RECOIL MESH ASTIVE.	:	· ±	5.00
<u> </u>	HOWITTER COUNTER-RECOILS, BARKEL STORS AT LOWE POSITION	COUNTER-RECOIL CYL ASTIVE.	<u>.</u>	30 C	1
adena kasawa Kaba	Žiei.	A, MAK, BRESCH BLOCK CMTRE LEMEN SK CLOSE TO GESM, ARLEASS LEWER,	73	ei ei	tro et
		9. ESETH BLOCK DEDIS 9 STOPS.	;	u .	ن چ:
		A. NAK. INSEPI SWAR, RERR END FIRST. THPL CRAPLE WINFOR.	a	3	ا با خ
		9. MAN, INSERT SWAD INTO CHANDER, RENOVE. C. REMOVE SWAD DA CRAM PRINCOL	:: :	un u rol sa	<: • •
		D. WIFE DRIEBATES STREET ASSY WY SWAP.	: ::	i io	, un
LAJ LAJ	SAM PSGJECTICE & TPAY, RETAALT ISAY	A, MAK, MOVE RAM CNIRL LEVR FR RETFACT TO PAM & RETRACT, SEL LEVR, PATYS TO RETPACT.	ξ;		د <u>با</u> ت
				95 0 45 0	: u
		9. LOAD TRAY AUTO. RETRACTE, STORE AT PLATEM	; ;	, e.	n un
다. 6.	HET CARRES, POSTTROM IN SOCION	4, MAY, LIFT BAS CHARGE FROM TRAIL TO CRADLE. E. MAK, LIFT CHARGE THRO CFASLE MINGON A ELGGE AREAD OF SWISS NOTCH IN BREECH.		m = 1	us us Li ei
מוססת המפשעה הייטות	20	A. MAN. MOVE BREECH BLOCK CHTRL LEVER FR DPCH TA CHOSE BRIEGES FROM	5	13.8	<u>ت</u> ن
		E. PRESCH BLOCK SHIMES CLOSED.	: 5	f : 0;	67 67 65 €

MAX RATE OF FIRM PROCEDURES

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LOADING PROCEDURE, MAX FATE OF FIFE - CONT'D

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a la	= = :	ದ : ಕ	2011 631 Ma Mari	
6. MAN, MOVE CARRON POS LOAD CATAL LEUP FP HOLD TO BATTERN LMITL STOPS IN DATTER. 8. BARBEL HOVES TO BUTTERY FOR & STORS. C. MAN, REL CAMPON FOR LOAD CUTTL LEVER. RETURNS TO HOLD.	A. MAN. MONE PRIMER CHTRL LENES FE PETME TO ESTABLIS. 9. SCENT POINER IS ENTRACTED. 1. MAN. REL. PRIMER (NITAL LENER. CETTS 12 PKIME.) 12 PKIME. 13 PKIMER IS INCERTED.	A. CH FIRE COMMAND, MAN. MOUS LANYARD COUTROL CO LEDGO FO GEDOY TO FISC. MRET S TO PEDDA. B. PIRING PIN STRINGS PRIMED.	A. PLACE PROJECTILE 19 TRAY. E. MAN. CARFY PROJECTILE TO TOAY.	MAN, TAPRY BAS THRESE TO THAIL.
MOVE PASREL LYTE PATTERY	CYCLE SPIMES AUTOLOGICS	FIRE FROM TILE	STASE PROCESSUE	303eh2 Se6 35e12
				rı

FIRING RETHEEN COO AND BOO MILES

LOWER BREEL DUBLIS TIME OF COUNTER-RECOIL TO LOBD FOSTITOY (1,78 EEC.) ELEVATE BASECL & ACTUST FIRE CONTROL DUSINGS TIME DAGREL MOVES INTO PATTERY (1,9 SECY + TIME FRIMER FUTCLORDER IS CYCLED (1,8 SECY + 1,5 ADDITIONAL SEC, TOTAL CYCLE TIME = 29.7 SEC. MAX RATE OF FIRE proceditas

RATE OF FIRE

(FILE: SUSTF)

FMC LTHD - LOADING PROCEDURE, SUSTAINED FIRING

Control Control

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								SUSTAINED RATE PROCEDURES
MERE	;	1		5	5 555	5	5	S.
мон	PROJECTILE FIRES, RECOIL MECH ACTIVE.	COUNTER-RECOIL CYL ACTIVE.	LOOK AT ACCUMULATOR PRESSURE BAGE. CHECK IF NEEDLE IS ABOVE "LOAD AND FIRE" MARK ON VOLUME DIAL. IF ABOVE, CONTINUE. IF RELOW, PUMP WITH HAND-PUMP.	A. MAN. MOVE BREECH BLOCK CONTROL LEVER FR Close to open, release lever. B. breech block opens & Stops.	A. MAN. INSERT SWAB, REAR END FIRST, THKU CRADLE WINDOW. B. MAN. INSERT SWAB IN CHAMBER, REMOVE. C. REMOVE SWAB FR CRADLE WINDOW. D. WIPE OBTURATOR SPINDLE ASSY W/ SWAB.	A. LOOK THROUGH TUBE. B. CALL "BORE CLEAR."	NAN. REMOVE PRIMER DRUM, REPLACE. NAM. CLEAN PRIMER VENT WITH VENT- CLEANING TOOL, REANING TOOL OR PRIMER CHAMBER CLEANING BRUSH.	A. TURN AZIMUTH KNOB UNTIL DEFLECTION APPEARS IN DEFLECTION COUNTER, B. READ SETTING TO CHIEF OF SECTION.
WHAT	HONITZER RECOILS TO 8.75 FT MAX. (8.5 FT W/ HOT 20NE 85)	HOWITZER COUNTER-RECOILS, BARREL STOPS AT LOAD POSITION	CHECK FOR MIN ACCUMULATOR PRESSURE	OPEN BREECH BLOCK	SWAB CHAMBER	CHECK FOR CLEAR BORE	REPLACE PRIMER DRUM EVERY 20 ROUNDS CLEAN PRIMER VENT EVERY 20 ROUNDS	ENTER DEFLECTION DN M171 TELESCOPE & QUADRANT MOUNT

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			SS LEVEL VI NOB TO ADJ. CH LEVEL VI MOB TO ADJ. CT COUNTER.	M171. FR HOLD TI ORTATVED. S ON LEVEL	NOB UNTIL 10n countei Ection.	SS LEVEL V.	V LEVEL VII FR LOCK TI LEVEL VIAL.	R RETRACT. O RETRACT. RAY STOPS.	AIL TO BREI Le Window (Eech.	RL LEVER FI	
Š			BLE IN CRO. CONTROL KI PLE IN PITI CONTROL KI CONTROL KI ELEV CORREG	EPIECE OF L LAY CNTRL HT PICI IS RED BURBLE!	CONTROL KI IN ELEVAT CHIEF OF SI	PLE IN CRO. CONTROL KI	PLE IN ELE LAY CNTRL T IN ELEV (ITRL LEVR F. DVANCES, TIETRACIS, S.	ARGE FR TR Thru cradi otch in Bri	I BLDCK CN7 Rel. Lever NGS CLOSED MARKS.	
×	ZINCD		R CENT BUR ROSS LEVEL R CENT BUB 17CH LEVEL R 7ERO 1N	THROUGH EY OVE CANNON PROPER S16 FOR CENTE	ELEVATION NT RPPEARS ETTING TO	IR CENT BUB Ross Level	IR CENT BUB IOVE CANNON BUBBLE CEN	IOVE RAM CN REL. LEVER RAY/PROJ A TILE RAMS. RAY AUTO R	IFT CHARGE OF SWISS N	IDVE BREECH O CLOSE. I BLOCK SWI E WITNESS	
3		MOH	A. CHK FU B. TURN CI C. CHK FO D. TURN P. E. CHK FO F. TURN EI	A. SIGHT B. MAN. M. UNTIL I	A. ROTATE PUADRAI B. READ SI	A. CHK FOI B. TURN CI	A. CHK FO B. MAN. M UNTIL	A. MAN. M RAK. 1 B. LOAD T C. PROJEC D. LOAD T	A. MAN CA B. MAN. L AMEAD	A. MAN. N OPEN 1 B. BREECH C. OBSERV	
•			11 EL		NTROL	6 0		T TRAY			
	1300JC		EVEL OF M1'	COLL INATOR.	18 FIRE CO	EVEL OF MI	QUADRANT	AY, RETRAC	RREECH		
7			EL & ADJ LI T mount	& LAY ON - AKES	DRANT ON N	EL & ADJ L	ON TUBE TO LEV LEVEL	CTILE & TR	CHARGE IN	ECH PLOCK	
)	TOVE	WHAT	CHECK LEVI & QUADRAN'	CHECK LAY Aining Sti	ENTER DUAL Quadrant	CHECK LEY	NOVE CANNO 4 CHECK EI	RAM PROJE	P0S1110M	CLOSE BRE	
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075											
		LOADING PROCEDURE - SUSTAINED FIRING - CONT'D	I SOR THE SOLVE DUKE - SUSTAINED FIRTHER THERE	LOADING PROCEDURE - SUSTAINED FIRITGE CONT D WHAT WHAT CHECK LEVEL 1 ADJ LEVEL OF MIJI TEL A. CHE FOR CENT BURBLE IN CROSS LEVEL VIAL. P. QUADRANT MOUNT C. CHE FOR CENT BURBLE IN PITCH LEVEL CONTROL. KNOB TO ADJ. E. CHE FOR TERO IN ELEV CONTROL. F. THAN ELEV CONTROL. KNOB TO LEND COUNTER. F. THAN ELEV CORRECT COUNTER. F. THAN ELEV CORRECT KNOB TO LEND COUNTER.	WHAT WHAT WHAT WHAT WHAT WHERE CHECK LEVEL A BOJ LEVEL OF NITH TELL AND THE WHERE IN PROSS LEVEL VIAL. B. THUN CHEST EVEL UP NITH TELL CHANGE KIND TO ADJ. C. CHK FOR CENT BURNER IN PROTECT VINDE. C. CHK FOR CENT BURNER IN PICH LEVEL CONNECL. F. THUN CHEST FOR CENT THROUGH EVEPTECE OF NITH. C. CHK FOR TENT THROUGH EVEPTECE OF NITH. F. THUN CHEST KIND THROUGH EVEPTECE OF NITH. C. CHK FOR TENT THROUGH EVEPTECE OF NITH. F. THUN CHEST CONNECL. F. THUN CHEST KIND THROUGH EVEPTECE OF NITH. F. THUN CHEST KINDE OF NITH THROUGH EVEPTECE OF NITH. F. THUN CHEST KINDE OF NITH THROUGH EVEPTECE OF NITH. F. THUN CHEST KINDE OF	LOADING PRICEDUKE - SUSTAINLE FIRITIF - CONT D WHAT WHAT WHAT WHAT WHAT WHENE DEET LEVEL A ADJ LEVEL OF M731 TEL A. CHK TON CENT BURNE IN FORUSE LEVEL VIAL. 3 4 QUADRANT NOUNT C. CHK TON CENT BURNE IN FORUS LEVEL VIAL. 3 5 QUADRANT NOUNT C. CHK TON CENT BURNE IN FORUS LEVEL VIAL. 3 5 QUADRANT NOUNT C. CHK TON CENT BURNE IN FORUS IN FAN. 3 5 QUADRANT NOUNT C. CHK TON CENT BURNE IN FORUS IN FAN. 3 5 QUADRANT NOUNT C. CHK TON CENT BURNE IN FORUS IN FAN. 3 5 QUADRANT NOUNT C. CHK TON CENT BURNE IN FAN. 3 5 QUADRANT NOUNT C. CHK TON CENT BURNE IN FAN. 3 5 QUADRANT NOUNT C. CHK TON CENT BURNE IN FAN. 3 5 QUADRANT NOUNT C. CHK TON CENT BURNE IN FAN. 3 5 QUADRANT NOUNT C. CHK TON CENT BURNE IN FAN. 3 5 QUADRANT NOUNT C. CHK TON CENT BURNE IN FAN. 3 5 QUADRANT NOUNT C. CHK TON CENT BURNE IN FAN. 3 5 QUADRANT NOUNT C. CHK TON CENT BURNE IN FAN. 3 5 QUADRANT NOUNT C. CHK TON CENT BURNE IN FAN. 3 5 QUADRANT NOUNT C. CHK TON CENT BURNE IN FAN. 3 5 QUADRANT NOUNT C. CHK TON COULT NOT CHIEF TO SETTION C. RE-CHK TON COULT NOT CHIEF TON TON CHIEF TON	LOADING PROCEDURE - SUSTAINLE FIRTIG - CONT D WHAT WHAT DECRETERED AND LEVEL OF NITHER AND THE LEVEL COMPRESSED AND TH	LOADING PROCEDURE - SUSTAINED FIRITED WHEE CHECK LEVEL & AD LEVEL OF MI) TEL A. CHE FOR EAST BUBBLE IN CROSS LEVEL VIA. 3 ** QUADRIANT MOUNT CHECK LEVEL & AD LEVEL OF MI) TEL A. CHE FOR EAST BUBBLE IN CROSS LEVEL VIA. 3 ** QUADRIANT MOUNT CHECK LEVEL & AD LEVEL OF MI) TEL A. CHE FOR EAST BUBBLE IN CROSS LEVEL VIA. 3 ** CHECK LEVEL AND LEVEL OF MI) TEL A. CHE FOR EAST BUBBLE IN CHARLES. CHECK LEVEL AND LEVEL OF MIS FIRE CONTRIB. A. SIBHT HANDEM EVENTER OF MILLIAM CONTRIL. CHECK LEVEL AND LEVEL OF MIS FIRE CONTRIB. B. MAN MOVE CARRON LONE TO QUADRIANT CHECK LEVEL LATE OF CONTRIB. CHECK LEVEL LATE OF CONTRIB. CHECK LEVEL AND LEVEL OF MIS FIRE CONTRIB. B. MAN MOVE CARRON LONE TO QUADRIANT CHECK LEVEL LATE OF CONTRIB. CHECK LEVEL LATE OF CARRON LATE FOR EAST BUBBLE IN A FOR EVENT LIVEL. CHECK LEVEL LATE OF CARRON LATE FOR EAST BUBBLE IN A FOR EVENT LIVEL. CHECK LEVEL LATE OF CARRON LATE FOR EAST BUBBLE IN A FOR EVENT LIVEL. CHECK LEVEL LATE OF CARRON LATE FOR EVENT LIVEL. A. DINNE CRORN LATE FOR EVENT LIVEL VIAL. A. DINNE CRORN LATE FOR EVENT LIVEL VIAL. CHECK LEVEL LATE OF CARRON LATE FOR EVENT LIVEL. A. DINNE CRORN LATE FOR EVENT LIVEL VIAL. LOADING PROCEDUKE - SUSTAINED FIRATOR - CANT D WANT WANT DESCRIPTION OF DELIBATOR - SUSTAINED FIRATOR - CANT D WHERE DESCRIPTION OF DELIBATOR - SUSTAINED FIRATOR - CANT D WHERE DESCRIPTION OF DELIBATOR - SUSTAINED FIRATOR - CANT D WHERE DESCRIPTION OF DELIBATOR - SUSTAINED FIRATOR - CANT D WHERE DESCRIPTION OF DELIBATOR - SUSTAINED FIRATOR - SUSTAINED TO SUSTAINED WHERE DESCRIPTION OF DELIBATOR - SUSTAINED TO SUSTAINED TO SUSTAINED WHERE DESCRIPTION OF DELIBATOR - SUSTAINED TO SUSTAINED TO SUSTAINED WHERE WHERE DESCRIPTION OF DELIBATOR - SUSTAINED TO SUSTAINED TO SUSTAINED WHERE WHERE DESCRIPTION OF DELIBATOR - SUSTAINED TO SUSTAINED TO SUSTAINED WHERE WHERE WHERE DESCRIPTION OF DELIBATOR - SUSTAINED TO SUSTAINED WHERE WH	HAND TO A TO	L CANSIANG PROCEDURE - Sustantial Resident and resident a	

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HIB?	ном	WHERE
MOVE BARREL INTO BATTERY	A. MAN. MOVE CANNON POS LOAD CNTRL LEVER FR HOLD TO RATTERY UNTIL STOPS IN BATTERY. B. BARREL NOVES TO BATTERY POS & STOPS. C. MAN. REL. CANNON POS LOAD CNTRL LEVER. RET'S TO HOLD.	5 5
CYCLE PRIMER AUTOLOADER	A. MAN. MOVE PRIMER CNIRL LEVER FR FRIME TO EXTRACTED. B. MAN. REL. PRIMER CNIRL LEVER. RET'S TO PRIME. NEW PRIMER IS INSERTED.	5
FIRE PROJECTILE	ON FIRE COMMAND, MAN. MOVE LANYARD CONTROL FR READY TO FIRE. RET'S TO READY.	5
STAGE PROJECTILE	A. SET PROJECTILE IN TRAY. B. MAN. CARRY PROJECTILE TO TRAY.	TRAY
CUT BAG CHARGE	MAN. CUT BAG CHARGE TO REQ'D NUMBER.	CHARGE
STAGE BAG CHARGE	MAN. CARRY RAG CHARGE TO TRAIL.	6 ~

SUSTAINED RATE OF FIRE

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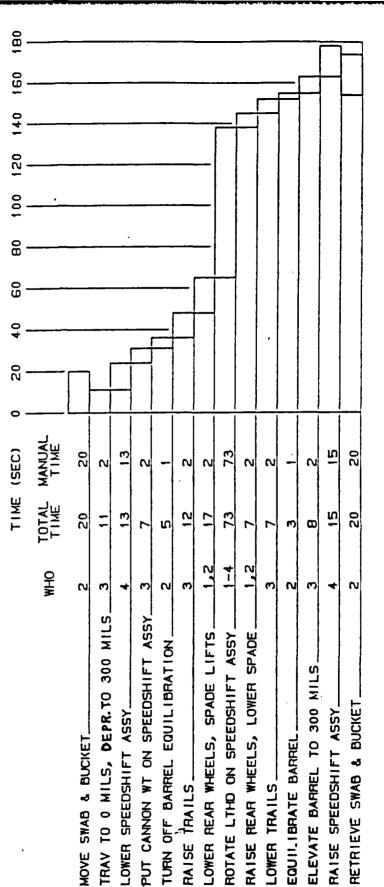
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SPEEDSHIFT TIMELINE



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FNC LIHD - SPEEDSHIFT PROCEDURE

														SPEEDSHIFT	PROCEDURE
HOM	20	vs -	· w •	·	ĸ	₽	s	•	v -	- •	=	<u></u>	91	5	-
N	0	۰ -	. NJ ~0	, e	=	91	61	24	22 23	32	38	37	84	6	3
WHERE	TRAIL	\$/6	2/6	\$/8	SP.SH.			5/6	5/6	6 0	5/6	2/6	2,4		2,4
MOH	NAM. PICK UP BUCKET, SWAB, MOVE DUT OF SPEEDSHIFT RADIUS.	A. MAN. MOVE TUBE LAY CNTRL FR HOLD TO L/R UNTIL NOTCHES ON GINBAL & PLATFM ALIGN. B. BARREL MOVES TO 0 DEG TRAV. NOTCHES ALIGN.		E. MAN. REL TUBE LAY. RET'S TO HOLD.	A. HAN. PULL SPRING-PIN ON SPEEDSHIFT ASSENBLY.	8. MAN. SWIVEL DUT SP SKIFT ASSY. AUTO PINS IN DOWN POSITION.	C. RETR. SP SHIFT DISK FR BII BOX & PLACE ON END OF SP SH ASSY.	A. MAN. MOVE TUBE LAY FR HOLD TO DOWN UNTIL CANNON WT ON ASSY, & STOPS,	B. CANNON LOWERS TO ASSY, STOPS. C. MAN. REL. TUBE LAY. RET'S TO HOLD.	A. NAM. MOVE EQUIL DM/DFF FR DM 10 OFF. B. FRESSURE DROPS, BARREL EQUIL STOPS.	A. HAM. HOVE CANNON LAY FR HOLD TO DOWN	B. TRAILS RAISE AND STOP. C. NAN. REL CANNON LAY. RET'S TO HOLD.	CONCURRENT OPERATION ON LEFT AND RIGHT: A. MAN. NOVE REAR WHL CUTRL FR ON/HOLD TO DOWN UNTIL BOTTOM OF SPADE CLEARS GROWIND BY 1. FT	B. KR WHLS LOWER, SPADE OFF GND BY 1 FT.	C. MAN. REL. RR MM. CNTRL. RET'S TO DN/
BHAT	MOVE RUCKET, SWAB	HOVE BARREL TO APPROX O MIL TRAVERSE & 360 MIL QE			SWIVEL DOWN SPEEDSHIFT ASSY			PUT CAMMON WI ON SPEEDSHIFT ASSY		TURN DEF BARREL EQUILIBRATION	RAISE TRAILS, LIFTING OFF GROUND		LOWER REAR WHEELS, LIFTING SPADE		
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SPEEDSHIFT

QH _B	ID MAHAT	MOM	WHERE	WHEN	108
2 - -	ROTATE LTHD ON SPEEDSHIFT ASSY	A. MAN. LIFT UP RR PLATF TO BAL ON SS ASSY. B. MAN. HOLD LUM., TRAIL HANDLES, PLATF AT	PLATFN	65	900 P
7	-	MAIST HT AND MAN. PUSH HORIZ. ROTATING CN TO NEW POSITION. IF ETTREMELY MARD GROUND:			i
~		MARK HOLE LOC'S FOR SPADE, ROTATE LIND, DIG HOLES, ROTATE TO NEW FIRE POS. C. MAN. PUSH UP ON LUNETTE TO RETURN WT TO RR WHLS.	=	22	u ⁿ
 	? RAISE REAR WHEELS, LOWERING SPADE	CONCURRENT OPERATION ON LEFT AND RIGHT: A. MAN. HOVE REAR WHL CNTRL FR ON/HOLD TO UP THAT I WAS STOP	2,4	138	- 0
		B. RR WHLS RAISE, STOP, SPADE IN GROUND. C. MAN. REL RR WHL CNIRL. RET'S 10 ON/HOLD.	2,4	139	vo
m	LOWER TRAILS	CONCURRENT OPERATION ON LEFT, RIGHT & REAR: A. MAN. MOVE CAMMON LAY FR HOLD TO UP UNTIL TRAILS ON GROUND.	5/6	145	-0
		B. TRAILS LOWER TO GROUND. C. MAN. REL CANNON LAY. RET'S TO HOLD.	5/6	2 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	v
2	EQUILIBRATE BARREL	A. MAN. NOVE EQUIL ON/OFF FR OFF 10 ON. B. BARREL EQUILIBRATES.	œ	152 153	- 2
~	ELEVATE BARREL 10 APPROI 300 MILS		5/6	155	~
		B. CANNON RAISES TO APPROX 300 MILS, STOPS. C. MAN. MOVE CANNON LAY FR UP TO HOLD.	5/6	156 162	9
•	RAISE SPEEDSHIFT ASSY	A. MAN. REMOVE SS DISK. B. MAN. PULL SPR-PIN ON SS ASSY. C. MAN. SWIVEL SP SH ASSY UP. AUTO LOCKS IN UP POSITION.	SP.SH. ASSY	163 168 173	N W W
2	RETR BUCKET, SMAB	MAN. RETRIEVE BUCKET, SWAB, AND PLACE BY INSIDE LEFT TRAIL.	TRAIL	155	20

DISTACEMENT PLOCETURE

FMC LTHD - DISPLACEMENT PROCEDURE

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2	T GITTE	HOM	WERE
2	MOVE BARREL TO 0 DEG TRAVERSE	A. MAN. HOVE CANNON LAY CTL FR HOLD TO L/R 5// UNTIL NOTCHES ALIGN ON GIMBAL & PLATFORM, B. BARREL MOVES TO 0 DEG TRAV, OBS NOTCHES, C. MAN. REL CANNON LAY CTRL. RET'S TO HOLD, 5//	516 576
~	DEPRESS BARREL TO GROUND	A. MAN. MOVE CANNON LAY CTL FR HOLD TO DOWN 5// UNTIL CRADLE RESTS ON GROUND. 8. CRADLE RESTS ON GROUND. C. HAN. REL CANNON LAY CTRL. RET'S TO HOLD. 5//	5/6
-	TURN OFF BARREL EQUILIBRATION	A. MAN. NOVE EDUIL ON/OFF VALVE FR ON 10 OFF, B B. RAKREL EQUILIBRATION TURNED OFF,	,
	RETRACT CANNON TO TOM POSITION	A. MAN. MOVE CANNON POS-EMPLACE CNIRL FR 8 HOLD TO TOW UNTIL BARREL STOPS. B. BARREL RETRACTS, STOPS. C. MAN. REL. CANNON POS-EMPLACE LEVER.	,
•	SECURE RARREL TRAVEL LOCKS	A. MAN. PUSH IMMARD ON LEFT TRAVEL LOCK 8 LEVER UNTIL STOPS. B. MAN. PUSH IMMARD ON RIGHT TRAVEL LOCK 8 LEVER UNTIL STOPS.	
~	RAISE TRAILS	A. MAN. MOVE CANNON LAY CTL FR HOLD TO DOWN 5/ UNTIL TRAIL MOVE UP AND STOP. B. TRAILS RAISE AND STOP. C. MAN. REL CANNON LAY CTL. RET'S TO HOLD. 5/	5/6 5/6
2,3	UNPIN TRAILS FROM PLATFORM	CONCURRENT ON LEFT & RIGHT: MAN, RCIATE BOLT T-HEAD CCH UNTIL LOOSE 5, AND BOLT SPRINGS OUTWARD FROM NUT.	5,6
- :	PUSH TRAILS IMMARD UNTIL TRAILS Stop	CONCURRENT ON LEFT AND RIGHT: MAN. PUSH HORIZ & PERP TO TRAILS AT ENDS & 9, WALK UNTIL TRAILS HITS CRADLE STOPS.	9,10
~	LOWER TRAILS	A. MAN. MOVE CANNON LAY CNTRL FROM HOLD TO 5/1 TO UP UNTIL TRAILS LOWER AND STOP. B. TRAILS LOWER AND STOP. C. MAN. REL CANNON LAY CNTL. RET'S TO HOLD. 5/1	5/6 5/6

DISPLACEMENT PROCEDURE - CONT.D.

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ACCEPTATION CONTROL DESCRIPTION OF STANKING STANKING THE PROPERTY.

DH.	TAHAI	HOM	WHERE
es	LOCK L & R TRAIL TO CRADLE	A. MAN. PUSH ON LEFT PLUNGER PIN NITH THUMB. B. MAN. PULL & REMOVE PLUNGER PIN. C. MAN. SNIVEL TRAIL LOCK OUT & DOWN. D. MAN. REPLACE PLUNGER PIN IN MOLE. E. KEPEAT ON RIGHT SIDE OF CRADLE.	6 6 6 6 0 1
<u>-</u>	LOWER REAR WHEELS	CONCURRENT ON LEFT AND RIGHT: A. MAN. HOVE REAR WHL CONTROL FR ON/HOLD TO DOWN UNTIL SPADE IS OUT OF GROUND. B. REAR WHLS LOWER, SPADE IS OUT OF GROUND. C. MAN. REL. REAR WHL CONTROL. RET'S TO ON/HOLD.	2,4
* .	LOWER FRONT WHEELS TO LOCK WHL FRANES	CONCURRENT ON LEFT AND RIGHT: A. MAN. PULL DUTWARD ON FRAME HANDLE. 1,3 B. MAN. SMIVEL FRAME UP & HOLD IN POSITION. 1,3 C. MAN. MOVE FRI WHL CNTRL FR ON/HOLD TO DFF. 1,3 D. FRI WHLS LOWER, FRAME ALIGNS WITH STOP. E. MAN. RELEASE FRAME HANDLE. (FRAME LOCKS.) 2,4 E. MAN. SMIVEL FRAME UP & ALIGN M/ STOP.	1,13 1,13 1,13 2,14 2,14
* .	LOWER FRONT WHEELS TO SHIFT WEIGHT (O FRONT WHEELS	COMCURRENT ON LEFT AND RIGHT: A. MAN. MOVE RR WHL CNTRL FR DN/HOLD TO UP UNTIL 4 WHLS DN GROUND. B. RR WHLS RAISE, 4 WHLS DN GROUND. C. MAN. MOVE RR WHL CNTRL FR UP TO DFF. D. MAN. MOVE FRT WHL CNTRL FR OFF TO TO DOWN UNTIL REAR WHLS ARE OFF SROUND ABOUT 3 INCHES. E. FRT WHLS LOWER TO LIFT REAR WHLS OFF GND. F. MAN. REL. FRT WHL CONTROL. RET'S TO ON/HOLD.	2,4 1,3 1,3
1,3,	1,3,4 RAISE FRONT OF LTHO OFF GROUND	COMCURRENT OM LEFT AND RIGHT: MAN. HOLD HAMDLES AT TRAIL ENDS, RAISE FRI OF LTHD OFF GROUND.	9,10

DISTUCEMENT

DISPLACEMENT PROCEDURE - CONT.D.

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۲.	POSITION, GUIDE TRUCK	DRIVE TRUCK UNTIL PINTLE IS UNDER LUNETTE.	12
£.	1,3,4 LOWER LUNETTE TO PINTLE	CONCURRENT ON LEFT AND RIGHT: MAN. HOLD HANDLES AT TRAIL ENDS, LOWER LUNETTE ONTO PINTLE.	9,10
* :	SHIFT NT TO ALL FOUR WHEELS	CONCURRENT ON LEFT AND RIGHT: A. MAN. MOVE FRT WHL CNTRL FR ON/HOLD TO OFF. 1,3 B. ALL FOUR WHLS REST ON GROUND.	1,3
n	CLOSE ACCUMULATOR VALVE	MAN. NOVE PUMP CONTROL VALVE FROM ON TO OFF.	ĸ
7	OPEN & ATTACH BRAKE AIR LINES	A. MAN. DISCNCT SERV HOSE ASSY FR DUMMY CPLG. 10 B. MAN. COUPLE ENERG AIR LINE COUPLING. C. MAN. DISCNCT SERV HOSE ASSY FR DUMMY CPLG. 10 D. MAN. COUPLE SERVICE AIR LINE COUFLING. E. MAN. OPEN ENERGENCY AIR LINE CUTOUT COCK. 12 F. MAN. OPEN SERVICE AIR LINE C/O COCK.	10 12 12 12
* .	REL FRONT WHEEL MANUAL BRAKES	CONCURRENT ON LEFT AND RIGHT: A. MAN. TAKE BRAKE HANDLES FR CLAMPED POS & PLACE IN SOCKET. B. MAN. PULL UP ON HAMDLE UNTIL STOPS. C. MAN. REPLACE HANDLE TO CLAMPED POS.	1,3 1,3
m	LOCK LUNETTE TO PINTLE	A, MAN. CLOSE TOWING PINTLE LATCH. B. MAN. INSERT PINTLE COTTER PIN.	12
2	CNCT. BLACKOUT LIGHT CABLE ASSY To prime mover	A. MAN. DISCNCT ASSY FR BUMMY COUPLING. B. MAN. COMNECT P.M. B/D LT CABLE ASSY.	2 2

DISPLACEMENT (ROCEDURE

FMC LTHD COMBINED MISFIRE PROCEDURE TASKS

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CONTRACTOR CONTRACTOR

INCLUDES ALL TASKS FOR:

HANGFIRE, STICKERS WITH CHARGE 1 AND M107 OR M549 FAMILY, CHARGE HOT TUBE, COLD TUBE, WARM TUBE ON HOT DAY, WARM TUBE ON COOL DAY, OR 2 AND M483 FAMILY, CHARGE 1,2 OR 3 AND M712 PROJECTILE.

THE CONDITIONS DETERMINE WHICH TASKS SHOULD BE PERFORMED.

FOR ALL FAILURES TO FIRE AFTER ACTIVATING THE FIRING MECHANISM;

- 1. ATTEMPT TO REFIRE
- A. MANUALLY MOVE LANYARD CONTROL FROM SAFETY TO FIRE.
 - 3. REPEAT.
- 2. WAIT 2 MINUTES (OR EVACUATE IF TUBE IS "HOT"),
- 3. RETURN BARREL TO LOAD POSITION, INSPECT PRIMER.
- MANUALLY MOVE PRIMER LEVER FROM PRIME TO EXTRACT. (PRIMER IS EXTRACTED.)
- MAN. MOVE BREECH FOSITION LEVER FROM BATTERY TO LOAD. (BARREL MOVES FROM BATTERY TO LOAD POSITION.)
 - C. MAN. REMOVE PRIMER AUTOLOADER DRUM.
- 1. MAN. PULL CAM PLATE OUT UNTIL STOPS.
- 2. PUSH IN PLUNGER ON PIN WITH THUMB.
- 3. PULL PIN. (HALF-CIRCLE DRUM RETAINER SWINGS DOWN.)
 4. LIFT DRUM OFF RETAINER.
 - D. INSPECT PRIMER.
- 1. INSPECT EXPOSED PRIMER TO SEE IF DENTED BY FIRING PIN. IF NOT, FIRING MECHANISM IS FAULTY.
 - 2. INSPECT PRIMER TO SEE IF FIRED.
- IF DENTED BUT PRIMER NOT FIRED, PRIMER IS A DUD. IF PRIMER FIRED, A MANGFIRE HAZARD REMAINS, AND THE CREW SHOULD WAIT AT LEAST 10 MINUTES - MORE IF THE TUBE IS HOT - REFORE OPENING THE BREECH BLOCK.

MISFIRE PROCEDURE TASKS MISFIRE PROCEDURE TASKS - CONTID.

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TO CLEAR BY FIRING:

- REPLACE CHARGE (IF FIRED AS A STICKER, OR IF UNSERVICEABLE AS A RESULT OF A HANGFIRE, WETNESS OR HEATING).
- A. MAN. MOVE BREECH BLOCK LEVER FROM CLOSE TO OPEN. (BREECH OPENS.)
 - REMOVE CHARGE (IF NOT A STICKER).
- MAN. MOVE BREECH BLOCK LEVER FROM OPEN TO CLOSE. (BREECH CLOSES.) LOAD FRESH CHARGE (CHARGE 5 OR HIGHER FOR A STICKER).
- 5. INSTALL FRIMER DRUM, RETURN BARREL TO BATTERY POSITION.
 - A. MAN. REPLACE PRIMER AUTOLOADER DRUM.
 - 1. REPLACE DRUM INTO RETAINER.
- 2. SWIVEL UP DRUM RETAINER, HOLD AND INSERT PLUNGER PIN. MAN. MOVE BREECH POSITION LEVER FROM LOAD TO BATTERY. (BARREL MOVES TO BATTERY POSITION,) ä
- 6. LOAD NEW PRIMER.

MAN. MOVE PRIMER LEVER FROM EXTRACT TO SET. (NEW PRIMER POSITIONED.)

7. FIRE.

MANUALLY MOVE LANYARD CONTROL FROM SAFETY TO FIRE.

TO CLEAR BY UNLOADING: OR:

- 4. REMOVE CHARGE.
- MOVE BREECH BLOCK LEVER FROM CLOSE TO OPEN. (BREECH OPENS.)
 - REMOVE AND DISPOSE OF CHARGE.
- EITHER: 5. REMOVE PROJECTILE.
- REMOVE M107, M483, M549 FAMILY PROJECTILES BY:
 - 1. ASSEMBLE RAMMING STAFF.
- INSERT RAMMING STAFF IN MUZZLE. LOWER BARREL TO 0 DEG QE.
- INSERT PADDING INTO CHAMBER.
- MOVE BREECH BLOCK LEVER FROM OPEN TO CLOSE. (BREECH CLOSES.)

PROCEDURE MISFIRE 155KY

PROCEDURE MISFIRE

1 ASKS

MISFIRE FROCEDURE TACKS . COUT'D.

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CERTAIN SECTION INSPERSE INSPERSE CONTRACT SECURIOR IN 1999

PUSH PROJECTILE INTO CHAMBER WITH RAMMING STAFF.

MOVE BREECH BLOCK LEVER FROM CLOSE TO OPEN. (BREECH OPENS.)

REMOVE PADDING.

(LDAD TRAY ADVANCES SLOWLY AND STOPS AT BREECH.) ADVANCE LOAD TRAY BY MOVING RAM CONTROL FROM RETRACT TO

FUSH PROJECTILE INTO LOAD TRAY WITH RAMMING STAFF. 10.

REMOVE RAMMING STAFF.

A. PUSH IN PLUNGER ON PIN WITH THUMB. UNPIN LOAD TRAY FROM LOAD TRAY LINK.

B. FULL FIN.

MAN. FUSH LOAD TRAY, PROJECTILE TO BACK-MOST POSITION. 13.

REMOVE AND DISPOSE OF PROJECTILE.

REMOVE NUCLEAR OR COPPERHEAD (M712) PROJECTILE BY: 1. ASSEMBLE EXTRACTION DEVICE. æ.

er.

LOWER BARREL TO O DEG ME.

INSERT EXTRACTOR INTO BREECH.

ATTACH TO PROJECTILE.

EXERT REARWARD FORCE (HYDRAULIC FOR NUCLEAR, RATCHET FOR M712) TO UNSEAT PROJECTILE FROM FORCING CONE,

DETACH EXTRACTOR.

(LOAD TRAY ADVANCES SLOWLY AND STOPS AT BREECH.) ADVANCE LOAD TRAY BY MOVING RAM CONTROL FROM RETRACT TO

MANUALLY PULL PROJECTILE INTO LOAD TRAY.

A. PUSH IN PLUNGER ON PIN WITH THUMB, UNPIN LOAD TRAY FROM LOAD TRAY LINK.

B. PULL FIN.

MAN. FUSH LOAD TRAY, PROJECTILE TO BACK-MOST POSITION. REMOVE AND DISPOSE OF PROJECTILE. 10.

C. CLEAR WEAPON BECAUSE WEAPON IS "HOT" AND CALL FOR EXPLOSIVE ORDNANCE DIVISION (EDD) TO REMOVE PROJECTILE. DESCRIPTION: Master schedule

STATUS: The LTHD Phase II Design (pg 2) and Hardware (pg 3) master

schedules are current and complete.

AUTHOR: Robert Rathe, Bart Anderson

DATE	REVISION (LTHI) PROBLE LOGI	REDITE PRIASE II
	EVENT	MONTHS AFTER CONTRACT
I YEM	NOTETION (NEW PRINCIPLE)	1 2 3 4 5 6 7 6 9 10 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
	PHASE 11: DETAILED HARDWARL DESIGN.	
-		
~	PREPARE SPECIFICATIONS	
7		
-	PREPARE LEVEL 11 URAWINGS	
S.		
g	DEVELOP CONTROL LOGIC (AS APPLICABLE)	
_		
8	ORDER LONG LEAD TIME ITEMS AND MATERIAL	
6	DEVELOP PRODUCT ASSURANCE PLAN	
è	CONDUCT TRADEOFF ANALYSIS	
=	DEVELOP ACCEPTANCE TEST PLAN	
12	CONDUCT INTERNAL REVIEWS (TENTATIVE)	
5		
	DELIVERABLE ITEMS;	
-	PRODUCT ASSURANCE TEST PLAN C.2.C.2.	
~	MATERIAL TEST SAMPLES C.2.C.2.4	Δ
÷	LEVEL II DRAWINGS (A004)	
	PROGRAM PLAN (A005) (REVISE AS REQUIRED)	
5.	RELIABILITY STRESS ANALYSIS REPORT (A012)	
မ်	LONG LEAD TIME TIEMS LIST (A013)	
-	COMPUTER PROGRAM LISTING (A014)	
.8	COMPUTER PROGRAM MANUAL (A015)	
6	COMPUTER PROGRAM FLOW CHARTS (A016)	
-	PERFORMANCE AND COST REPORT (A017)	
Ξ	PURCHASE DESCRIPTION (A019) (AS REQUIRED)	
12	AGENDAS (A020) (NLT 15 DAYS PRIOR TO MEETING)	
-3	MEETING MINUTES (A021) (NLT 15 DAYS AFTER MEETING)	
=	TRADEOFF ANALYSIS REPORT C.2.C.2.9	
15		
9	DEMONSTRATOR SPECIFICATION C.2.C.2.0	
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MANAGEMENT SETTINGS	45710	
Dynamic Analysis Report - Revised		
PHA Report - Revised	:	
Quality Program Plan	:	
Stress Analysis Meport		
dware (see Note 1)		
Gistal-platfors-spade assembly.	624-01/64	
Cradit assembly to gradal	001 0143	
Trail subasseably	100-01/54	
Irail assembly to system	71.70-DI	
Struct test, redesion, reports		
Cal 1004 fr gimbal-platfm-spade.		
Compound actuator subassembly	C/7-01/C+	
Compound actuator assy to cradie	097-01/5	
Alignment: Mays to BE/AL axis	C/1-01/C+	
Fire ctrl (static.struct only)	Mh-61/C4	
Hydraulic piping network	E94-01/C4	
Uperational procedures/resting	5710-750	
Canada Subassemuly	45710-760	
Cannon assembly into traditions	5710-470	
mystable system startup a timot		
The first and make the contract of the contrac	£5710-475	
Fire chr & notice completion	+5710-400	
Walking bear whis and axles	+5710-625	
Walking beans	\$5710-650	
Walking brans to trails	+5710-675	
Brake system	•5710-225	
Taillight and niring	(5710-225	
Speedshift assy	676-0176	
Basic issue lieus and Container.	15710-500	
Install 1st article euzzle brake		
Ship to Army test site	:	

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THO ASSENBLY SEQUENCE AND SCHEDULE

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Hele 1: Material should arrive 1-3 wks early to allow for inspection and croblems.

DELIGRIFIEN: FABRICATION GROUNDWORK

(MARE BUY AND SHOP LOADING)

STATUS: A preliminary make/buy decision meeting was held in which the major LTHP parts to be produced in-house were identified. FMS s facilities, equipment and man-power capabilities were compared with the needs of the program. In-house fabrication includes welding and machining titanium parts, machining Al/SiC parts, and the constriction of small composite parts.

Manufacturing was in the process of gearing up for LTHD parts to be manufactured in-house. Dedicated cells were being set up in weld, machining and assembly. A limited production cell was also being set up for composites.

AUTHOR: Jim Wallece, J.F. Tousle:

XX

THD MAKE BUY

1 ST REVIEW

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STATES CONTRACTOR

Lateral Constant Constant Participate Constant

PREPARED BY J. WALLACE J. TOUSLEY M. HASS

CHO ASSEMPLY SEDVENCE AND SCHEDULE

				LTHD ASSEMPLY SE	LTHO ASSEMPLY SEQUENCE AND SCHEDULE			
			1987			8861	6	
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Oynamic Analysis Report - Revised								
PHA Report - Revised								
Colore Boalesic Report								
Acceptance Test Plan						•	:	
Hardware (see Note 1)	•							
Giebal-Blatform-spade assembly.	15710-425		E					
Cradle assembly to gimbal	+5710-340		•					
Trail subassembly	+5710-600							
frail assembly to system	+5710-595	•						
Struct test, redesign, rework	:					•		
Commend actuator subassembly	+5710-275	: :						
Compound actuator assy to cradle		•						
Alignment: ways to DE/Al axis	_					:		
Fire ctrl (static.struct only)	+5710-400			E				
Hydraulic piping network	45/10-465	•						
Uperational procedures/testing	\$5710-750							
Cannon assembly into cradle	+5710-240	•		•				
Hydraulic system startup & chkot	45710-470			L				
Install PSA muzzle brake		•						
Load tray and may	+5710-475	•						
Fire ctrf & optics completion	45710-40U	•						
Majering bras whis and axing	059-01/6							
Talking Deals to trails	15710-675	•						
Brake system	\$5710-225			:::::::::::::::::::::::::::::::::::::::				
Taillight and miring	•5710-225							
Speedshift assy	45710-575				•			
Based assume and lenstance.	15710-500	_						
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Contract participal property contracts respected to the property of the proper

Control Control

CONCERNS FOR MANUFACTURING

- · NEW DESIGNS ASSOCIATED PROBLEMS
- . SHORT LEAD TIMES
- · NEW MATERIALS .T.I.
- · MIN. CUITING DATA AVAILABLE
- · CODLANT TYPES
- · CONTAMINATION
- · HEAT TREAT
- · PLATING

MANUFACTURING MREA

CELL

1294 HRS

106 PARTS

229 GOERATIONS

OUTSIDE CELL

266 HRS

11 PARTS

22 OPERATIONS

AROCESSES REG

N/C PROGRAMS

TOOLING REQ

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11.

(A)

MANUFACTURING CONCERNS

- A. RECTANGULAR TUBE
- B. TOOLING LEAD TIME
- C. MECHANIEAL RE-CONTOURING PARTS
- D. FORMING (COLD-HOT)
- E. MATERIAL CONTAMINATION
- DESIGN, WELD JOINTS, ACCESSIBILITY ň

WELD MANNEALTURING

WELD CELL (CLERN ROOM)

183.0 Hes.

SET-UP & UNIT

ONT-SIDE CELL (WELD SHOP)

65B.0 HRS.

SET- UP & UNIT

PROCESSING

900.0 Has.

360 As

2.5/11.

Ne PROGRAMS

1088.0 HRS.

136 TAPES

8.0/TAPE

TOOLING

DESIEN

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595,0 HRS.

1190.0 Hes.

17 FixTubes

ASSEMBLIES (WELD)

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MACHININGS CELL NON CELL

Hours 703

179 PCS 7 PCS

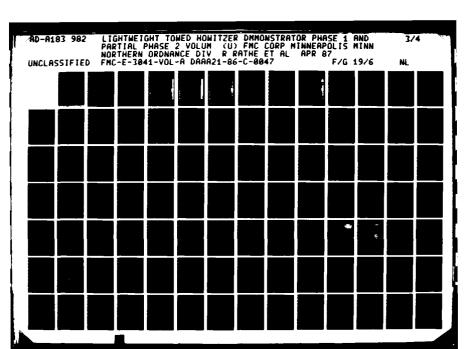
PROCESS REQ

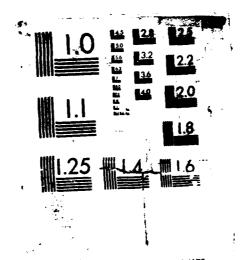
TOOLING

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Jim WAIlAGE

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Declaration Application (Separate Opening)		
FTG GROOT TOWN		
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fire ctrl (static struct poly)	· 62.16-46v	
Mydraulic pizing networb	*5719-465	
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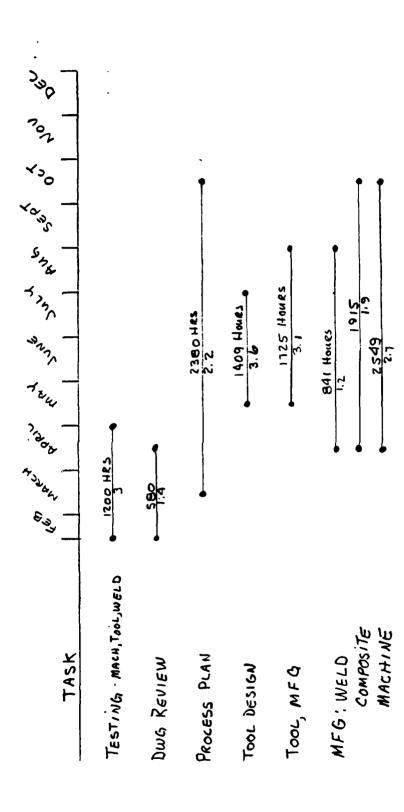


MICROCOPY RESOLUTION TEST CHART

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LTHD MFGENG DLAN FY 1987



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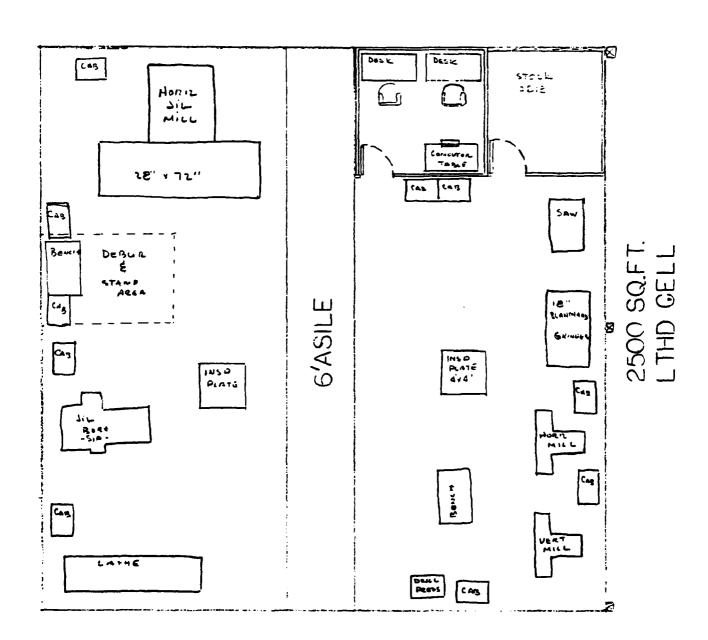
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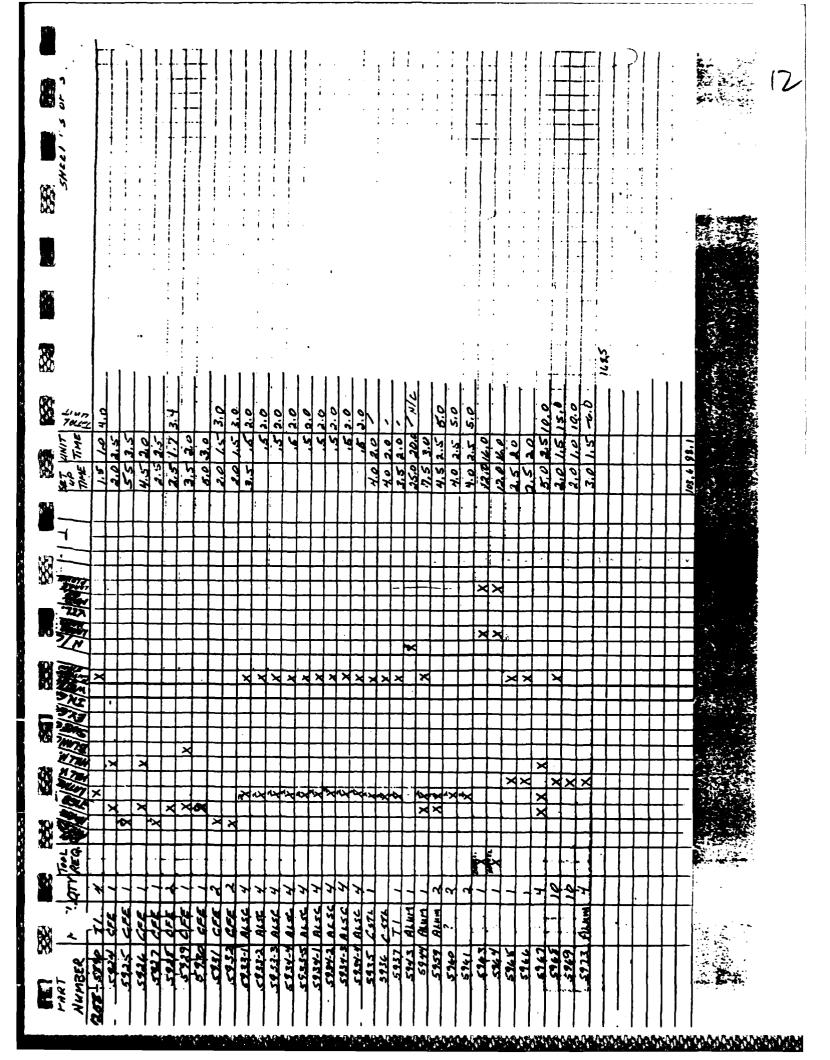
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MFG ENG MAN POWER



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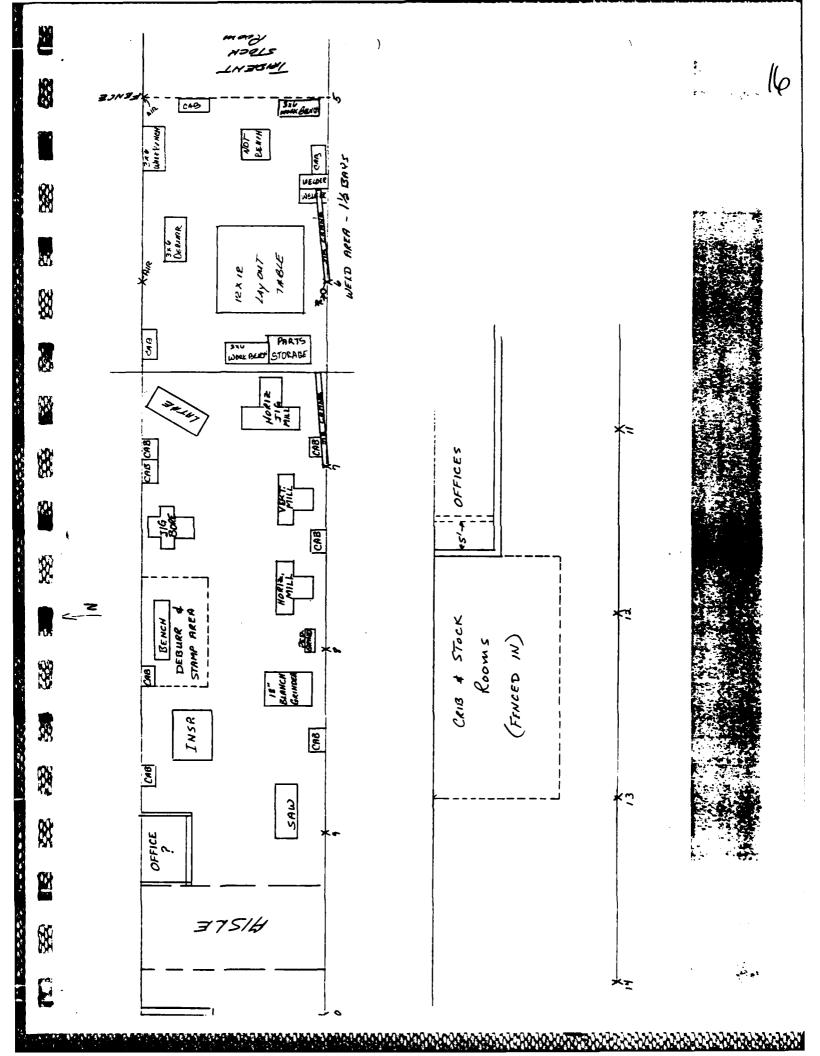
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2500 SQ.FT. LTHD GELL

Northern Ordnance Division Minneapolis

REQUEST FOR FACILITY DEPARTMENT SERVICES

			2-26-87
GD, LANNUL	Extension 72//	Department LUEL D	Account Number
19 Support of the new w	veld Gran Ruon , The	· Lightweight Towed H	SI. switzer Program requires Fitt
bays in the wold Dept	for a Machine shop	area and a wold-la	yout, NDT and Cleanup area
for building the prot	to type LWTH, The	s proposed area show	11 be washed down, painted
Floor Sealed, Chair la	nk fenced have a	mber drep curtain fr	- ceiling to floor wall effect
Location West 11th Posts	Salan at cut &	3) 1/2 Ti 13 Existing	or for machines and wolders.
Budgeted	Amount Bu	dgeted Cost Limit	AFE Budget Number
SPECIAL CONSIDERATIONS:			
Space Planning/Layout		Estimate Require	ed; Date <i>ASAP</i>
Shop Office		Equipment Insta	
Utilities Required		Telephone	
Air Water		Quantity	
		Name Plate(s); N	ame(s)
Furniture Required			
Equipment Delivery Date		Operational Requiremen	nt Date
Executive Paper Approval		7/26/62	
FACILITY DEPARTMENT ACTION	N		
Assigned To		Exten	sion
Action			
DISTRIBUTION:		Facili	ty Department Approval
Facility Dept. Requestor Office Info. Services MIS		Office Repair Control Center	
THE THE PARTY OF T			



DESCRIPTION: PHASE II SCHEDULE

STATUS: A schedule for the completion of Phase II was developed based on estimates of completion from all individuals working on the project. The schedule is current and complete.

AUTHOR: Bart Anderson, Dave Boudreau and Scott Dacko

PROJECT: LTHD REMAINING PHASE II

FILE: PHASE2

	L EMAINING PHASE !!		lHrs	: Nho		ch 198 9	97 16	23	30	¦Apri 6	1 1 ¹		27	lMay 14	1987 11	18	25	June	1987 8
			==+====	}====:	+===	=+====	:+===:	+====	+===:	:+== :	+==:	==+===	+====	+== = =	+EEE E	+===:	+====	+====	+====+
		11100		i	11	•	•	•	•	•	•	•	•	•	•	•	•	•	• •
	ADV-NF6 METAL	350213 31091	, ,	, w e	!!	•	•	•	•	•	•	•	•	•	•	•	•	•	• •
	MANUFACTURING REVIEW	750711 71011	: 400		;									-	•	•	•	• 1	• •
	MECHANICAL ENGRG	350211 31011	; 70	•	11	•	•	•	•	•	•	•	•	•	•	•	•	•	
100	STABILITY ANALYSIS		: 70		;;	•	•	•	•	•	•			•	•	•	•	•	• •
200	ASEMBLY DRWG PREPARATION	ŀ	817		;									-	•	•	•	•	•
	UPDATE TOP		: 484	—	i		.~~~								•	•	•	•	
	LONG LEAD ITEMS	44644	: 60		;							• •	•	•	•	•	•		•
37	=	11200		i	11	•	•	•	•	•	•	•	•	•	•	•	•	•	• •
	MECHANICAL ENGRG	350212 31011		i -	11	•	•	•	•	•	•	•	•	•	•	•	•	•	
	DRWG PREPARATION		1 128	HE					•	•	•	•	•	•	•	•	•	•	• •
		11300			11	•	•	•	•	•	•	•	•	•	•	•	•		
4/5	ADV MFG-ORGANIC COMPOSITE			•	11	•	•		•	•	•	•	•	•	•	•	•		
	MF6 REVIEW OF COMPOSITES		200		;									-		•	•		
00	CEL-STRUCTURAL ANALYSIS	350211 31031	:	•	11	•	•	•	•	•	•	•	•	•	•		•		
	CEL/RODAMACHER		: 0:	CE								٠.			•				
_	MECHANICAL ENGRG	350213 31031	1	•	11	•		•		•		•	•				•		
485	MANIFOLD/HL DESIGN		1 647		¦						•	•			•				, ,
	MANIFOLD/HL DRWG PREP		1 567	ME	!										-				
Qu	HYDRAULIC SYS DESIGN		: 493	ME	}									•					
	HYDRAULIC SYS DRWG PREP		430	HE	;										-				
Œ	LOAD TRAY & WAY DRWG PRE	P	646	ME	;									-					
22	TRAIL DESIGN		; 180	ME	;									٠.					
	'L DRWG PREP		1 586	HE	}										-				
	af,GIMB,SPADE DESIGN		1 192	ME	 														
	PLAT, SIMB, SPADE DRWG PRE	P	1 591	HE															
Θ	CRADLE DESIGN		1 120	ME	;									٠.					
	CRADLE DRWG PREP		100	ME	;										-				
	SMALL PARTS DESIGN		i 42	ME															
	SMALL PARTS DRWG PREP		: 65	ME	 										-				
170	MEIGHT REDUCTION		: 575	ME	{														
	NON-STRUCT MECHANICS	350213 31021	:	;	11														
68	N S ANALYSIS		: 130	: AE	;														
	ORGANIC COMPOSITES	350213 31011	:	;	11														
3.3	COMPOSITE ANAL/DESIGN		: 684	30:	;														
_	STRUCTURAL MECHANICS	350213 31041	!	:	!!														
70	S M AMALYSIS		: 925	AE	!										· -				
	YORK INDUSTRIES	350213 31061		;	11														
	YDRK ANAL/DESIGN/DRWGS			YD										-	_		•		
70		11400		;	11									_		•	•		
	MECHANICAL ENGRG	350214 31011		:	11	-			-	•		•		•	•	•	•		
	FIRE CONTROL DRWG PREP		449	ME	11	•	•			· 	•		•	-		•	•		
		12100		<u>-</u>	;;	•	•	•	_						•	•	•	•	
88	QUALITY ASSURANCE	350211 31021		;	;;	•	•		•	•	•	•	•	•	•	•	•	•	• •
Ç,	QUALITY PROGRAM PLAN	•••••		QA	!	· 	•	· 		' 	, ,	, 		•	•	•	•	•	• •
		12200		:	::										•	•	•	•	• •
JW.	TEST	350222 31011	-	:	11	•	•	•	•	•	•	•	•	•	•	•	•	•	•
N.	ID TEST VENDOR	A4444 A1A11	: 81		!				•	•	•	•	•	•	•	•	•	•	• •
		12300		; IE	11								•	•	•	•	•	•	• •
	COMPOSITES	17200		, !	11	•	•	•	•	•	•	•	•	•	•	•	•	•	
36	SUPPORT COMPOSITE TEST		: 150	i Par	1	•	•	•	•		•		•	•	•	•	•	•	•
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PAGE 2

MAR 02 87 PROJECT: LTHD REMAINING PHASE II

FILE: PHASE2

0				Harc	h 198	7			April	19B	7		Hay	1987			:June	198	7
	L' EMAINING PHASE II	Hrs	: Who	12	9	16	23	30	16	13	20	27	14	11	18	25	!1	8	
	SYSTEMS ENGRG 13100	+==== !	+===== !	:+==== 	+====	+====	+====	+====	:+====)=== <u>=</u>	+25EE	+====	. + 2 2 2 2		+====	+= ===	+====	+===:	=+
1	SYSTEMS ENGRG 350231 31011,21,31,41	:	•	11	•	•		•			•			:		•			:
N.T.	QPP, REL, TEST, SPECS, PHA PROJECT MANAGEMENT 13200	: 323	ISY											•	•	•	•	•	•
	MECHANICAL ENGRG 350232 31031	;	:	11				•		•	•		:		•		•	•	:
	PROJECT COORDINATION	200	IME	;													•	•	•
	PROJECT MANAGEMENT 350232 31011 PROJ MANAGEMENT	; ; 80	: !PM		• 					•	•			•	•	•	•	•	•
	TRAVEL NOD/CEL 350232 31021	}	:	11		•							•			•		•	
OL/	TRAVEL MODEL	; 0 ;	: X :	;- ;;	•	•						•	•	•	•	•	•		•
W	MODEL MAKING	. 0	1 X	 		•				•	•	•	•	•					
X	DATA PREPARATION 14100 MECHANICAL ENGRG 350241 31011	; !	 !	# !!	•	•	•	•	•	•	•	•	•	•	•	•	•	•	•
Li ■·	COMPILE REPORTS	1 448	ME	;;	•	•	•	•			•	• 		•	•	•	•	•	
X	BFE SUPPORT 15100 MECHANICAL ENGRG 350251 31011	!	;		•	•	•	•	•	•	•	•	•	•	•	•	•	•	•
_	SFE COORDINATION	: 46	HE	;;		•	•					-	:		•	•	:		:
	SPARE PARTS 15400 MECHANICAL ENGRG 350254 31011	;	:	#	•	•	•			•	•		•	•		•	•	•	•
32,	MECHANICAL ENGRG 350254 31011 COMPILE PARTS LIST	1 90	IME			• 				, 	· 	• 	-			•			:

PAGE 3

PROJECT: LTHD REMAINING PHASE II

FILE: PHASE2

				iMa	arch	1987			11	April	1987		;M	ay 198	7		June	1987
	L' EMAINING PHASE II		: Who	-		•			30 1		13	20 2	7 14	11	18	25	11	8
	RESOURCE SUMMARY	*****	:+====:	:+=:	===+=	===+:	:22=+:	===+:	E===+:	===+:	====+:	====+=	SEE \$ 2	#=# + ##	:EE † BE =	=+===	:=+E2==	+== = =+
Ŷ	MANUFACTURING ENGR 55	21	:MF	ŀ	67	67	67	67	67	67	67	67	65					
(Ly	NECHANICAL ENGR 55	21	HE	;	904	904	904	936	936	964	891	B30	708	190	4			
	CEL 40	1	ICE	;														
	ANALYTICAL ENGR 52	21	IAE	;	121	121	121	121	109	92	92	92	92	92				
(1)	COMPOSITE ENGR 52	H	:00	1	78	78	78	78	78	78	78	78	62					
~•	YORK PESONNEL 40) {	140	1														
(EA	QUALITY ENGR 52	!}	!₽A	;	9	9	9	9	9	9	9	9	9					
K	TEST ENGR 53	21	ITE	1	12	12	12	12	12	12	11							
	SYSTEMS ENGR 52	}}	ISY	;	36	36	36	36	36	36	36	36	35					
	PROJ MANAGER 48);	i PM	;	8	8	8	8	8	8	8	В	8	θ				
	UNASSIGNED (H	!X	ł				•										
5287	TOTAL HOURS	;	;	: 1	1234	1234	1233	1266	1254	1265	1192	1119	9 79	290	4			

		R 02 87 DJECT: LTHD	ACTIV REMAIN	ITY STATU ING PHASE	S - ACTUAL II	. VS	PLAN	FILE	PAGE : PHA		
					End Planned Rev/act		Actual (+) To-Date T	Fcst (=)	Latest (vs)	Orig (=	
	P010	FINAL ASSEMBLY 11100									
	A0 10	ADV-NF6 NETAL 350213 31091									
	T040	MANUFACTURING REVIEW		3-02-87	5-01-87	MF	0	400	400	400	0
	A 020	MECHANICAL ENGRG 350211 31011									
	T040	STABILITY ANALYSIS		4-13-87	4-29-87	ME	0	70	70	70	0
8	T08 0	ASEMBLY DRWG PREPARAT	FION	3-02-87	5-01-87	ME	0	817	817	B17	0
	T126	UPDATE TDP		3-02-87	5-01-87	ME	0	484	484	484	0
8	T160	LONG LEAD ITEMS		3-02-87	4-15-87	ME	0	60	60	60	0
273	P02 0	CAMMON 11200			•						
***	A 010	MECHANICAL ENGRG 350212 31011									
2	T040	DRWG PREPARATION		3-02-87	3-20-87	ME	0	128	128	128	0
	P03 0	CARRIAGE 11300									
No.	A 010	ADV NFG-ORGANIC COMPO 350213 31071	SITE								
X	T040	NFG REVIEW OF COMPOS	ITES	3-02-87	5-01-87	MF	0	200	200	200	0

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MAR 02 87 ACTIVITY STATUS - ACTUAL VS PLAN PROJECT: LTHD REMAINING PHASE II

PAGE 2 FILE: PHASE2

				Si	tart	F	nd		*****	4 :	ork Effort ()	lrs)	
	·	Sta	itus CP	Planned		_	Rev/act	Mho	Actual () Fost (=	Estimate		
	A02 0	CEL-STRUCTURAL ANALYSIS 350211 31031										•	
	T040	CEL/RODAMACHER		3-02-87	1	4-15-87	•	CE	0	0	0	0	0
	A0 30	MECHANICAL ENGRG 350213 31031											
	T04 0	MANIFOLD/HL DESIGN		3-02-87	1	4-06-87	•	ME	0	647	647	647	0
	T08 0	MANIFOLD/HL DRWG PREP		3-02-87	7	5-08-87	,	ME	0	567	567	56 7	0
	T120	HYDRAULIC SYS DESIGN		3-02-87	•	4-30-87	,	ME	0	493	493	493	0
	T160	HYDRAULIC SYS DRWG PREP		3-02-87	7	5-08-87	,	ME	0	430	430	430	0
	T200	LOAD TRAY & WAY DRWG PREF	•	3-02-87	1	5-01-87	,	ME	0	646	646	646	0
	T24 0	TRAIL DESIGN		3-02-87	7	4-30-87	1	ME	0	180	190	190	0
***	T280	TRAIL DRWG PREP		3-02-87	,	5-08-87	,	ME	0	586	586	586	0
	T320	PLAT, GIMB, SPADE DESIGN		3-02-87	7	4-10-87	,	ME	0	192	192	192	0
	T360	PLAT,GIMB,SPADE DRWG PREF	1	3-02-87	,	4-24-87	,	ME	0	591	59 1	59 1	0
X						2		_	•			•.•	·
	T400	CRADLE DESIGN		3-02-87	7	4-30-87	1	ME	0	120	120	120	0

		02 87 DJECT: LTHD	ACTIV Remain	ITY STATUS ING PHASE	S - ACTUAL II	. VS	PLAN	FILE	PAGE: PHA		
			Status CP		Planned Rev/act	Nho	Actual (+) To-Date - To	Fcst (=)	Latest (vs.	Orig (=)	Ahead
	T440	CRADLE DRWG PREP		3-02-87	5-08-87	ME	0	100	100	100	0
	T480	SMALL PARTS DESIGN		3-02-87	4-15-87	HE ,	0	42	42	42	0
	T520	SMALL PARTS DRWG PREP		3-02-87	5-08-87	HE	0	65	65	65	0
	T56 0	WEIGHT REDUCTION		3-02-87	4-15-87	ME	0	575	575	575	0
	A04 0	NON-STRUCT MECHANICS 350213 31021									
	T040	N S ANALYSIS		3-02-87	4-01-87	AE	0	130	130	130	0
	A0 *	ORGANIC COMPOSITES 350213 31011									
	T040	COMPOSITE ANAL/DESIGN		3-02-87	4-30-87	30	0	684	684	684	0
	A0 60	STRUCTURAL MECHANICS 350213 31041									
2	T040	S M AMALYSIS		3-02-87	5-08-87	AE	0	925	925	925	0
	A 070	YORK INDUSTRIES 350213 31061									
	T040	YORK AMAL/DESIGN/DRWG	S	3-02-87	5-01-87	YO	0	0	0	0	0
	P04 0 I	FIRE CONTROL 11400									



A010 NECHANICAL ENGRG 350214 31011



MAR 02 87 ACTIVITY STATUS - ACTUAL VS PLAN PAGE 4
PROJECT: LTHD REMAINING PHASE II FILE: PHASE2

4 2															
					Start								t (Hrs)		
			Status	CP	Planned	Rev/act	Planned	Rev/act	Who			=) Latest (v: Estimate		(=) Ahead (Behind	
1	040	FIRE CONTROL DRWG PR	EP		3-23-87	1	5-01-87	,	ME	0	449	449	449		
P	050	QA/QC/TEST 12100													
8 A	010	QUALITY ASSURANCE 350211 31021													
8	040	QUALITY PROGRAM PLAN			3-02-87		5-01-87		QA	0	80	80	80		
P P	060	ACCEPTANCE TEST 12200													
A S	010	TEST 350222 31011													
	040	ID TEST VENDOR			3-02-87	•	4-17-87		TE	0	81	81	81		
		TEST EXPENSE 12300													
	010	TEST COMPOSITES													
Ī	040	SUPPORT COMPOSITE TE	ST		3-02-87	•	4-30-87	,	ME	0	150	150	150		
P	080	SYSTEMS ENGRG 13100													
	1010	SYSTEMS ENGRG 350231 31011,21,31	,41												
ξ. τ Χ	040	SPP, REL, TEST, SPECS, PI	HA		3-02-87	1	5-01-87	,	SY	0	323	323	32 3		
Ç P	090	PROJECT MANAGEMENT 13200													
*	1010	MECHANICAL ENGRG 350232 31031													
۲ *	; -	PROJECT COORDINATION			3-02-87	1	5-11-87	,	ME	0	200	200	200		

MAR 02 87 ACTIVITY STATUS - ACTUAL VS PLAN PAGE 5
PROJECT: LTHD REMAINING PHASE II FILE: PHASE2

PAGE 5

					St	art	End			Mn	rk Effort (Hr	el	
_			Status	CP	Planned	Rev/act	Planned Rev		o Actual (Latest (vs)		(=) Ahead (Behind)
						-			TO-Bate	10-cap1	Cationt	LTON	(Sening)
	A 020	PROJECT MANASEMENT 350232 31011											
	T040	PROJ MANAGEMENT			3-02-87	•	5-11-87	Pff	0	80	80	80	0
	A030	TRAVEL NOD/CEL 350232 31021											
_	T040	TRAVEL			3-02-87	,	3-02-87	x	0	0	0	0	0
2	A04 0	NODEL											
	T040	HODEL NAKING			3-02-87	,	3-31-87	1	0	0	0	0	0
	P100	DATA PREPARATION 14100											
	A.	MECHANICAL ENGRG 350241 31011											
	T 04 0	COMPILE REPORTS			4-06-87	•	5-01-87	ME	0	448	448	448	0
	P110	GFE SUPPORT 15100											
	A 010	MECHANICAL ENGRG 350251 31011											
	T040	GFE COORDINATION			4-06-87	,	4-24-87	ME	0	46	46	46	0
	P120	SPARE PARTS 15400											
	A 010	MECHANICAL ENGRG 350254 31011											
.57	T040	COMPILE PARTS LIST			3-02-87	•	5-01-87	ME	0	90	90	90	0

MAR 02 87 ACTIVITY STATUS - ACTUAL VS PLAN PAGE 6
PROJECT: LTHD REMAINING PHASE II FILE: PHASE2

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Status CP Planned Rev/act Planned Rev/act Who Actual (+) Fcst (=) Latest (vs) Grig (=) Ahead To-Date To-Cepl Estimate Plan (Behind)

THE PROPERTY OF THE PROPERTY O

TOTAL PROJECT 3-02-87 5-11-87 ALL 0 11,079 11,079 0

MAR 02 87 ACTIVITY STATUS - ACTUAL VS PLAN PROJECT: LTHD REMAINING PHASE II FILE

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PAGE 7 FILE: PHASE2

•			St	art	E	nd			No	rk Effort	itrs)	
RL	Status	CP	Planned	Rev/act	Planned	Rev/act	Mho	Actual (+ To-Date) Latest (Estimate	rs) Orig Plan	(=) Ahead (Behind)
NAMUFACTURING ENGR MECHANICAL ENGR CEL							HF ME CE	0	600 8,176 0	60 0 8, 176	600 8,176	0
AMALYTICAL ENGR COMPOSITE ENGR YORK PESONNEL							AE OC YO	0	1,055 684 0	1,055 684 0	1,055 684 0	0
 OUALITY ENGR TEST ENGR Systems Engr Proj Manager							DA TE SY PH	0 0 0	80 81 323 80	90 91 323 80	80 81 323 80	0 0 0
UNASSIGNED TOTAL PROJECT			3-02-87	,	5-11-87		X All	0	0 11,079	0 11,079	11,079	0

DESCRIPTION: LONG LEAD ITEM SCHEDULE

STATUS: A schedule of long lead items was developed from cost and lead time information from vendors supplying long lead time parts.

The schedule is current and complete.

AUTHORS: Bart Anderson, Dave Boudreau and Scott Dacko

FMC LITHD

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LONG LEAD ITEM SCHEDULE

PAGE 1

PROJECT: LTHD LONG LEAD ITEMS/SCHDL

FILE: LTHDLL

	LI JNG LEAD ITEMS/SCHI	,,	H	Mho									LHC. L	MOY	Dec	Jan	Feb	RAF	Apr	Пау
	1 MARKET 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1				Feb		+	··		+					—			-		·····
	INCOMING/IMPROCESS INSP					1	•	•	•	•	• •	•	•	•	,	• •	•	' '		1
	SHIP TRAIL BULKHD/FITTING						•	•	•	•	• •	•	•	•	• •	• •	•	•	• •	11
	WHEEL BULKHEAD	5846					•	•	•	•	• •	•	•	•	•	• •	•	•	• •	11
3	BETAILING		0	X		1		• •	•	•	• •	•	•	•	•	•	•	• •	• •	11
	MATERIAL PROCUREMENT FABRICATING & MELDING	j	1	Ŷ	•		•	•	•				•	•	•	•	•	•	•	11
	MACHINING & MELDING		0	,			•		•	•		•	•	•	•	• •	•	'	•	11
8	SHIP		0	Ŷ	١.	1	•	•	•	•	• •	•	•	•	•	• •	•	'	• •	ĸ
	BIMBAL/PLAT/SPADE ASSY	5710-425		^			•	•	•	•		•	· •		'		•	' '		11
	SIMBAL	5811			1	-			•	•	•						•			11
	DETAILING	55.	0	ly l) 2001				•										.\$
A.	MATERIAL PROCUREMENT		Ŏ	Ŷ		1			. 226	•										. [[
	FABRICATING & WELDING		ŏ	Y						22222	· ·									.#
2	MACHINING		٥	y !	1												•			11
	ASSEMBLY		١٥	Ŷ					•	•							•			11
	PLATFORM	5801	ľ	[1	•										•			11
ì	DETAILING	0007	0	Y		(2006			•								•	'		11
Š	MATERIAL PROCUREMENT			Ŷ	║ .	1		•	•	•	• •	•	•	•	,	•	•	'	•	11
K	FABRICATING & WELDING		ŏ	 	11 .	1				•	• '	, ,	,	'	•	•	•	'	•	11
	MACHINING		0	1	11 .					•	•	'	'	'	•	•	•		•	11
ř	ASSEMBLY		1	Ŷ	11 .	1	•	•	•	•	•		•	, ,	•	•	•	, ,	•	11
Çi L	SPADE	5821	ľ		ll '		•	•	•	•	•	_		'	•		•		•	11
-	DETAILING	JUL.	١٨	x	11 '	1		•	•	•	• '	' '	'	'	•	•	•		• '	11
27	RIAL PROCUREMENT		ŏ	Ŷ	Ⅱ .	1		• '	•	•		' '	, ,	•	•	•	•	, ,	• •	11
Į	tmorticating & WELDING		0	Ç I	II .		•	•	•	•		, , 	•	'	•	• •	•	•	•	11
8	MACHINING		0	\ \ \ \	II .	-	•	•	•	•	•		•	'	•	• •	•	•	•	11
	ASSEMBLY		1	Î	11 .	1	•	•	•	•	•			'	•	• •	•	•	•	11
3	BOLTING		ľ	'	11 '	1	•	•	•	•	•			,	•	•	•	•	•	'#
3	MATERIAL PROCUREMENT		١,	x	║ .	1								•	•	•	•	1	•	11
	ASSY LD FTMGS TO CRADLE		ľ	<u> </u>	11 '	1	,							, ,	•	•	•	•	•	"[[
_	CRADLE	5831	1		11 .	1	•	•	•	•	•	•	•	, ,	•	• •	•	•	•	1
ľ	DETAILING	2021	0	ا ا ا	11 .		,		•	•	•	'	•	, ,	•	•		,	•	11
	TOOLING		0	()	11 .	1				•	•	• '		• •	•	• •	•		•	11
	MATERIAL PROCUREMENT		0	Ŷ	11 .		•			•	•		•	•	•	• •	, ,	•	•	1
0	FABRICATING		0	l,	11 .		•			•	-	•	•	•	•	•	• •	•	•	11
8	MACHINING		ň	l x			•	•	• -				•		•	•	'	•	•	'
	ASSEMBLY		0	l _x		1	•		•						•	•	'	•	•	11
_	FRONT CRADLE MANIFOLD	5944	ľ	'	[]	İ	•		•						•	•	•	•		"
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PROJECT: LTHD LONG LEAD ITEMS/SCHDL

FILE: LTHDLL

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PROJECT: LTHD LONG LEAD ITEMS/SCHDL

FILE: LTHDLL

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PROJECT: LTHD LONG LEAD ITEMS/SCHDL

FILE: LTHDLL

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PROJECT: LTHD LONG LEAD ITEMS/SCHDL

FILE: LTHDLL

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	cSTING/PROCEDURES	10	l _x]] .															.11
-	TAIL LIGHT AND WIRING 5710-580			11 .					•							,			.}}
	ASSEMBLY	10)x	1	.]											••			.]]
	SPEED SHIFT ASSEMBLY 5710-575		1]] .	.														.]]
73	ASSEMBLY	0	X	ll .	.														.
	BASIC ISSUE ITEMS & CONTR 5710-200			1	.						,								.]]
	INSTALL	10	x]]	.											88	•		.]]
	MANE PLATES & LOCATION 5710-500	1		11	.											•	•		.]]
	INSTALL	0	X	11	.									•					.]]
	PREPARE TO SHIP]	11															.]]
	PREPARE	0	X	II .	.			•		,							-		.]]
XX.	SHIP TO ARMY TEST SITE	1	l	11	.														. }
	SHIP	10	X	1	.												, 991	١,	.
	INSTL 1ST ART MUZZLE BRK		1	II .	.	•								•		•			.
	ASSEMBLY	10	X	II	.		•								•	•	, 901	١,	.

PAGE ACTIVITY DETAIL MAR 02 87 FILE: LTHDLL PROJECT: LTHD LONG LEAD ITEMS/SCHDL --ID-- STATUS ------MANE------ BUS DAYS START END ------MHO------ Hrs ANOUNT OF LK PTY CAT INCOMING/INPROCESS INSP PHA 006 SHIP TRAIL BULKHD/FITTING PHA 008 WHEEL BULKHEAD 5846 ACT 050 TSK 040 **DETAILING** 25 3-12-87 4-15-87 UNASSIGNED 10 7-08-87 7-21-87 UNASSIGNED MATERIAL PROCUREMENT TSK 120 9-01-87 UNASSIGNED 25 7-29-87 TSK 160 FABRICATING & WELDING 9-10-87 UNASSIGNED **MACHINING** 3 9-08-87 TSK 200 5 9-14-87 9-18-87 UNASSIGNED TSK 210 SHIP SIMBAL/PLAT/SPADE ASSY 5710-425 PHA 010 5811 ACT 010 SIMBAL 26 3-02-87 4-06-87 UNASSIGNED DETAILING TSK 040 10 6-04-87 6-17-87 UNASSIGNED MATERIAL PROCUREMENT TSK 120 FABRICATING & WELDING 24 6-19-87 7-22-87 UNASSIGNED _60 5 7-31-87 8-06-87 UNASSIGNED MACHINING TSK 200 10 9-21-87 10-02-87 UNASSIGNED TSK 240 ASSEMBLY 5801 **ACT 020 PLATFORM** TSK 040 DETAILING 16 3-02-87 3-23-87 UNASSIGNED 12 4-02-87 4-17-87 UNASSIGNED MATERIAL PROCUREMENT TSK 120 TSK 160 FABRICATING & WELDING 4-24-87 6-18-87 UNASSIGNED i c 6-24-87 6-29-87 UNASSIGNED TSK 200 MACHINING TSK 240 **ASSEMBLY** 9-21-87 10-02-87 UNASSIGNED **ACT 030** 5821 SPADE TSK 040 DETAILING 3-02-87 4-15-87 UNASSIGNED TSK 120 MATERIAL PROCUREMENT 10 B-10-87 8-21-87 UNASSIGNED

10 8-31-87 9-11-87 UNASSIGNED

TSK 160

FABRICATING & WELDING

PAGE ACTIVITY DETAIL MAR 02 87 FILE: LTHDLL PROJECT: LTHD LONG LEAD ITEMS/SCHDL --ID-- STATUS -------WAME----- BUS DAYS START END -------WHO------ Hrs AMOUNT CP LK PTY GAT 4 9-14-87 9-17-87 UNASSIGNED **MACHINING** 10 9-21-87 10-02-87 UNASSIGNED TSK 240 **ASSEMBLY** PHA 280 **BOLTING** 132 4-01-87 10-01-87 UNASSIGNED MATERIAL PROCUREMENT TSK 040 PHA 012 ASSY LD FINGS TO CRADLE 5831 CRADLE DE ACT 010 55 3-02-87 5-15-87 UNASSIGNED TSK 040 **DETAILING** 23 5-18-87 6-17-87 UNASSIGNED TOOL ING TSK 080 23 5-18-87 6-17-87 UNASSIGNED MATERIAL PROCUREMENT TSK 120 38 6-23-87 9-13-87 UNASSIGNED TSK 160 **FABRICATING** 8-24-87 9-15-87 UNASSIGNED MACHINING TSK 200 TSK 240 9-21-87 10-02-87 UNASSIGNED **ASSEMBLY** ACT 110 FRONT CRADLE MANIFOLD 3-02-87 5-01-87 UNASSIGNED TSK 040 DETAILING 83 4-13-87 8-05-87 UNASSIGNED TSK 120 MATERIAL PROCUREMENT 8-10-87 9-18-87 UNASSIGNED 30 TSK 200 MACHINING 10 9-21-87 10-02-87 UNASSIGNED TSK 040 ASSEMBLY STR TEST, RETEST 6PSA PHA 014 33 3-02-87 4-15-87 UNASSIGNED TSK 010 PROCURE TEST SITE 32 10-05-87 11-17-87 UNASSIGNED TEST-RETEST TSK 040 PHA 016 STR TEST, RETEST CRADLE 32 10-05-87 11-17-87 UNASSIGNED TSK 040 TEST-RETEST COMPOUND ACTUATOR ASSY 5710-275 PHA 060 **ACT 010** RECOIL CYLINDERS 5945,46,75 40 3-02-87 4-24-87 UNASSIGNED TSK 040 DETAILING

6-08-87 7-31-87 UNASSIGNED

Tr. .40

TOOLING

MAR 02 87 ACTIVITY DETAIL PROJECT: LTHD LONG LEAD ITEMS/SCHDL

PAGE 3
FILE: LTHDLL

	11	STATUS		BUS DAYS	START	END		Hrs AMOUNT CP LK PTY CAT
	75 ₁ .	_0	MATERIAL PROCUREMENT	20	6-09-87	7-06-87	UNASSIGNED	0
	TSK	160	EXTRUDING	5	B-03 -8 7	8-07-87	UNASS I GNED	0
殿	TSK	200	MACHINING	30	8-17-87	9-25-87	UNASSIGNED	0
W tr		240	ASSEMBLY	32	10-05-87	11-17-87	UNASSIGNED	0
12	ACT	020	RECOIL RODS 5949	,50				
RZNi	TSK	040	DETAILING	40	3-02-87	4-24-87	UNASSIGNED	0
	TSK	080	TOOLING	40	6-08-87	7-31-87	UNASSIGNED	0
150	TSK	120	MATERIAL PROCUREMENT	20	6-08-87	7-03-87	UNASSIGNED	0
	TSK	160	EXTRUDING	5	8-03-87	8-07-87	UNASSIGNED	0
	TSK	200	MACHINING	30	8-17-87	9-25-87	UNASSIGNED	0
	TSK	240	ASSEMBLY	32	10-05-87	11-17-87	UNASSIGNED	0
	ACT	030	END CAPS 595	,52				
10 8	İċ.	-40	DETAILING	40	3-02-87	4-24-87	UNASSIGNED	0
	TSI	080	TOOL ING	40	6-08-87	7-31-87	UNASSIGNED	0
921	TSI	120	MATERIAL PROCUREMENT	20	6-08-87	7-03-B7	UNASSIGNED	0 .
	TSI	160	EXTRUDING	5	8-03-87	8-07-87	UNASSIGNED	0
	TSI	200	MACHINING	30	8-18-87	9-28-87	UNASSIGNED	0
37	TS	(240	ASSEMBLY	32	10-05-87	11-17-87	UNASSIGNED	0
	AC	T 040	NUT 595	5				
	TS	C 040	DETAILING	40	3-02-87	4-24-87	UNASSIBNED	0
	TS	K 080	TOOLING	40	6-08-87	7-31-87	UNASSIGNED	0
(?)	TS	K 120	MATERIAL PROCUREMENT	20	6-08-87	7-03 -8 7	UNASSIGNED	0
S. S. S. S. S. S. S. S. S. S. S. S. S. S	TS	K 160	EXTRUDING	5	8-03-87	8-07-87	7 UNASSIGNED	0
	TS	K 200	MACHINING	30	B-17-87	9-25-87	7 UNASSIGNED	0
	TS	K 240	ASSEMBLY	32	10-05-87	11-17-87	7 UNASSIGNED	0
	ľ	`5 0	ACTUATORS					

		R 02 DJECT	87 : LTHD LONG		CTIVI ITEMS				PA FILE: L	AGE 4	
	11) STATUS	NAME		BUS DAYS	START	END		Hrs AM	OUNT CP LK PTY	CAT
	i.	J40	DETAILING		46	3-02-87	5-04-87	UNASSIGNED		0	
		120	MATERIAL PROCUREMENT		13	6-09-87	6-25-87	UNASS16NED		0	
	TSK	200	MACHINING		45	7-06-87	9-04-87	UNASSIGNED		0	
•	TSK	160	ASSEMBLY & TEST		16	9-07-87	9-28-87	UNASSIGNED		0	
	75K	240	ASSEMBLY		32	10-05-87	11-17-87	UNASSIGNED		0	
6	ACT	060 120	CONTROL VALVES								
KS	TSK	120	MATERIAL PROCUREMENT		30	6-01-87	7-10-87	UNASSIGNED		0	
No.	TSK	200	MACHINING		31	7-13-87	8-24-87	UNASSIGNED		0	
W.D	TSK	160	ASSEMBLY & TEST		24	8-25-87	9-25-8 7	UNASSIGNED		0	
200	TSK	240	ASSEMBLY		32	10-05-87	11-17-87	UNASSIGNED		0	
σM	ACT	070	FRONT CRADLE MANIFOLD	5944							
	TSK	040	DETAILING		45	3-02-87	5-01-87	UNASSIGNED		0	
	Ţe"	.50	MATERIAL PROCUREMENT		88	4-13-87	8-12-87	UNASSIGNED		0	
	TSK	200	MACHINING		30	B-17-87	9-25-87	UNASSIGNED		0	
	TSK	240	ASSEMBLY		32	10-05-87	11-17-87	UNASSIGNED		0	
	ACT	080	MID CRADLE MANIFOLD	5943							
	TSK	040	DETAILING		45	3-02-87	5-01-87	UNASSIGNED		0	
		120	MATERIAL PROCUREMENT		88	4-13-87	8-12-87	UNASSIGNED		0	
	TSK	200	MACHINING		30	8-17-87	9-25-87	UNASSIGNED		0	
	TSK	240	ASSEMBLY		32	10-05-87	11-17-87	UNASSIGNED		0	
	act	090	FITTINGS-ROSAN								
	TSK	120	MATERIAL PROCUREMENT		96	5-15-87	9-25-87	UNASSIGNED		0	
M	TSK	240	ASSEMBLY		32	10-05-87	11-17-87	UNASS I GNED		0	
7	PHA	030	TRAIL SUBASSEMBLY	5710-600							
	ACT	010	TRAIL, UPPER	5841,97							
	Ţ	10	DETAILING		45	3-02-87	5-01-87	UNASS I GNED		0	

ACTIVITY DETAIL FILE: LTHDLL PROJECT: LTHD LONG LEAD ITEMS/SCHDL 23 6-29-87 7-29-87 UNASSIGNED ٥ TOOL ING 1.. .40 MATERIAL PROCUREMENT 23 6-29-87 7-29-87 UNASSIGNED TSK 120 38 7-27-87 9-14-87 UNASSIGNED TSX 160 **FABRICATING** 17 9-21-87 10-13-87 UNASSIGNED TSK 200 MACHINING 8 10-22-87 11-02-B7 UNASSIGNED TSK 240 ASSEMBLY TRAIL, LOWER FRONT 5843,99 **ACT 030** 45 3-02-87 5-01-87 UNASSIGNED DETAILING TSK 040 23 6-29-87 7-29-87 UMASSIGNED TSK 080 TOOLING MATERIAL PROCUREMENT 23 6-29-87 7-29-87 UNASSIGNED TSK 120 38 7-27-87 9-16-87 UNASSIGNED TSK 160 **FABRICATING** 17 9-22-87 10-14-87 UNASSIGNED MACHINING TSK 200 8 10-22-87 11-02-87 UNASSIGNED TSK 240 ASSEMBLY AP: 140 5933,34 LATTICE 45 3-02-87 5-01-87 UNASSIGNED n TSK 040 DETAILING 23 6-29-87 7-29-87 UNASSIGNED TSK 080 TOOLING 23 6-25-87 7-27-87 UNASSIGNED TSK 120 MATERIAL PROCUREMENT 38 7-27-87 9-16-87 UMASSIGNED TSK 160 **FABRICATING** 17 9-21-87 10-13-87 UMASSIGNED TSK 200 MACHINING TSK 240 8 10-22-87 11-02-87 UNASSIGNED **ASSEMBLY** CANNON SUBASSY-NO MZ BRK 5710-250 PHA 120 ACT 010 5963,64 RAILS 35 3-02-87 4-17-87 UNASSIGNED TSK 040 **DETAILING** TSK 080 TOOL ING 40 7-13-87 9-04-87 UMASSIGNED TSK 120 MATERIAL PROCUREMENT 20 7-14-87 8-10-87 UMASSIGNED TSK 160 5 9-09-87 9-15-87 UMASSIGNED **EXTRUDING** 7)0 30 9-21-87 10-30-87 UNASSIGNED

MAR 02 87

MACHINING

PAGE

MAR 02 87 ACTIVITY DETAIL PROJECT: LTHD LONG LEAD ITEMS/SCHDL

PAGE 6

	10-	- STATUS	NAIE	BUS DAYS	START	END	 	Hrs AMOUNT CP LK PTY	CAT
	rs	.40	assenbly	24	11-03-87	12-04-87	UNASSIGNED	0	
	ACT (COLLARS 5781	-III					
	TSK (040	DETAIL ING	33	3-02-87	4-15-87	UNASSIGNED	0	
	TSK	080	TOOLING	40	7-13-87	9-04-87	UNASSIGNED	0	
	TSK	120	MATERIAL PROCUREMENT	20	7-14-87	B-10-B7	UNASS I ENED	0	
			FORGING	5	9-09-87	9-15-87	UNASSIGNED	0	
×	TSK TSK	200	MACHINING	30	9-21-87	10-30-87	UNASSIGNED	0	
	TSK	240	ASSEMBLY	24	11-03-87	12-04-87	UNASSIGNED	0	
25	ACT		CAMBION 5767						
	TSK	120	MATERIAL PROCUREMENT	60	2-24-87	6-15-87	UNASSIGNED	0	
	TSK	240	ASSEMBLY	24	11-03-87	12-04-87	UNASSIGNED	0	
	ACT	030	CLAMP PLATE 596	,					
262	Ţŗ	`4 0	DETAILING	44	3-02-87	4-30-87	7 UNASSIGNED	0	
	! TSK	080	TOOLING	40	7-13-67	9-04 -8	7 UNASSIGNED	0	
S.	TSK	120	MATERIAL PROCUREMENT	20	7-14-87	8-10-8	7 UNASSIGNED	0	
	TSK	160	EXTRUDING	5	9-09-87	9-15-8	7 UNASSIGNED	0	
	TSK	200	MACHINING	30	9-21-87	10-30-8	7 UNASSIGNED	0	
	TSP	240	ASSEMBLY	26	11-03-8	7 12-04-8	7 UNASSIGNED	0	
8	PH	124	STR TEST TRAILS						
_	TSI	C 040	TEST TRAILS	31	8 11-03-8	7 12-24-8	7 UNASSIGNED	0	
	PH	A 070	COMPOUND ACT ASSY/CRADLE 5710)-260					
EX.	TS	K 240	ASSEMBLY		7 11-20-8	7 11-30-6	T UNASSIGNED	0	
		A 020	CRADLE ASSY GIMB/PLATF 571	0-340					
	TS	K 040	ASSEMBLY		7 11-20-8	7 11-30-	B7 UNASSIGNED	0	
7.5	Pi	IA OBO	ALIGN WAYS TO BE/AZ AXIS 571	0-175					
K.	- •	740	ASSEMBLY		7 12-01-6	17 12 -09 -	87 UNASSIGNED	0	
_									

MAR 02 87 ACTIVITY DETAIL PROJECT: LTHD LONG LEAD ITEMS/SCHDL

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FILE: LTHDLL

	10 STATU	SNAME		BUS DAYS	START	END	HH()	Hrs ANOUNT- CP LK PTY	CAT
	Pi J90	FIRE CONTROL (STATIC) 5	710-400					·	
	ACT 010	FIRE CONTROL LINKS 5	646,49,50						
	TSK 060	DETAILING		33	3-05-87	4-20-B7	UNASSIGNED	0	
-	TSK 105	TOOLING		23	8-10-87	9-09-87	UNASSIGNED	0	
	TSK 138	MATERIAL PROCUREMENT		23	8-10-87	9-09-87	UNASSIGNED	0	
N	TSK 163	FABRICATING		38	9-14-87	11-04-87	UNASSIGNED	0	
N. C.	TSK 182	MACHINING		17	11-09-87	12-01-87	UNASSIGNED	0	
8	TSK 240	ASSEMBLY		5	12-07-87	12-11-87	UNASSIGNED	0	
0.5	PHA 150	INSTALL PSA MUZZLE BRAKE							
	ACT 010	MUZZLE BRAKE	5766						
	TSK 040	DETAILING		1	2-27-87	2-27-87	UNASSIGNED	0	
X	TSK 080	PATTERN		35	4-27-87	6-12 -8 7	UNASSIGNED	0	
	Tr .50	CASTING PROCESS		97	6-15-87	10-27-87	UNASSIGNED	0	
	TSK 200	MACHINING		25	11-02-87	12-04-87	UNASSIGNEB	0	
	TSK 240	ASSEMBLY		14	12-07-87	12-24-87	UNASSIGNED	0	
NI.	PHA 160	LDADTRAY AND WAY	5710-475						
	TSK 240	ASSEMBLY		14	12-07-87	12-24-87	UNASSIGNED	0	
	PHA 170	FIRE CNTROL & OPT COMPL	5710-400						
	TSK 060	SCOPE PROCUREMENT	5970	163	5-01-87	12-15-87	UNASSIGNED	0	
An.	TSK 240	ASSEMBLY		9	12-14-87	12-24-8	7 UNASSIGNED	0	
8	PHA 130	CAMMON SUBASSY TO CRADLE	5710-240						
	TSK 240	ASSEMBLY		7	12-14-87	12-22-8	7 UNASSIGNED	0	
	PHA 100	HYDRAULIC PIPING NETWORK	5710-465						
	ACT 010	ACTUATORS							
Å4	TSK 040	DETAILING		45	3-02-97	5-01-8	7 UNASSIGNED	0	
	T 70	MATERIAL PROCUREMENT		15	8-03-87	8-21-8	7 UNASSIGNED	0	

FILE: LTHDLL PROJECT: LTHD LONG LEAD ITEMS/SCHDL --ID-- STATUS ------NAME------ BUS DAYS START END ------NHO-----Hrs AMOUNT CP LK PTY CAT 55 8-25-87 11-09-87 UNASSIGNED **MACHINING** 1... 200 19 11-16-87 12-10-87 UNASSIGNED ASSEMBLY & TEST TSK 160 9 12-14-87 12-24-87 UNASSIGNED TSK 240 ASSEMBLY CONTROL VALVES ACT 020 44 6-09-87 8-07-87 UNASSIGNED MATERIAL PROCUREMENT TSK 120 57 8-17-87 11-03-87 UNASSIGNED **HACHINING** TSK 200 23 11-09-87 12-09-87 UNASSIGNED ASSEMBLY & TEST TSK 160 9 12-14-87 12-24-87 UNASSIGNED ASSEMBLY TSK 240 FITTINGS ACT 030 71 8-31-87 12-07-87 UNASSIGNED MATERIAL PROCUREMENT TSK 120 9 12-14-87 12-24-87 UNASSIGNED ASSEMBLY TSK 240 TRAIL ASSY TO SYSTEM 5710-595 PHA 030 20 12-15-87 1-11-88 UNASSIGNED Tr: 140 ASSEMBLY WALKING BEAM WHLS & AXLE 5710-625 PHA 180 5748 ACT 010 DISC BRAKE ROTOR 1 2-25-87 2-25-87 UNASSIGNED DETAILING TSK 040 40 8-31-87 10-23-87 UNASSIENED TOOLING TSK 080 20 8-31-87 9-25-87 UNASSIGNED MATERIAL PROCUREMENT TSK 120 5 10-27-87 11-02-87 UNASSIGNED TSK 160 EXTRUDING 30 11-10-87 12-21-87 UNASSIGNED MACHINING TSK 200 4 12-28-87 12-31-87 UMASSIGNED TSK 240 ASSEMBLY 5710-650 MALKING BEAMS PHA 190 LEAD & LAGGING BEAMS 5732,3,92,3 **ACT 010** 11 3-02-87 3-16-87 UNASSIGNED DETAILING TSK 040 30 7-27-87 9-04-87 UNASSIGNED TOOL ING TSK 080 50 7-27-87 10-02-87 UNASSIGNED MATERIAL PROCUREMENT .20

ACTIVITY DETAIL

PAGE

MAR 02 87 ACTIVITY DETAIL PAGE FILE: LTHDLL PROJECT: LTHD LONG LEAD ITEMS/SCHDL 45 10-12-87 12-11-87 UNASSIGNED FABRICATING & WELDING 75 20 10 12-14-87 12-25-87 UNASSIGNED TSK 200 MACHINING 4 1-05-88 1-08-88 UNASSIGNED N3: TSK 240 ASSEMBLY 5803 **ACT 020** CROSS SUPPORT 11 3-02-87 3-16-87 UNASSIGNED TSK 040 **DETAILINS** 30 7-27-87 9-04-87 UNASSIGNED TSK 080 TOOLING 50 7-27-87 10-02-87 UNASSIGNED MATERIAL PROCUREMENT TSK 120 5 11-19-87 11-25-87 UMASSIGNED FABRICATING & WELDING TSK 160 5 12-09-87 12-15-87 UNASSIGNED TSK 200 MACHINING 4 1-05-88 1-08-88 UMASSIGNED ASSEMBLY TSK 240 HYDR SYS STRTUP & CHKOUT 5710-470 PHA 140 15 1-05-88 1-25-88 UMASSIGNED STARTUP/CHECKOUT TSK 040 MALKING BEAMS TO TRAIL 5710-675 Pu" 200 5 1-12-88 1-18-88 UNASSIGNED TSK 240 ASSEMBLY PHA 210 OPER INSIDE TEST/PROCED 37 1-12-88 3-02-88 UNASSIGNED TESTING/PROCEDURES TSK 040 HOR PIPING NIME IN TRAILS PHA 214 7 1-15-88 1-25-88 UNASSIGNED TSK 040 PIPING IN TRAILS 5710-225 BRAKE SYSTEM PHA 216 7 1-15-88 1-25-88 UNASSIGNED TSK 240 ASSEMBLY OPER OUTSIDE TEST/PROCED PHA 218 32 1-19-88 3-02-88 UNASSIGNED TSK 040 TESTING/PROCEDURES TAIL LIGHT AND WIRING 5710-580 PHA 220 10 1-19-88 2-01-88 UNASSIGNED TSK 240 ASSEMBLY SPEED SHIFT ASSEMBLY 5710-575 PHA 230

10 1-19-88 2-01-88 UMASSIGNED

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ASSEMBLY

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22	-ID- STATUS	NAHE	BUS DAYS	START	END	WHO	Hrs AMOUNT CP LK PTY CAT
	140	BASIC ISSUE ITEMS & CONTR 5710-200				·	
305	TSK 040	INSTALL	6	1-26-88	2-02-89	UNASSIGNED	0
	PNA 250	NAME PLATES & LOCATION 5710-500					
W.	TSK 040	INSTALL	10	2-02-88	2-15-88	UNASSIGNED	0
60	PHA 254	PREPARE TO SHIP					
	TSK 040	PREPARE	15	2-16-88	3-07-88	UNASSIGNED	0
יטנ	PHA 270	SHIP TO ARMY TEST SITE					
	TSK 040	SHIP	9	3-09-88	3-21-88	UNASSIGNED	0
	PHA 260	INSTL 1ST ART MUZZLE BRK					
	TSK 240	ASSEMBLY	12	3-09-88	3-24-88	UNASSIGNED	0

2-24-87 3-24-88

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TOTAL PROJECT

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DESCRIPTION: KNOWLEDGE BASE AND LTHD FILE INDEX

STATUS:

The primary sources of information or references for the LTHD project and their respective status' are as follows:

Final Report. This document details the approach, methods, accomplishments and conclusions of the LTHD project. See the Table of Contents of Volume A for a concise description of the report. The report is current and complete.

LTHD Project Files. These documents, to be stored at FMC through the summer of 1989, are indexed by the knowledge base shown in this section (as marked up). In addition to the information indexed, the LTHD files also include:

- 1. LTHD Final Report Master
- 2. FMC PC files (on floppy disk)
- 3. FMC VAX and Cyber files (on tape)
- 4. FMC CAD files (on tape)
- 5. York CAD files (on tape)
- 6. 1/12 scale models of major components

AUTHORS: Bart Anderson, Scott Dacko

Jeitsk war f

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PARTICULAR CONTROL OF THE PARTICULAR CONTROL

ENTER TRAINMENT

DIRECTORY

FILE: LIBMINEIGHT TONED HOWITZER

address... ABVANCEB CÓMPOSITE PROBUCTS; EAST HAVEN, CT

employs...CARUSO/ RICK

phone....203-449-4647 into loc..File: LiGHIWEIGHT TOWED HOWITZER

supplies..FDAM

Into loc.... PILES LIBHTWEIGHT TONED HOWITZER ALENET ALGME

address...COLUMBUS, DH / eeploys...FUJI, SAM /(CLOSEB LOOP TRAMSMISSIOMS) eeploys...FUJI, SAM /(CLOSEB LOOP TRAMSMISSIOMS) eeploys...HORMIS, AL (APPLICATIONS HANAGER)

BABE

LTHD KNOWLEDBE

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phone......(414) 4537-9353
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abstract...BUM TUBE DESIGN per.......DOUG PAGE see also..CANNONS AMCP-706-251
abstract..RECOIL SYSTEMS
per.....DOUG PAGE ANCP-706-252

AMCP-706-342 into loc..File: LIGHTWEIGHT TOWED HOWITZER

AMERICAN MOLD phone....571-8642

LTHD Disk 11/File . DIR

Page 2

(THD Disk 51/5:38 - DIR

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ARM AMALYBIS - DEMEMAL IMFO

BEATFACT..BATA GEMEMATION PROCESS, DESCRIPTION OF ALL BASIC COMPUTER FILES

Info loc., File: Librimeicht Tower Moutter
     -- 7100
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                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                           eddress...CORCMELLA, CA; 92234.
esploys...Erambsem, chris; Director of Marketing; Similar to Oman's
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370 RICHARDHINE ROAD; WHARTON, M.J 07885
phene.....201-989-007!
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ABUTTACE..SPECS, GUESTIONS, ANSWERS, SCHEMATICS
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2 industrial ave.; Lowell, MA 01851
phone......617-454-5790
LTHD KNOWLEDBE BABE....
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                      phone.....THI, 710-343-1284
espioys....BRUNER, LAMRENCE (LARRY);
RSP - PROBUCT APPLICATIONS
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recommends.FIDER IMMOVATIONS
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//1578880 (1199 GENERAL DRAWINS)
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//157723 (1199 BAREL ASSENLY)
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cocont...EDAMED, NAI W.

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cddress...NAI W. EDAMES; 1351 K BIREET MB; $UITE 820; MASHIMBION DC;
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eaploys...Ebahafib, Frame
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SEE ALSO..LING AMALYSIS - RECOIL ORIFICE
SEE ALSO..ARR AMALYSIS - RECOIL
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LIND KNOWLEDBE BABE..
                                                                                                                        Address... NATERIALS TEST LABORATORY,
MATERIORN, MASS.: 02:72
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eap......SAN JOSE ARPORT TURN RIGHT; FOLLOW MINDSHE ROAD, LEFT AT
INTERSECTION, LEFT AGAIN AT HEIT INTERSECTION, MOW YOU'RE ONE
17 MORTH; GO ABOUT FIVE HILES TO MONTAGUE, EIT RIGHT THEN TAKE
LEFT FORM (YOU'L, 60 DOVER 17 MORTH; HOTEL 19 ON YOU'R RIGHT (MM
CORNER OF 17 AND HONTAGUE).
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BROUND HARLAND 12005
BROUND HARLAND 12005
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used for..BALLISTES RESEARCH LAB
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esploys...BAVIS, RALPH; TECN DIRECTOR; 703-783-312;
esploys...BUTECLM, RAY, PLANT MANABER
edploys...INAMEMBRASK
esploys...RAMEMBRASK; MANEWDRA; PRODUCT MANABER
esploys...RAMEMBRASK; MANABER
esploys...ROTAINS; BILLY, PLANT MANABER
phone....402-444-6211; extension 4300 for FACB
address...437A BOULDER POINT BRIVE; MANCHESTER, NO 63022
esploys...RAMEMBRASK; MANGHESTER, NO 63022
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into loc., File: Lightweight towed Howitzer
phone....927-7874
supplier..Models
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L'THD KNOWLEDBE BABE....
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essigys...#ACCGM, BALE; BEVELOPHENT-CHIEF OF ANTILLERY SUPPORT; CARRON
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cement...THREE MAJOR DIVISIONS ARE RESEARCH, BEVELOPHENT AND
EMBINEERING SUPPONT
employs...2 AMBEA, BR. GIULANG (8); RESEARCH-CHIEF
phone...246-5964
employs...f18CELLA, RUBB
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espiryed., RUMMEL, PAUL, BIRETTOR OF BENET (RETIRED)
espirye...RUMMELLANS, REES HODEL SHOP
espirye...REES, BR. JOHN, 318-264-42415315
espirye...WEES, PR. JOHN, 318-264-42415315
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esploys...HAJOR DARFIELD, 14800, 13023
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                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                       phone....TELECOPY OFFICE, 614-424-7895
phone....TELECOPIER, 614-424-5263
             LITHD KNOWLEDBE BASE..
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espinys... AYBER, ART, OVERALL TECH COGNOMATOR FOR CEL
espinys... ANESE CC, 400-289-2224
espinys... ANESERICH, JIN, RECHARICS LEADHAN, FOUR DAR LINKABES.
free.......... WARTT SAN JOSE CIBA-GELGY BAR.....SOUTH ON 17 TO COLEMAN, WEST ON COLEMAN TO 1105. address... FMC CENTRAL ENGINEERING; 1183 COLEMAN AVE; BDX 580; broader...HBN STREWGTH-TO-WEIBHT HATERIALS
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epinys...abscone, Bill, Soop at PRODUCTION ASPECT OF COMPOSITES
supplies..COMPOSITES FABRICATION espioys...VERNIE, HERB, ENGR FOR MILITARY OFFROAD APPLICATIONS phone.....216-796-2121 PHONE ... STEPHENS, JOHN; 301-278-58755-7449 ... ALECTOP, 301-278-3193 SUPPLIEST. HODELS 120-5472 PHONE IN CAR HARFISCURE Row 30; R address...3075-84th LAME, WE! BLAIME, MN 53432 phone....780-338 or 780-3447 espioys...RE[LAME, BEMMY, PRESIDENT phone.....845-7349 (HOME) C. FILE: LIGHTWEIGHT TOWED HOWITZER Info loc. KILE: LIGHTWEIGHT TOWED HOWITZER KNOWLEDBE BABE. MIGH STRENGTH-TO-MEIGHT HATERIALS Anerramer. BATTEKING PROJECT HELKSOLV from BUAD CITY AIRPORT HEB.....KELSEY-HAVES narramer.. COMPOSITES LTHD B19# 11/F110 - DIR MAYES INDUSTRIAL BRAKE serving...RiA PPT SMGG BOODYEAR HARDLD'S HERCULES HEATH TECHNA Herwesser Done

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LTHD KNOWLEDBE BASE........Page 16 LTHO AMALYSIS - RECOIL ORIFICE

- BAST-ACT., PROGRAM REC., FORT USED WITH RECOIL FORT TO CONFIGURE ORIFICES,

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ser also., AMALYSIS - RECOIL MANIPULATION, ARM AMALYSIS - GENERAL INFO LTHO ANALYSIS - WOP

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esppiios..BEARINGS P1 80 EAST SEE SISS.. TH 9-2320-280-10, ARMY TECH MANUAL SUPPLIES.. TACON into loc..FILE: LIGHTWEIGHT TOWED HOWITHER AUPPLIES.. BARKES supplies.. WHEELS Info loc..FILE: LIBNINGLORY TONED HUNITER COSSONIS..MORKING MOSES AND SKETCHES into loc..FILE: LIGHTWEIGHT TOWED WOUTZER supplier..AM GENERAL supplier..NOTOR WHEEL info loc.. File: LISHTMEIGHT TOWED MONITIER supplier . . VORK (locking option). used for .. HAYES INDUSTRIAL BRAKE address...BOVER, MJ
employs...RcFabvER, JEFF
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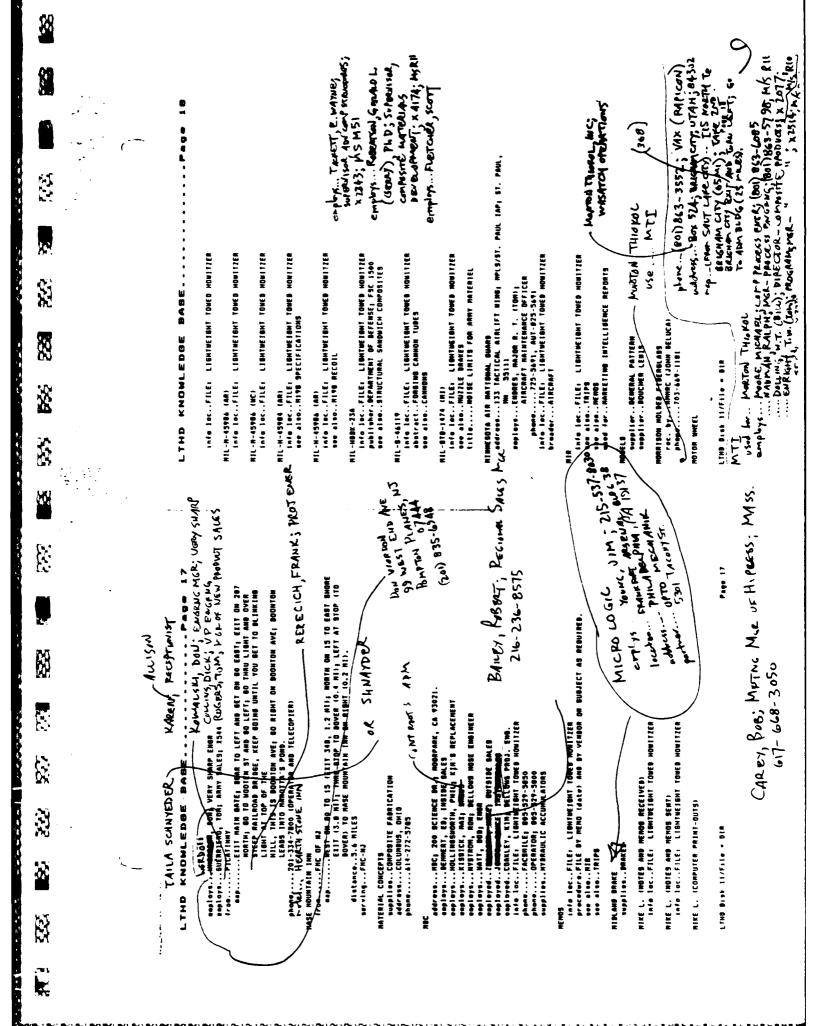
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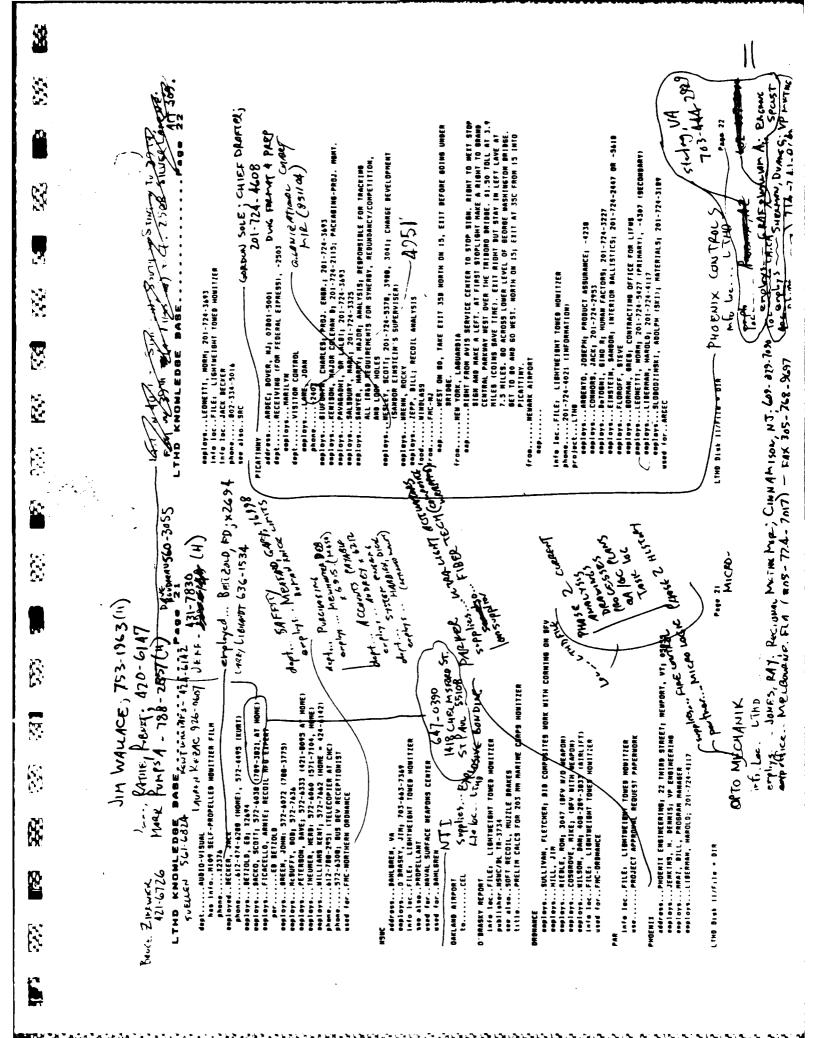
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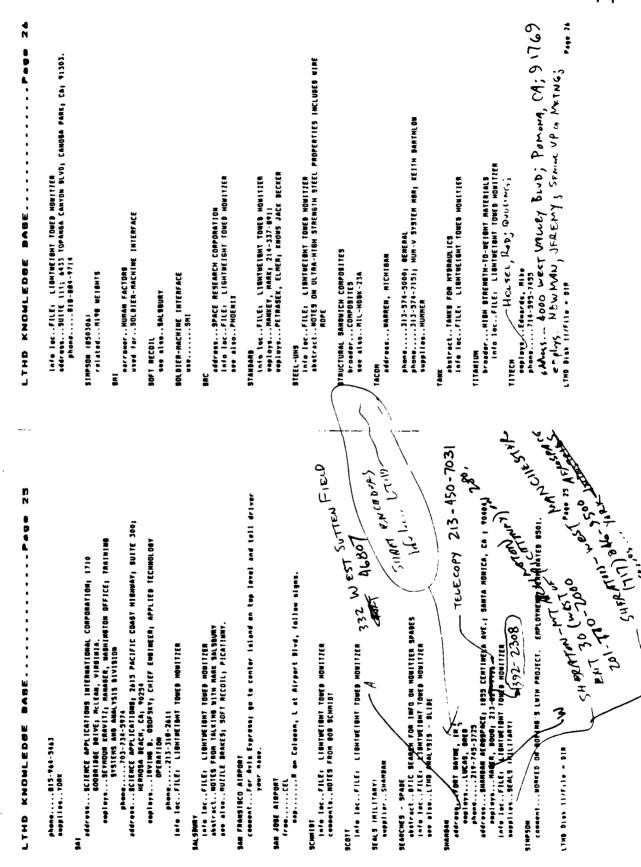
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NOTES: INFORMATION ORGANIZATION AND EXPLANATION OF SYMBOLS

- IMFORMATION WITHIN THE DIRECTORY IS ORGANIZED IN ABBREVIATED SENTENCE FORMAT Subject
- 1.A. THERE ARE NO CONSTRAINTS ON SUBJECT LENGTHS OR CHARACTERS, ELCEPT THAT THE SLASH (1) BE USED ONLY AS SHOWN BELOW.
- 1. B. MIERARCHIAL SUBJECTS ARE HANDLED IN THE FOLLOWING MANNER:

SUBJECT OF SUBJECT/
verb....OBJECT FOR SUBJECT LEVEL 2
LEVEL 3 OF SUBJECT FOR SUBJECT LEVEL 3
verb.....OBJECT FOR SUBJECT LEVEL 3
verb.....OBJECT FOR SUBJECT LEVEL 1

1.C. MULTIPLE LINES OF SUBJECT ARE HANDLED IN THE FOLLOWING MANNER:

SUBJECT LINE 1 SUBJECT LINE 2 SUBJECT LINE 3 verb.....OBJECT

- 1.D. VERDS ARE PRE-ASSIBNED (BEFORE USE) AND EITHER OME-WAY OR TWO-WAY (RECIPROCAL). SEE NOTE 1.J. FOR A LISTING OF THE VERBS IN USE IN THIS BIRECTORY.
- I.E. HIERARCHIAL VERUS ARE HANDLED IN THE FOLLOWING MANNER:

verb.....OBJECT FOR VERB LEVEL 1

verb.....OBJECT FOR VERB LEVEL 2

verb.....OBJECT FOR VERB LEVEL 2

- 1.F. THERE ARE NO CONSTRAINTS ON OBJECT LENBINS OR CHARACTERS.
- 1.8. MULTIPLE LINES OF OBJECT ARE HANDLED IN THE FOLLOWING MANNER:

verb.....OBJECT LINE 1 OBJECT LINE 2 OBJECT LINE 3 "BELE (| Set - . / 0123454881(C-000 ABCOFF GHIJKLMUDGASTUVWEYZ(\))" abcdefghijklenopgretuvweyz(:)"

LIND Disk 11/File - DIR

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2. FILES UNDER BIRECTORY CONTROL (LISTED AT FIRST PAGE OF BIRECTORY);

- 2.A. SHOULD MAVE A LABEL ON THE LEFT MAND SIDE OF THE FIRST FILE OF EACH DRANER THAT: IDENTIFIES THE FILE GROUP (EG, LIGHTMEIGHT TONED MONITZER).
- 2.0. SHOULD MANE A LABEL ON THE RIBHT SIDE OF EACH FOLDER THAT IDENTIFIES THE STARTING LABEL OF CONTENTS OF THAT FOLDER (EG. BENET).
- 2.C. SHOULD MAYE THE CONTENTS OF EACH "PARCEL." OF INFORMATION LABELED AND DATECORDED SUCH THAT EACH PARCEL, IT DEREGUES, CAN BE RETURNED TO 11S EIACT AND IF REFERENCED, CAN BE PRETURNED TO 11S EIACT LABELS THAT PROVIDE SUITABLE LOCATING CODES FOLLOW:

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 BENET (850501) INFORMATION ACQUIRED I MAR 1985, PAGE (

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David Peterson

Namen T. LIONETTI

Harold Liberman

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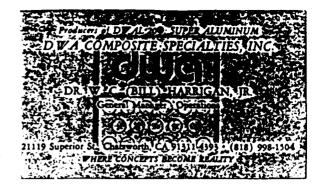
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Adolph "Ski" Slobodzinski	Plastec SMCARAET-O	(201)724-3189 -5859	Materials
Joseph Argento	QA AMSMC-QAH-T	(201)724-4238	Product Assurance
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Marcoim pare	Benet Weapons Lab	(518)266-555 4162,5501	Cannon Configuration
Joe Marvel 3 MAS/MB	Watervliet Arsenal	(518) 26 6-4157	Procurement
Bob McDow	Rock Island Contracting Officer	(309)782-6777	Cannon Contracts
Gerry Cooper	Rock Island	(309)782-6474	Cannon Contracts
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:4	(***)	HOWITZER, 155 MM, TOWEL, MIRE & ASSOCIATED AMMUNITION, THIR ECITION, DOTORES, 1979	PROD FILE	PROJECT	MIRE MANUAL	
15		SANASA DE 10% ASHIOTES MBYSTE MILH HOKIISENS	FRG2 FILE	PROJECT	HOWITZER TON	SENICLES
1:	84-001 (ARM)	ANALYSIS OF FEASIBILITY OF MIRS WEIGHT ROOTH	PROJ FILE	PROJECT	HOWETZER HOF A	4K4_1911
17		SUMMARY OF BIDES COMPUTER SEARCHES	PROC FILE	PROJECT	INFORMATION BE	E450+
12	75-DET-17 (ASMS	FREE FISTON SUBSTITUTE FOR IMPACT DAMPER	PROJ FILE	PROJECT	RECOIL	

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19			SURVEY OF RECOIL ANALYSIS METHODS USED FOR INDUSTRIAL AND AIRCRAFT APPLICATIONS	PROC FILE	PREJECT	RECOIL
20 (AMER 706-042	(ARMY)	DESISN HANDBOOK FOR RECOIL SYSTEMS	PROJ FILE	PROJECT	RECOIL
21		(ASLE)	SURFACE PROFILE REDUCES PECOIL BEARING WEAF	PRO2 FILE	PROJECT	RECOIL BEARINES
22		(ARKY)	COMPRESSIBLE FLUID RECOIL ELIMINATES SEPARATE RECUPERATOR	PROJ FILE	PROJECT	RECOIL
27			COLLECTION OF ARTICLES ON RECOIL FLUIDS	PRGC FILE	PROJECT	RECOID FLUIDS
<u> </u>			USE OF A SERVO SYSTEM TO SIMULATE CANNON FORCES	PROJ FILE	FROUECT	RECOIL CANNON FORCE SIMULATION
je 			OPERATOR AND DESAMILATIONAL MAINTENANCE MANUAL -HOWITZER, LIGHT, TOWER 105 MM SOFT RECOIL (XM204 MDDEL AD)	PROG FILE	PROJECT	RECOIL-SOFT
ī ē			SURVEY OF COMPUTER PROGRAMS AVAILABLE	PROJ FILE	FROJECT	DDMPUTER ANALYSIS
5			ITEMS REQUESTED FROM BUSINESS DEVELOPMENT	PROJ FILE	PROJECT	REQUISITIONS
<u>:</u>		· 	NOTES FROM INFORMATION SEARCH	PROJ FILE	PROJECT	INFORMATION SEARCH
3	E245-1096	(SIDE	TEST PROVES WOMEN CAN RUN MIRB	PROJ FILE	PROJECT	M198 CREW
3:	E700-2400	(DITE	M198 NEEDS NARROWES TIRES FOR LOADING	PROJ FILE	PROJECT	M198 ENVELGEE
72	FT 155-40-0	(45#s	155 MM FIRING TABLES	PROJ FILE	KRAHER	BALLISTICS
::			DESIGN AND DEVELOPMENT OF FIGHTING VEHICLES BY R. M. OSORYJEWHICI, C. 1969, DOUBLEDAY.	STAIERT	STAIERT	BALLISTICS
74			PHOTOGRAPHS OF HOWITZERS	PROC FILE	KRAMER	HOWITZERS
75			LITERATURE REQUESTS (LIBRARY AND J SCHWENCY)	PROJ FILE	PROJECT	REQUISITIONS
J5 (MIL-Y-45985 (AR)	(ARMY)	M45 RECOIL PROCUREMENT SPECIFICATION	PROJ FILE	PROJECT	RECOIL
۔۔۔۔۔ مامنہ		~	PORMACH OF A RESCUL PROGRESSION SPECIALITY	ي بسيونند	(L
78 _/ /	MIL-H-459844 (AR)	CAFAR	M198 PROCUPEMENT SPECIFICATION	PROJ FILE	PROJECT	M198 PROCUREMENT SPEC
: 01			DTIC REPORTS	********	***************************************	DTIC REPORTS
5 00	AD-A009 677	(5715)	RISE ANALYSIS OF MIPS BOTTOM CARRIAGE MADE OF MONOCLITHIC ALUMINUM	PROJ FILE	PROJECT	M198 CARFIASE MEISHT REDUCTION
401	AD TEX TET	(5715)	M198 RECOIL ANALYSIS SHOWS FEAR-TE-AVERAGE FORCE RATIC IS COMPRIMISED BY VARIABLE STROKE.	PROJ FILE	PROJECT	MINE RECOIL RECOIL
470	AS A131 239	([710)	US ARMY TEST AND EVALUATION PROCEDURES FOR RECOILLESS RIFLES	PROJ FILE	PROJECT	RECOILLESS

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NUMBER	CALL NUMBER (SOURCE)	DESCRIPTION	LOCATION	OWNER	KEYWORD(S)
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400		(3110)	NOTES ON THE WEIGHTS OF SUNS, MORTARS, RECOILLESS WEAPONS, AND AMMUNITION	PRO2 FILE	KRAMEF	HOWITZERS
4 0E	AD-A113115	(DTIE)	BLAST FIELD INVESTIGATIONS	PROJ FILE	PROJECT	MUDDLE BRANES
40F		(DT1C)	BORON/ALUMINUM LANDING BEAR FOR NAVY AIRCRAFT	PROJ FILE	PROJECT	COMPOSITES
408		(DT1C)	BRAPHITE COMPOSITE LANDING BEAR COMPONENTS	PROJ FILE	PROJECT	COMPOSITES
404	AD A125 020	(DTIC)	ENERGY ABSORPTION OF COMPOSITE MATERIALS	PRGS FILE	PROJECT	COMPOSITES ENERGY ARROPRIJON
401	AD E400 388	(2710)	DEVELOPMENT OF 105 MM TOWED HOWITZER WITH SOFT RECOIL AND FIRE OUT OF BATTERY	PROJ FILE	PROJECT	REDOIL-BOFT FIRE DUT OF BATTEFY
400	#2 #036 CC	(PTIC)	BRAPWITE COMPOSITE AIRCPAFT LANDING SEAR WHL	PROJ FILE	PROJECT	DOMPGEITES WREELS
41+			PERTINENT MIL STDs			
4;;	(KIL-STE-14748 (MI))		NOISE LIMITS FOR ARMY MATERIEL	PRCV FILE	PROJECT	MUIZLE BR4/EE
42+		(2710)	DTIC INFORMATION SEARCHSE	PROJ FILE	PROJECT	INFORMATION SEARCH
421		(0710)	TECHNICAL REFORT SUMMARN: SHOCK ABSOREEFE	PROJ FILE	PROJECT	RECOL
428		(2712)	TECH REFORT SUMMARY: TRAVERSING MECHANISMS	FRON FILE	PROJECT	TRAVERSINS MECHANISM
420		(2710)	TECH REPORT SUMMARY: ENERGY ABSORBERS	FROD FILE	PROJECT	RECOIL
4 21		(ETIE	TECH REFERT SUMMASN: MOWITZERE	PROC FILE	PROJECT	HOWITZEPE
425		(DTIC)	TECH REPORT SUMMARY: LANGING IMPACT	PROC FILE	PROJECT	RECOIL
427		(BTIC)	TECH REPORT SUMMARY: LANDING SEAR	PROJ FILE	PROJECT	RECOIL
423		(DTIC)	TECH REPORT SUMMARY: SUN BARREL ATTACHMENTS		PROJECT	MUZZLE BRAKES CANNONS
414		(DTIC)		PRGC FILE	FROJECT	APL
17			PHOTOS OF HOWITZER EMPLACEMENTS IN VIET NAM	STATEFT	STATERT	HOWITZER EMPLACEMENT
44+			VENDOR LITERATURE AND MISC INFORMATION	PROJ FILE	PROJECT	VENDOR LITERATURE
44:			LONE STROKE CYLE	PROC FILE	PROJECT	ELEVATING MECHANISMS
445			HYDRAULIC POWER AND PORTABLE MILITARY BENERATORS	PRGJ FILE	PROJECT	AFU
440			TIRES	PROC FILE	PROJECT	
44;			SELF-ALIBNING SELF- LUBRICATING BEARINGS	FROC FILE	PROJECT	REARINGS

NUMBER CA	LL NUMBER (SOURCE)	DESCRIPTION	LOCATION	OWNER	KEYWORD(S)
44E		SHOCK ARSORRERS	PROJ FILE	PROJECT	RECOIL
44F		HYDRAULIC VALVES POTENTIALLY CAPABLE OF HOLDING RECOIL FORCE CONSTANT	PRCJ FILE	PROJECT	RECOIL VALVES
448		POTENTIAL RECOIL BEARINGS	PROJ FILE	PROJECT	RECOIL BEAFINGS
44%		POSITION TRANSDUCERS	PRGJ FILE	PROJECT	POSITION TRANSDUCERS
44;		PANCAKE CYLINDERS	PROJ FILE	PROJECT	CYLINDERS
440		COMPOSITE CYLINDERS	PROJ FILE	PROJECT	COMPOSITES CYLINDERE
44)		MUTILE BRAKES	PROJ FILE	PROJECT	MUIILE BRAYES
44.		ACCUMULATORS	PROV FILE	PROJECT	METAL BELLOWS ACCUMULATORS
44*		AIRCRAFT FOR HOWITZEF TRANSPORTATION	PROJ FILE	PROJECT	MOWITIER AIRLIFT VEHIOLE AIRCRAFT
15±		PROJECT INFORMATION STORED ON DISC	ANDERSON	PROJECT	COMFUTER FILES
452		PROBRAM TO CONVERT RECOIL DUTPUT TO RECOIL TABLE INPUT. NAME = LMTH45A.			
484		INFORMATION ACQUIRED THRU DISCUSSIONS WITH BOS SCHMIST	PROJ FILE	PROJECT	HOWITZERS
£" ŧ		REQUEST FOR INFORMATION PROCEDURES, EXAMPLES, AND NOTES.	PROC FILE	PROJECT	INFORMATION BEARCH REQUEST FOR INFORMATION
47.6		REI DN RECOIL SYSTEMS (ALSO DN DISK, SEE \$45)			RECOIL
475		SPEED SHIFT CYLINDER	PROJ FILE	PROJECT	
48		SUMMARY OF MISB SPECS. ALSO ON DISA, SEE #45	PROJ FILE	PROJECT	MIRE SPECIFICATIONS
4=		LIGHTWEIGHT HOWITZER IDEA LIST	PROJ FILE	PROJECT	CONCERTS
50-4		LIGHTWEIGHT HOWITZER BYSTEM CONCEPTS	PROJ FILE	PROJECT	CONCESTE
50-1		CONCEFT :	PROJ FILE	PROJECT	
50-0		CONCEPT 2	PROJ FILE	PROJECT	
5(-7		CONCEPT D (ALSO ON DISK, SEE NUMBER 45)	PROJ FILE		
56-4		CONCEPT 4	PROJ FILE	PROJECT	

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\$(-\$		CONCEFT 5: SPACE FRAME	PROJ FILE	PROJECT	SPACE FRAME RECDIL ELEVATING MECHANISME
5;		COMPUTER RUNS FROM PROGRAM "RECOIL"	PROZ FILE	PROJECT	RECOIL PROGRAM AND
52		MEETINGS AND PLANE (BY DATE)	PROJ FILE	PROJECT	PLAN HEHC HEETINGS BUTGET
53		TRIF TO RIA 840909	PROJ FILE	FROVECT	MISE HONITZER (CO)T TRIFE 10 17 Space OF
54		COMPUTER RUNS FROM PROBRAM "INTEAL"	PRGS FILE	PROJECT	BALLISTICS PONEUTEP PALALYSIS
ee		BRAINSTORMING SESSION OF 840910-14 WITH CEL	PROD FILE	PROJECT	BRAINSTORM CONCEPTS RECOIL
58	E0 1100 (FMC) E0 1005 (FMC) E0 807 (FMC)		PROCETLE	PROJECT	RECOIL COMPUTER ANALYSIS
5- (MIL-HDBH-23A (BOD)	STRUCTURAL SANDWICH COMPOSITES	PROJ FILE	PROJECT	COMPOSITES
53 53		MARKETING INTELLIGENCE REPORTS	PROC FILE	FROJECT	INTELLIBENCE
5¢		RHEINMETAL HANDROD: 'BUUE'	RATHE	CMC.	WEAPON DESIGN
6(KF-6-5097 (ARMY)	MIRE TRAINING FILM	FORT #ECC:	£p≡.	MISE TRAINING FILM
é!	E156-2438 (G1DEP	ANALYSIS OF A COMPRESSIBLE FLUID SOFT RECOIL (CFSR) FOR 155 MM HOWITZEF	ekti elli	PROVECT	RECOIL-SOFT
62+		LIGHTWEIGHT HOWITZER - COMPONENT CONCEPTS	PROX FILE	PROJECT	CONCEFTS
624		APU	PROJ FILE	PPD2ECT	APL
٤٦		MISC TRIP REPORTS	PROJ FILE	CBESSES.	TR152
64			*** ****		
65 +	(APMY	SCIDING ADJUSTABLE CHAMBER BREECH (DIF SCALE	PROJECT FOR	CFC/III	F), #;
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£54		FILE			
ESE		MARRATIVE TEXT			
6 2	TM 9-1025-211-10) (ARM)	. MIRE OPERATOR S MANUAL, CREW, DCT TO		PPCCECT	MIRE MAKUAL

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 -	TM 9-37(5)	(\$RPV)	PRINCIPLES OF ARTILLERY WEAFONS	PROJ FILE	PROJECT	HOWITIEPS RECOIL ARTILLERY
3 (ESPANE	(FMC)	ENHANCED SELF-PROPELLED ARTILLERY WEAPON SYSTEM			• ••••••
,84			VOL 1 - TECHNICAL REPORT APPENDICES EXCEPT D, SECTIONS AND F	Schemmal	+mc	RECOIL MUIILE BRANES BALLISTICS BREECHES CANNONS
5 <u>:</u>			VOL 1 - TECH REPORT ON 3 (EXC 3.2), 5-6	*	V	• ••••••••••••
¢	FE-7(-7442	•••••	EXPERIMENTAL INVESTIGATION OF MUZZLE BRAKES FOR XM198 HOWITIER. JAN 70. SALSBURY, SMAFFER. ARTILLERY SYSTEMS LAE. ROCK ISLAND, IL.	PROC FILE	PROJECT	METTLE PRAMES
	0: 00:	(N43A	ELECTRONIC CONTROL FOR AN ELECTROPYCRAULIC ACTIVE CONTROL LANGING BEAR FOR THE 5-4	PROJ FILE	FROUEDT	VALIVES SERVO CONTROL
7		••••	CEL SUPPORT OF PROJECT	1	V 1	CEL WOLK
12		(FMC)	PROCRAM TO CAUCULATE RECOLD DEFICE AREA FOR SCOTT LONGLIFE	MAIN ×6771		RECOIL
13	TM9-1075-7	Air (Air	PRETS AIN STEEMS OF SUCCES		i conce	MISE MAILUE

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MISS ENVELOPE	
M198 HOWITIER	
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#198 MANUAL	(19, 13)
M198 PROCUREMENT SEEC	2° × ×
MIRE RECOIL	
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CLAMBTIC EXTREMS

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SPECIFICATIONS

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PUBLICATIONS

MIII BIDED 662

AR 70-38

STANAG 2831

SPANAG STABLE

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Shock Tests, H.I. (High Impact), Ship Machinery, Equipment And Systems

Mechanical Vibrations Of Shipboard Equipment
Climatic Extremes For Military Equipment
Reliability Testing For Engineering Development,
Qualification And Production
Glossary Of Environmental Terms
Test Requirements For Space Vehicles
Environmental Criteria And Guidelines For Air-Launched
Weapons
Calibration System Requirements

Research, Development, Test And Evaluation Of Materiel For Extreme Climatic Conditions Climatic Environmental Conditions Affecting The Design

Of Materiel For Use By NATO Forces Operating In A Ground Role

Environmental Test Methods For Aircraft Equipment And Associated Ground Equipment

(Copies of specifications, standards, handbooks, drawings, and publications required by manufacturers in connection with specific acquisition functions should be obtained from the contracting activity or as directed by the contracting officer.)

2.2 Order of precedence. In the event of a conflict between the text of this standard and the references cited herein, the text of this standard shall take precedence.

3. DEFINITIONS

- 3.1 The following definitions shall apply:
- a. Accelerated test. A test designed to shorten the test time by increasing the frequency or duration of environmental stresses that would be expected to occur during field use.
- b. Aggravated test. A test in which one or more conditions are set at a more stressful level than the test item will encounter in the field in order to reduce test time, reduce sample sizes, or assure a margin of safety.
- c. Ambient environment. The conditions (e.g., temperature and humidity) characterizing the air or other medium that surrounds material.
 - d. Environmental conditions. (see Forcing function).

DESCRIPTION: WHO-WHAT LIST

STATUE: A list of "who does what" and estimates of hours required for Phase II completion was developed with input from the project participants.

The information is current and complete as of 12 March 1987 with σ inor changes.

AUTHOFS: FMC NOD team members.

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Part Anderson
Critical Path sanagement of design.
York
Nerotta
Unassigned items.
Front Silde sanifold to emplacement (swive) up) skid plate
Inter-tess support.
Equilibration mounting tube (work with Gene)
Bob Mortenson
Bob Schaidt
Charles Ortloff Start system model update (coordinate with Larry) Finish model runs (coordinate with Larry) Organize Appendix of raw data (with Larry) Provide Text cover for Appendix (coordinate with Larry)
Charlie Dalside Audit recoil and hydraulic system analysis
Dan MacGuire Structural analysis of
Speedshift salve! (work with Kent).
Load Way Mork with Rom)
Fire control abunts and links (work with Scott).
TOTAL THE TRANSPORT OF THE PROPERTY OF THE PROPERTY BEACH TOTAL TO

Critical Path aanagement of Ph 3 hardware Long Lead Sem List. Critical path item East charting. Taillights and wiring.	A 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	0430 238
Generate cash flow curves for critical path items (committed 5 and sunk 5). Rearod critical path hardware drawings. Add drawing status to PN-log. Band comming status to PN-log.	A E E	0124
Muzzle brake. Coordination of design practices and checking of all drawings. Coeplete Assyr practs - general. May 01. 28		0128
Spare parts. Spare parts. Standard Document Book. Finish off walking beams. Inter-team support.		3014
Dave Flippo Ph 3 costa for atructural testing (where?)	""	7٠16
Platform under the action of t	777 777 777 777	774
The state and years and revise dwgs (loads per Jett) Orawings to orthofs. Finish contribution to final stress analysis report. Spade. Spade. Inter-team support.		21 27 27160 7716 1716
Recoil cylinder dugs (revise). Recoil cylinder dugs (revise). Rid-manifold mounting (layout, detail, revs). Simbal manifold, hoses, and hardline (layout, detail, revs). Tube bundles on cradic (layout, calcs, detail, revs). Compound actuator drawings (layout and revs). Finish hydr syst detailing (incl revs).		77
Midslide samifold to detailing. Integrate York and Marotta reve	9 F	27 2040 7740

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Drane Tollette Trail (with Lauren, Suellen). Swivel tierin to cradle. Structural analysis report. Plastics applications: trail pads, load tray quides, and etc per Mark.
Ellen Brady
Errol Quic Test plan Contractor developed specs
Fred Appleton Assy at BCF or MP7. Charge number for weld plece parts.
Checking of drawings. Checking Deserved to the control of the con
Trails swivel tre-in to platform. Plastic parts. Inter-teem support.
Front slide eanifold and all attachments to it, ready to detail Front slide eanifold and all attachments to it, ready to detail Revisions Lifting eyes for FSM (with Scott) Integrate York and Marotta revs Inter-team support.
Jeff Treland Complete hydraulic system design and analymis. Review Marotta dags and work out rest of problems. Review Mork dags and work out rest of problems. Work with Charlie on audit. Finish hydraulic system detailing (with Bart, Davew, Joe, Scott) and stay abreest of chh ng.
Jim Riem (II wkm w 8 hrs/mk = 90)
Jim Wallace Nake/Buy. Source selection of critical composite parts. Ph 3 costs.
Joe Flabbine Optimize stress analysis of collars. Titanius bulkhead joint analysis.

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Cannon GFE to Benet/ARDEC (with Joe).			٠	•	•
Finish autoprimer mount (with Tim)				č	, c
Finish balance of cannon assembly			•	=	- L
AZ/GE aliquaent tool pper					
finish hydraulic system detailing (with Part, DaveW, Jeff, Scott) and stay abreast of this ng Assist in detailing and charting					
Inter-teas support.					
final annii hration/alauston analuse					
Prepare Appendix for inclusion in DAR (work with Scott)					
Prepare text for DAR (work with Scott)					
J.R. Tousley					
judy Lobeck Check MSM for port intersections and mall thicknesses.					
Detail MSM.					
Check FSM for port intersections and wall thicknesses					
ent Williams Design of cradle and all interfaces					
Finalize all bearing joint designs.					
Grant Art of the Control of the Cont					
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	inalysis report. A-2 review, or Scott) E load pos in E load pos in are in place th Bart, Daeek, Larry) Control valves Cott) Cott

Terry Bottmaldt

Preliminary Mazard Analysis Report....... Complete analysis of resaing system.

Check porting logic, structurally analyze, and minimize weight of MSM.

Check porting logic, structurally analyze, and minimize weight of FSM.

Prepare Appendix sections for DAR (work with Scott). Prepare Appendix sections for Stress Analysis Report (nork with Scott).

Prepare text for DAR (work with Scott).

Prepare text for Stress Analysis Report (work with Scott). Auto primer (mith Joe)..... Inter-team support..... Inter-teem support. Platform Cradle. Comparisons binn is and sti for ab (Bob Mortenson).... Inter-team support..... How auch energy can be recovered by firing a 17 or 185 mithout projo Will outer band self-center? Should seell Is parts be heat treated to 150 ks; (bulinead, wb 5:)? Unanswered questions (that may never be answered) Employ directional properties in composites? Design trail with one lattice knocked out? Weight reduction opprotunity search Will a projo carrier be needed? basic issue items and carrier preparation for cdr Spec control dwg for APU Equilibrator link ass'y Trail howe to platform Taillights and wiring Composites spreadsheet cradle torque anchors Ph 3 engrng resp Trail hardware to FSM DA/DC of composites Test of composites MSM mounting pads trail foot ass'y Update FBD's Add fatigue fom Hillstrom Mug Logo? Toe Rudolf Unassigned for Mastk

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Provide text supporting non-structural analysis.

Provide Ph 3 cost estimate and delivery schedule. Revise prints per markups.
submit complete print parkage.
Submit Appendix of raw data from structural analysis.
Submit Appendix of raw data from non-structural analysis. Provide approval prints for all assemblies.....

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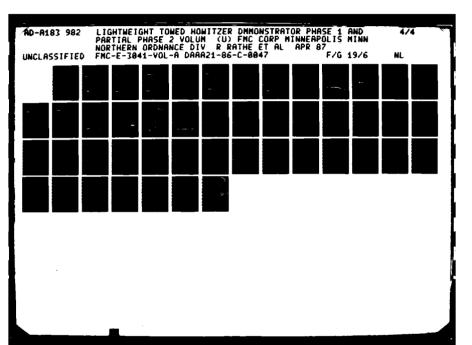
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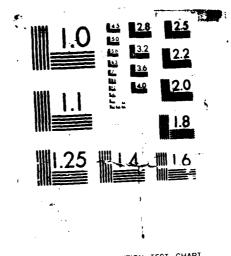
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NAME DAVE WARNICK (WITH MARK) EATE 87-2-26

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NAME DAVE WARWICK
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NAME DAVE WARVICK

DATE 87- 2-26

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ANALYST'S HR ESTIMATE

ESTIMATE OF ADD'L ANALYSIS NEEDS:

NAME DAVE WARUNCK (WITH BART, JEFF, JOE, SCOTT)
DATE 87-2-26
ASSEMBLY MINBER

ABSENDEY NAME FINISH HYD. SYS DETAILING

NAME DIANE TOLLETTE (WIT LEINGRIN, SUELLEN, KENT)

ASSENBLY NUMPER . 5710- 595

DATE 87- 2-26

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5101 Mg 42 5 48/11/5 175 911 1117 WILL 12018 15 707 7L1 A7T3E:9W 12018 8/2058 320513-3091 Any questions on this requirements, feel free to contact me on Ext. 6332. required in Phase II. Mote: This does not include manufacturing support hours on weld parameters and tool development (cutting techniques) nor any effort associated with Phase III other than manufacturing cost The effort to complete this task is 984 hours 2. Effort decisions, Phase III manufacturing costs and complete approximately 70% of required weld sketches. Complete design review, make required vendor selections, make/buy FLED 1. Description end of Phase II on the LIMD Program: SABHEL ADVANCE MANUFACTURING HOURS cc D. F. Hartman J. Melquist "" J. Wallace JZW * R. Rathe? 7861 Kraunda 387 Hear Baiheseini 6 ع (51.4 ، و11) FINISH TASK REQUIRED TO F ANALYST'S HE A-1750, COMPLETION DATE (WK/MO) TOTAL HOURS TO FINISH TASK HOURS AS OF NUMBER 5710 - 810 - 835 Pw J Spec S.E.O. ADD'L ANALYSIS NEEDS: tel. Ü ELSE 皇 DMGS ALL FAST CALC/ASSIGNING OF ALL Q.1.6 (SHOF) DETAIL YOUR ANALYSIS/CALC RESIGNING OF (SHORT) F Ь PARTS LIST TASKS EST. TASKS REQ'D TO FINISH ASSEMBLY SGF() ġ APAK A Ь 101 Pet, J. _6M/53M REVIE: CNS DETAILING ESTIMATE CHECK ING ASSEMELY

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Plate & Misc Grp = \$300

Assume \$ 3500 TOTAL \$ 3120

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FINISH ASSEMBLY AS OF MON. MARCH 2

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NAME JOHN GREEN (WITH SCOTT) DATE 87-2-26 ASSEMPLY-MUNEER

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NAME JUDY LOSECK (WITH DAVE WARVICE) 22-2-L8 ELVO ASSEMBLY MOTEST

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NAME KENT WILL IRMS

DATE 87-2-76

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NAME MARK RUMPSA (WITT UPER)

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NAME FOR PUDDOLF DATE 87-02.26 ASSEMPLY NUMPER ASSEMBLY NAME TASKS RED'D TO FINISH ASSEMBLY

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HENE VOLKMANN DATE 87-2-26 CONTRACT A LEASE OF ASSENDEY NAME: ANALYSIS OF RAMMING SYSTEM

HOURS REDUIRED TO FINISH TASK AS OF MON. MARCH 2 YOUR ANALYSIS/CALC'S ----- Fright TASKS REQ'D TO FINISH ASSEMBLY DN: TOREST 9-1055

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AGGELBEY NAME PREPARE APPENDIX SECTIONS FOR DAR & SCREEN ESTEMATE OF ADD'L ANALYSIS MEEDS: NAME VOLKMANN ASSESSED A MUNICIPAL PROPERTY. DATE 87-2-26 ASSENDENT NAME CLIGCY FORTING LOGIC, STRUCTURALLY ANALYZE, AND MINIMIZE WEIGHT OF MID-WANIFOLD & FIFT MANIFOLD. HOURS REGUIRED TO FINISH TASK AS OF MON. MARCH 2 EST, COMPLETION DATE (WITHO) 87-5-29 150 TOTAL HOURS TO FINISH TASK COLD ANGERGRENCE DE TILL BEET LTS ESTIMATE DE ADD'L AMALYSTS MESDE: SET NO. DE DETAIL DW39 CHECKING AND OR SOMETHE YOUR ANALYSIS/CALC'S -TETATA-PHB 30 311,140,333 NAME VOLKIMALIAL CATE 87- 2-26 cardor TASKS REG'D TO FINISH ASSEMBLY ASSEMBLY ALTERED 1 OTHERD

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